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ADVANCED COMPOSITE VERTICAL STABILIZER FOR DC-10 TRANSPORT AIRCRAFT

CONTRACT NAS1-14869

SEVENTH QUARTERLY TECHNICAL PROGRESS REPORT 25 SEPTEMBER 1978 THROUGH 31 DECEMBER 1978

DOUGLAS AIRCRAFT COMPANY

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PREPARED FOR LANGLEY RESEARCH CENTER CONTRACT NAS1-14869

DRL Item Number 005

SEVENTH QUARTERLY TECHNICAL PROGRESS REPORT 25 September through 31 December 1978

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FOREWORD

This report was prepared by the Douglas Aircraft Company, McDonnell Douglas Corporation, Long Beach, California, under Contract NAS1-14869. It is the seventh quarterly technical progress report covering work performed between 25 September 1978 and 31 December 1978. The program is sponsored by the National Aeronautics and Space Administration, Langley Research Center (NASA-LRC). Mr. Marvin B. Dow is the Project Manager for NASA-LRC.

The following Douglas personnel were the principal contributors to the program during the reporting period: G. M. Lehman, Project Manager; C. O. Stephens, Engineering Supervisor; A. V. Hawley, Structural Design; J. O. Sutton, Stress and Loads Analysis; P. W. Scott, Weight Analysis; M. M. Platte, Cost Analysis; H. M. Toellner, Materials and Producibility Engineering; B. Lyon and M. Nagaoka, Manufacturing Engineering; R. B. Anderson, Engineering Test Supervisor; R. G. Wolfe, Structural Testing, and G. J. Cassell, Lightning Panel Testing.

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SUMMARY

Structural design, tooling, fabrication, and test activities are reported for a program to develop an advanced composite vertical stabilizer (CVS) for the DC-10 Commercial Transport Aircraft. Structural design details are described and the status of structural and weight analyses are reported. A structural weight reduction of 21.7 percent is currently predicted. Test results are discussed for sine-wave stiffened shear webs containing cutouts representative of the CVS spar webs and for lightning current transfer and restrike tests on a panel representative of the CVS skins. Results are presented for mechanical property and fracture mechanics tests to substantiate design allowable stresses. Current status is reported for tooling, fabrication, and quality assurance activities on rudder fittings, skin panel, and spar verification test components. Recurring manufacturing cost projections for the CVS structural configuration indicate that a cost cross-over point with the conventional metal stabilizer will be achieved after production of 32 CVS units. Selected engineering drawings and supplemental mechanical properties data are included in appendices.

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TABLE OF CONTENTS

Section	<u>Page</u>
1	INTRODUCTION 1
2	DETAIL DESIGN
	DESIGN DEVELOPMENT- 5 Skin Panels- 5 Spar Assemblies 8 Base Rib - 13 Rib 295 - 13 Actuator Ribs 13 Tie-Rod Ribs 17 Plain Hinge Ribs 17 Substructure Assembly 21 Box Assembly 21 Access Doors 21 Trailing Edge Panels 21 System Installations 21 Drawing Release Status 25 STRUCTURAL ANALYSIS 27 WEIGHT STATUS 30
3	CONCEPT DEVELOPMENT COMPONENTS
	SPAR-WEB COMPONENT
4	JOINT DEVELOPMENT COMPONENTS
5	MECHANICAL PROPERTY TESTING 59
	LAMINATE PROPERTY TESTS
6	DESIGN VERIFICATION TEST COMPONENTS 87
	CONCEPT VERIFICATION PANELS
7	TOOL DESIGN
	SKIN FABRICATION TOOLING

TABLE OF CONTENTS (Continued)

Section	<u> </u>	age
8	COST ANALYSIS	01
9	QUALITY ASSURANCE	05
10	REFERENCES	09
APPENDIX A	ENGINEERING DRAWINGS	111
APPENDIX B	MECHANICAL PROPERTIES TEST DATA	145

LIST OF FIGURES

Figure		Page
1	Summary Schedule	3
2	DC-10 Composite Vertical Stabilizer Structural Components and Drawing Numbers	6
3	Skin Panel Assembly	7
4	Spar-Root Fitting Splice-Joint Details	9
5	Typical Spar Assembly	10
6	Typical Hole Reinforcement in Sine-Wave Web	12
7	Base Rib Assembly	14
8	Rib Station 295 Composite Moldings	15
9	Station 314 Actuator Rib Assembly	16
10	Tie-Rod Rib Fittings Configuration	18
11	Hinge Rib Fittings Configuration	19
12	Upper Hinge Support Assembly	20
13	Substructure Assembly	22
14	Typical Access Door Assembly	23
15	Typical Trailing Edge Panel Assembly	. 24
16	NASTRAN Model for the Composite Vertical Stabilizer	. 28
17	Fatigue Test Load Exceedence Spectrum for Z5943452 and Z5943454 Components	. 29
18	Composite Vertical Stabilizer Weight Trend	. 35
19	Sine-Wave Shear Web Component in Test Fixture	. 38
20	Flat Area in Z5943434-501 Sine-Wave Spar Web	. 39

LIST OF FIGURES (Continued)

Figure	<u>Page</u>
21	Unflanged Access Opening in Sine-Wave Shear Web Component 40
22	Failure of Shear Web at Unflanged Access Opening • • • • • • 42
23	Sketch of Lightning Panel Showing Test Points • • • • • • • 43
24	Test Setup - Simulated Lightning Current Transfer Test · · · · 44
25	Typical Test Waveforms - Current Transfer Tests 46
26	Test Setup - Simulated Lightning Restrike Test to Composite Panel (116 KA Peak)
27	Test Waveform - Simulated Lightning Restrike Test to Center of Panel
28	Composite Panel After 116 KA Simulated Lightning Restrike Test
29	Composite Panel After 116 KA Simulated Lightning Restrike Test (Close-up View)
30	Z5943453-1 Actuator Hinge Rib Component - View Looking Forward at Simulated Rear Spar
31	Z5943453-1 Actuator Hinge Rib Component - View Showing Internal Fitting and Simulated Sine-Wave Rib Web 57
32	Z5943453-501 Tie-Rod Rudder Fitting Component During Setup for Final Assembly
33	Fatigue Characteristics of T300/5208 Graphite/Epoxy Laminates - R = -1.0 60
34	Fatigue Characteristics of T300/5208 Graphite/Epoxy Laminates - R = 0.05 61
35	Fatigue Characteristics of T300/5208 Graphite/Epoxy Laminates - R = -1.0, Ambient Test Temperature 62
36	Fatigue Characteristics of T300/5208 Graphite/Epoxy Laminates - R = -1.0, Sandwich Beam Specimens 63
37	Test Setup for Damage and Debond Test 66
38	Close-up View of Plates Used to Prevent Buckling of Damage and Debond Specimens 67

LIST OF FIGURES (Continued)

Figure		Page
39	Fatigue Test Results for Laminate Tension Specimens with Debonds	68
40	X-Ray of Debond Specimens Showing Fatigue Damage	69
41	X-Ray of Debond Specimens After Fatigue Test Showing Static Failure	70
42	Fatigue Test Results for Damaged Laminate Tension Specimens	71
43	Trial Impact Energy Tests	73
44	Close-up View Showing Fatigue Damage in Impact-Damaged Specimen	74
45	X-Ray of Impact Damaged Specimen Showing Fatigue Damage	75
46	Typical Failure Modes of Damage and Debond Specimens	76
47	Typical C-scans Showing Fatigue Damage Sustained by Damage and Debond Specimens	77
48	C-scans of Damage Specimens and Moisture Control Coupons	78
49	Fatigue Characteristics of T300/5208 Graphite-Epoxy Laminates - R = -1.0, Ambient Test Temperature	79
50	Z3943443 Center Slit Panel Specimen in Test Machine	82
51	Failure of Z3943443 Center Slit Damage Specimen	83
52	Failure of Z5943428-501 Damaged Shear Panel	86
53	Z5943445 Cover Panel Combined Load Test Specimen	88
54	Final Assembly of Z5943445 Combined Load Test Specimen	89
55	Z5943446 Spar Component Laminating Mold	91
56	Z5943452-1 and -501 Specimens	91
57	Z5943452-1 Spar Web Splice	92
58	Z5943452-501 Spar Skin-Flange Splice	92

LIST OF FIGURES (Continued)

igure		Page
59	Average Moisture Content Versus Laminate Thickness for Various Times of Exposure to 170°F and 100 Percent Relative Humidity • • • • • • • • • • • • • • • • • • •	94
60	Tooling Concept for Composite Vertical Stabilizer Skin Panels • • • • • • • • • • • • • • • • • • •	96
61	Tooling Concept for Composite Vertical Stabilizer Spars • • • •	97
62	Tooling Concept for Composite Vertical Stabilizer Base-Rib	99
63	Projected Cost Cross-Over Point for DC-10 Composite Vertical Stabilizers · · · · · · · · · · · · · · · · · · ·	102
A1	Drawing AMC7840 - Skin Panel Assembly	113
A2	Drawing AMC7844 - Substructure Assembly	117
А3	Drawing AMC7847 - Forward Center Spar Assembly	118
A4	Drawing AMC7849 - Lower Rear Spar Assembly	128
A5	Drawing AMC7853 - Base Rib Assembly	132
A6	Drawing AMC7855 - ZFR 314.000 Rib Installation	138
A7	Drawing AMC7859 - ZFR 350.319 Rib Installation	142

LIST OF TABLES

<u>Table</u>		Page
1	Summary of Critical Margins-of-Safety for the Composite Vertical Stabilizer	31
2	Preliminary Weight Comparisons - Composite Vertical Stabilizer	32
3	Weight Change Summary - Composite Vertical Stabilizer	33
4	Weight Distribution by Material - Composite Vertical Stabilizer	34
5	Panel Resistance Measurements - Current Transfer Tests	47
6	Panel Resistance Measurements - Lightning Restrike Tests	50
7	Fatigue Test Results on Sandwich Skin Tension Panels with Transverse Center Slit	81
8	Damaged Shear Panel Test Results	84
9	Recurring Labor Hours for DC-10 Composite Vertical Stabilizer	103
10	Projected T ₁ Labor Hours for DC-10 Composite Vertical Stabilizer	104
11	Prepreg Quality Control Receiving Inspection Results	107
B-1	Sandwich Beam Fatigue Test Results	146
B-2	Fatigue Test Results for Debonded Laminate Tension Specimens	148
B-3	Fatigue Test Results for Damaged Laminate Tension Specimens	150
B-4	Strain-Gage Measurements for Z3943442 Damage and Debond Specimens	151

SECTION 1 INTRODUCTION

The overall objective of this program is to accelerate the use of advanced composite structures by developing technology and processes for early progressive introduction of composite structures into production commercial transport aircraft. Key steps in accomplishing this objective are:

(1) to develop low-cost design and manufacturing approaches which will produce a cost competitive structure, and (2) to initiate commercial airline service of a mid-sized composite primary structure, the DC-10 composite vertical stabilizer (CVS).

The Work Breakdown Structure (WBS) for the program is presently organized in eight major tasks as follows:

- 1. Preliminary Design
- 2. Detail Design
- 3. Manufacturing Process Development
- 4. Composite Structure Fabrication
- 5. Subassembly, Subsystem, and Other Fabrication
- 6. Assembly
- 7. Verification Testing
- 8. Contract Management and Plans Development

In Task 1, the Preliminary Design Synthesis culminated in selection of a four-spar, multi-rib structural configuration similar in geometry to the existing baseline metal stabilizer for structural interchangeability. The composite skin panels between spars and ribs will be stiffened by honeycomb sandwich construction for minimum weight and cost purposes, and the skin panels will be mechanically attached to the spar-rib substructure to enhance detail part fabrication, assembly, inspection, and maintenance access. The selected design concepts are presently being verified through testing of the concept and joint development component.

In Tasks 2 and 3, currently active, the structural detailed arrangements, tooling, and manufacturing processes are being developed through engineering and tooling design for the full-size CVS and through experimental development and testing of structural components representative of critical design features. The final development component will be a box-beam approximately eight feet long representing the lower portion of the CVS. The ribs, spar segments, and skin panels of this component will be fabricated and assembled using the full-scale tools for the CVS and will serve as a tooling and processing verification component as well as a structural test component.

Following completion of the box-beam tests, eight stabilizer units will be constructed in a serial production mode in Tasks 4 through 6. The first two of these units will be used in Task 7, Verification Tests, for the ground-based static and repeated load tests. The third unit will be flight-tested as part of the Federal Aviation Agency (FAA) certification testing and, together with the remaining five units, will be introduced into commercial airline service after receipt of FAA certification. Task 8 includes program management functions and the formulation of the plans necessary for development, certification by the FAA, and in-service inspection and maintenance of the CVS.

This report describes work accomplished during the seventh quarterly period of the program. Work continued on structural design and analysis; weight and cost analysis; and tooling, fabrication, and test of the structural development components and specimens. Detail design of the composite structure was continued and tool design was initiated. Overall schedule status is summarized in Figure 1.

The activities during the quarterly period are described under the headings Detail Design, Concept Development Components, Joint Development Components, Mechanical Property Testing, Design Verification Test Components, Tool Design, Cost Analysis, and Quality Assurance. Engineering drawings of the composite skin panels and selected spar and rib assemblies are included in the appendices.

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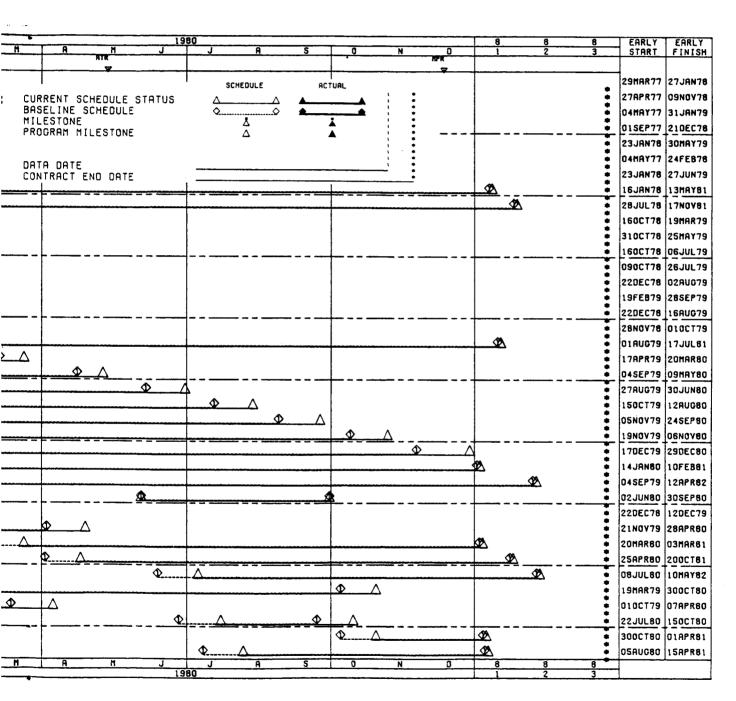
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FIGURE 1. SUMMARY SCHEDULE

Douglas Aircraft Company Contract NAS1-14869

The measurement values in this report are expressed in the International System of Units (SI) and also U.S. Customary Units in some cases. U.S. Customary Units were used for the principal measurements and calculations.

SECTION 2 DFTAIL DESIGN

Detail design of the CVS continued with engineering drawing preparation, concurrent structural analysis, and updating of the weight status report based on released engineering drawings.

DESIGN DEVELOPMENT

A CVS drawing list consisting of 89 detail, assembly, and installation drawings was completed and drawing numbers were assigned. Twenty-six of the required drawings are identical to or require only minor modifications to existing DC-10 drawings. Twenty-seven of the required drawings are new drawings of conventional metal details. The remaining 36 are new drawings defining the advanced composite structural elements.

The spar and rib locations of the existing metal stabilizer were retained in the CVS for interchangeability with the fixed-fin structure and the rudder system. The locations and drawing numbers of the major composite structural elements are shown in Figure 2. With the exception of the two uppermost ribs which were identified in the vertical stabilizer coordinate system (subscript V), the rib stations were identified in the forward rudder coordinate system (subscript FR). Engineering drawings of the skin panel assemblies, the substructure assembly, and typical rib and spar installations are included in Appendix A. Design features of the major composite structural elements are described in the balance of this section.

Skin Panels

Left and right-hand skin panels will each be made as a single molded honeycomb sandwich assembly. Spar and rib cap laminates will be included within the sandwich as shown in Figure 3. To maintain structural continuity at spar and rib cap intersections, the caps will be laid up with graphite unidirectional tape in a pseudo-isotropic pattern. The spaces between the caps will be filled by Nomex 4.0 pcf honeycomb core, 0.30 thick with a

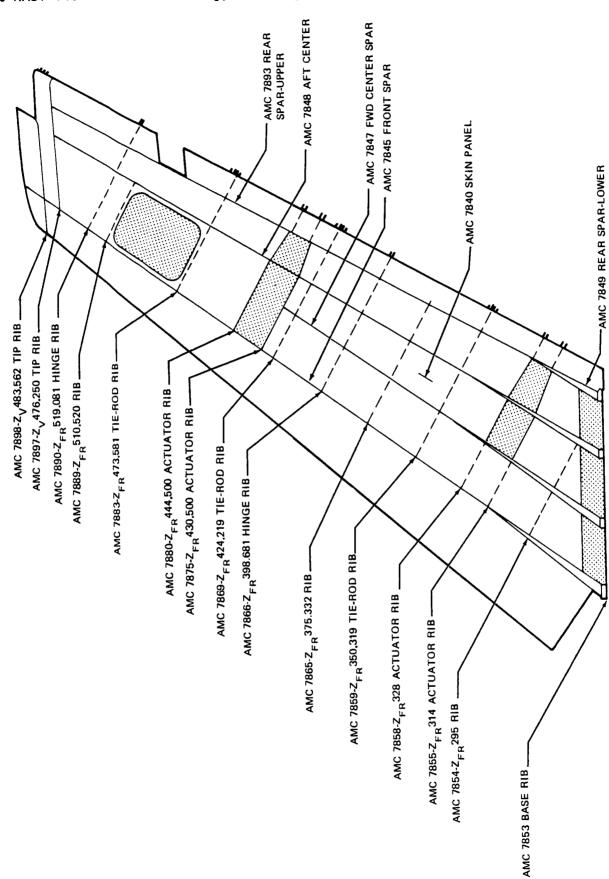


FIGURE 2. DC-10 COMPOSITE VERTICAL STABILIZER STRUCTURAL COMPONENTS AND DRAWING NUMBERS

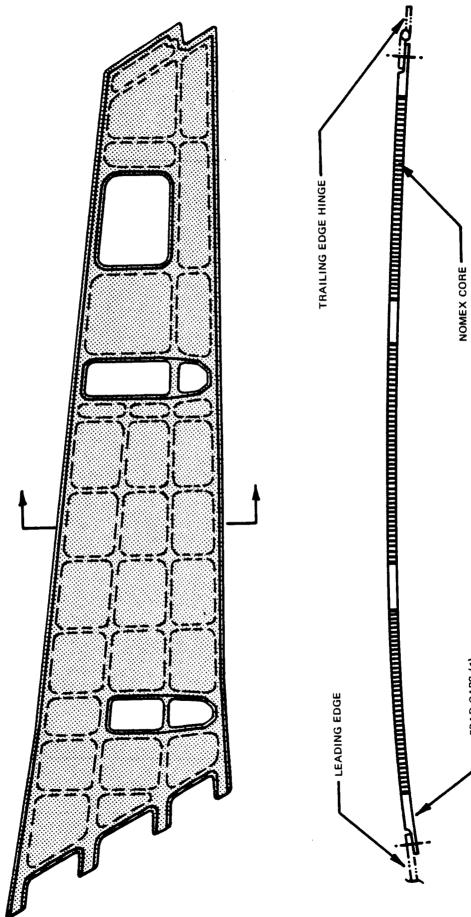


FIGURE 3. SKIN PANEL ASSEMBLY

- SPAR CAPS (4)

1/8 inch cell size. Syntactic foam will be used around the periphery of the core to stabilize the edges and to provide a transition region between the solid cap laminates and the thin sandwich facings. The gaps between the core edges and the surrounding laminates will be filled with foaming adhesive to ensure good shear connections at the interfaces.

The sandwich facing material will consist of graphite bi-weave cloth layers in a ±45° orientation with respect to the box axis. Near the root-end, some additional 0/90° layers will be added to help align the CVS box axis with the lower vertical stabilizer box axis. FM 300 K adhesive will be placed between the facings and the core, and the whole assembly will be cured in a single autoclave operation. Recesses for leading and trailing-edges and for access panels will be molded net, and the panel edges will be machined as the final fabrication operation.

Spar Assemblies

The four spar assemblies will have several design features in common. Each spar will have a pair of titanium fittings bonded within the graphite/epoxy at the root-end. The entire load of the stabilizer will be transferred into the lower vertical stabilizer through these fittings and their associated attach bolts. A detail view of the fitting installation is shown in Figure 4, and a more general view of the root-end of a typical spar in Figure 5.

Each titanium fitting will be basically a "tee" section where each of the three legs will be tapered to a narrow edge at the upper end, interfacing with the composite spar cap in a scarf joint. This joint will be co-cured and adhesively bonded during the autoclave cycle in which the entire spar assembly is cured. A secondary load path, capable of transferring design limit load, will be provided by means of mechanical fasteners. The cross-sectional area of the spar caps will be reduced rapidly away from the rootend as the load is transferred into the skin caps.

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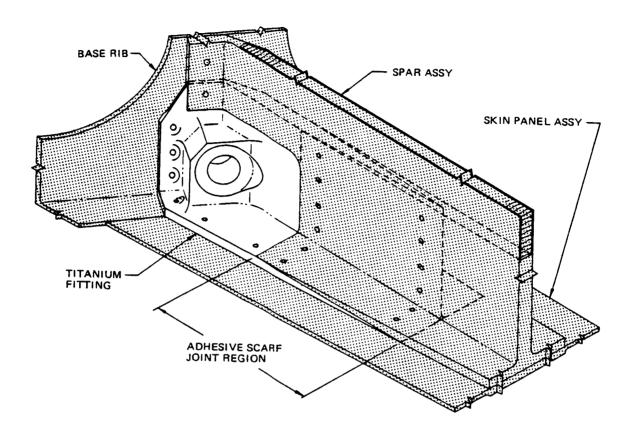
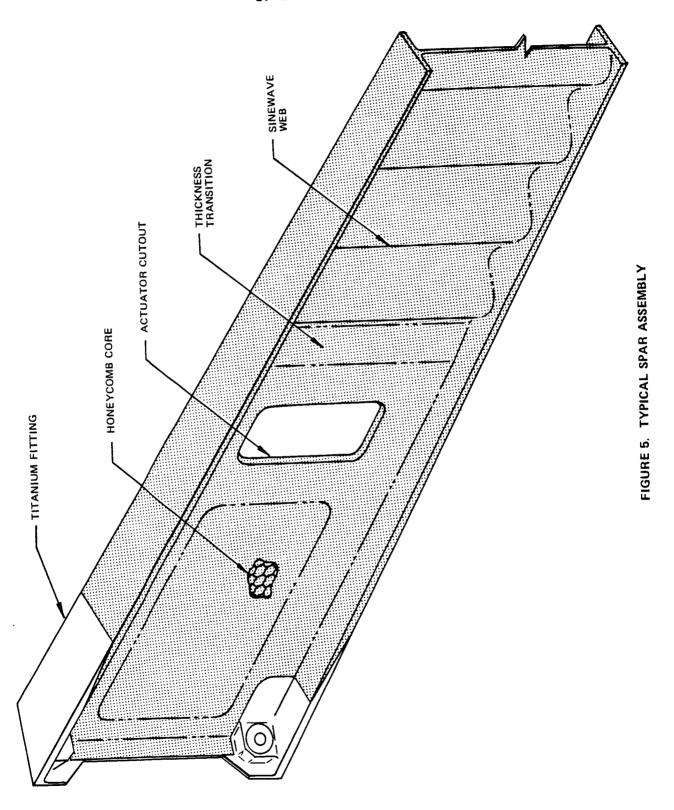


FIGURE 4. SPAR-ROOT FITTING SPLICE-JOINT DETAILS

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Localized honeycomb sandwich stiffening will be used near the root-end to stabilize the shear web against buckling. In most other regions of the substructure, sine-wave webs will be used to provide stabilization as shown in Figure 5.

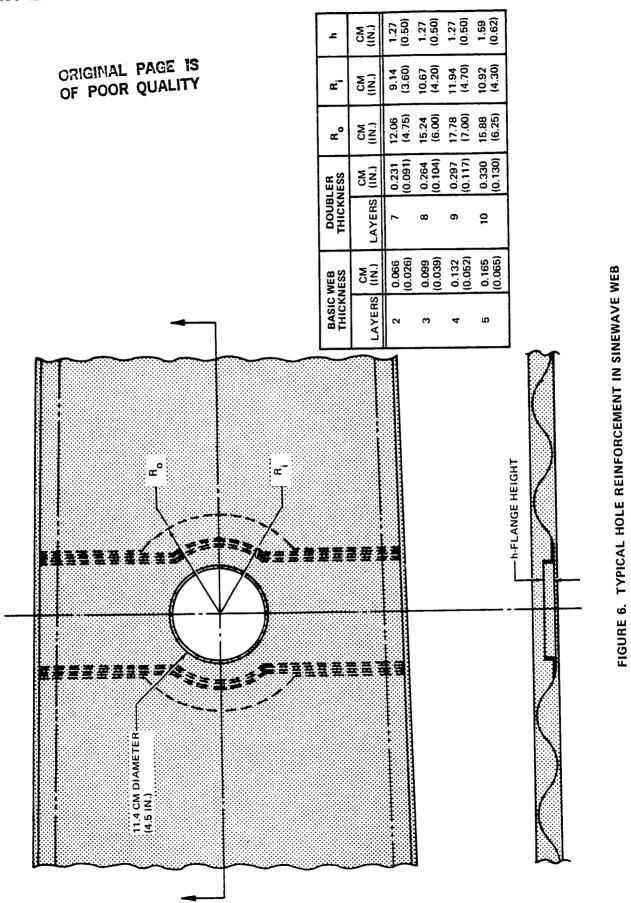
The sine-wave geometry is more properly described as intersecting circular arcs. A single wave geometry was selected for use throughout the entire substructure to reduce tool machining costs. This type of stiffening has been found to offer many advantages in cost and weight studies. Some of the weight advantage will be lost in reinforcing access holes and in mounting brackets and clips for rudder attachments and systems installations.

Plain holes in sine-wave webs were found to be inefficient because of the out-of-plane forces around the periphery of the hole. During the test program, a reinforced hole configuration was successfully developed in which a 0.50 inch wide flange was incorporated at the edge of the hole. The required doubler material around the holes was determined from test and analysis. The required local web thickness enabled the panel to remain flat in the region of the hole as shown in Figure 6.

In the lower segment of the front spar, the number of reinforced holes became so extensive that plain thick laminates with unreinforced holes were used with some cost saving and little weight penalty. Each of the three longer spars was divided into upper and lower segments to facilitate fabrication. The lower spar portion is shorter in each case but contains the thicker laminates and the titanium fittings. The upper portion will be primarily of sine-wave construction.

At the mounting stations for the rudder hydraulic actuators, cut-outs will be provided in the rear and aft center spar webs as shown in Figure 5.

These cutouts will be plain holes in the thick laminate material.



Where aluminum fittings attach to the graphite-epoxy structure, an interfacing layer of fiberglass will be cocured on the face of the laminate to alleviate galvanic corrosion effects. For smaller clips, the fiberglass will be attached by secondary bonding.

Provisions will be made for access into each bay of the substructure by means of 4.5 inch diameter access holes. This internal access will assist in initial fabrication and inspection and in subsequent inspection and repair.

Base Rib

The base rib will consist of two thick laminate segments joined at the rib center-line as shown in Figure 7. Since the adjacent rib in the lower vertical is capable of transmitting the re-distributing shear loads, the web of the base rib will be largely removed by three oval holes. The remaining material will provide load paths between the root attach bolts, and the skin and spar panels. The base rib edges will be flanged for attachment of the titanium spar fittings and the skin access panels. Projections at the aft end of the rib will support the hinged trailing-edge panels.

Rib 295

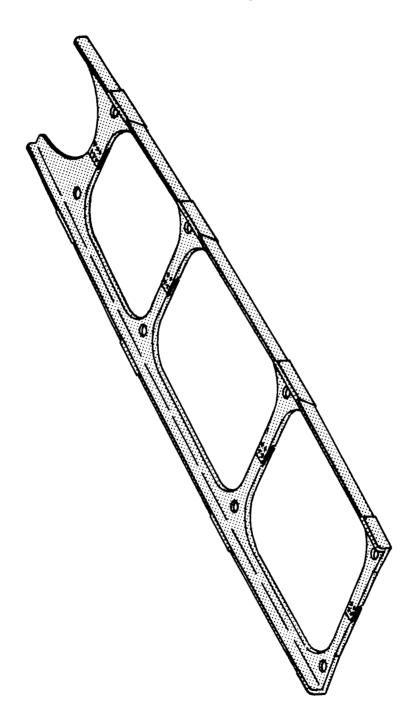
This rib is typical of the three ribs which do not incorporate rudder support fittings. The simple sine-wave webs without holes, Figure 8, are representative of many rib segments throughout the substructure. The skin edges of the webs will be reinforced and flanged to form "tee" section caps for attachment to the skin panels. Drainage holes will be provided in the valleys of the sine-waves to prevent standing water from accumulating.

Actuator Ribs

Actuator ribs are located at stations $Z_{FR} = 314.000$, 328.000, 430.500, and 444.500. The rib at station 314, illustrated in Figure 9, is typical of this group. The web construction will be similar to that of Rib 295 except that in this case, an access hole will be provided in the center segment. A cover-plate over this hole will prevent loose objects from falling into the bay below during hydraulic actuator maintenance.

FIGURE 7. BASE RIB ASSEMBLY

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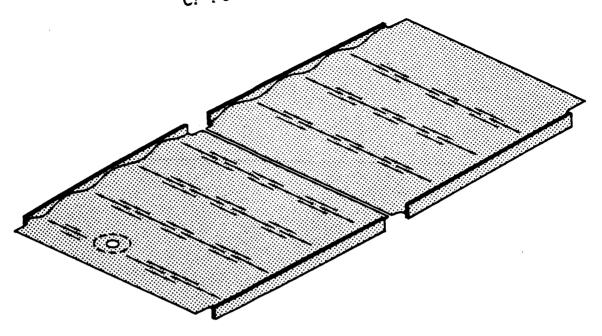
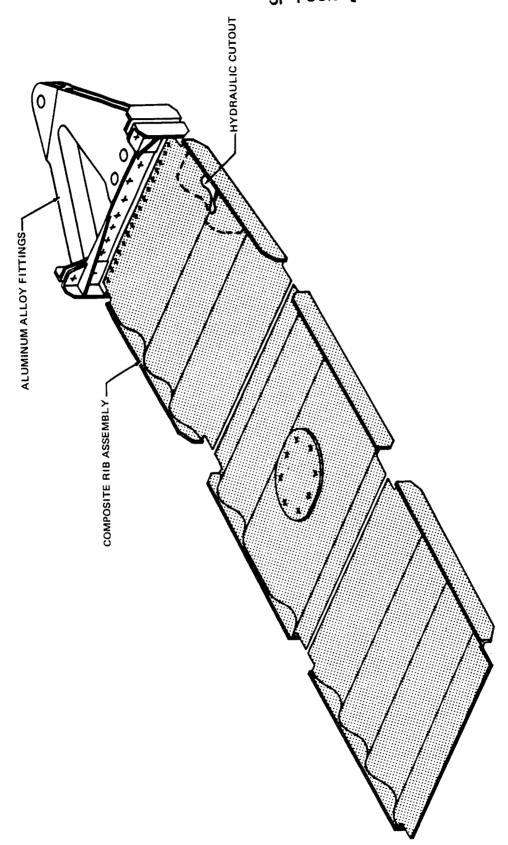


FIGURE 8. RIB STATION 295 COMPOSITE MOLDINGS

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The aluminum alloy actuator bracket will locate the rudder hinge point and the mounting provisions for the rudder hydraulic actuator trunnion. The actuator bracket differs only slightly from the bracket used on the metal stabilizer. Separate fittings, forward and aft of the rear spar, will transfer the rudder loads into the composite box structure. The shear webs at this fitting interface, and at the front spar, will be reinforced with doubler layers to accommodate loads from the rudder and from the control quadrant mounting on the front spar. This particular rib will also incorporate an oval cutout for hydraulic piping.

Tie-Rod Ribs

Tie-rod ribs are located at stations $Z_{FR}=350.319$, 424.219, 473.581, and in the tip region. These ribs differ from the actuator ribs in the manner in which the tie-rod brackets transfer loads into the composite box-structure. A typical fitting arrangement is shown in Figure 10. Fail-safe requirements make dual load paths necessary and hence two fittings run along the rib caps at each skin panel. Since it is not convenient to attach a sine-wave web to the face of one of these fittings, a thin honeycomb sandwich panel will be used in this rib segment.

At the upper tie-rod station, a separate structural sub-assembly will be added to the box-structure as shown in Figure 11. This complex region will provide fail-safe load paths for loads arising from the rudder balance weights. Seven existing metal fittings will be contained within this sub-assembly, including the brackets for two hinges and two tie-rod attachments.

Plain Hinge Ribs

The ribs located at station Z_{FR} = 398.681 and 519.081 support simple hinge brackets. Since the rudder loads are smaller at these ribs than at actuator or tie-rod ribs, the load transfer to the composite box structure will be accomplished by simple fittings mounted on the aft side of the rear spar as shown in Figure 12.

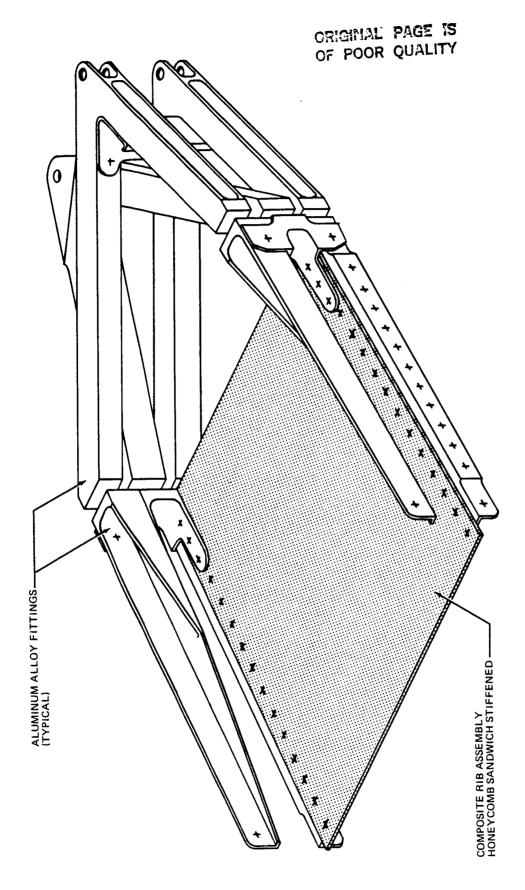
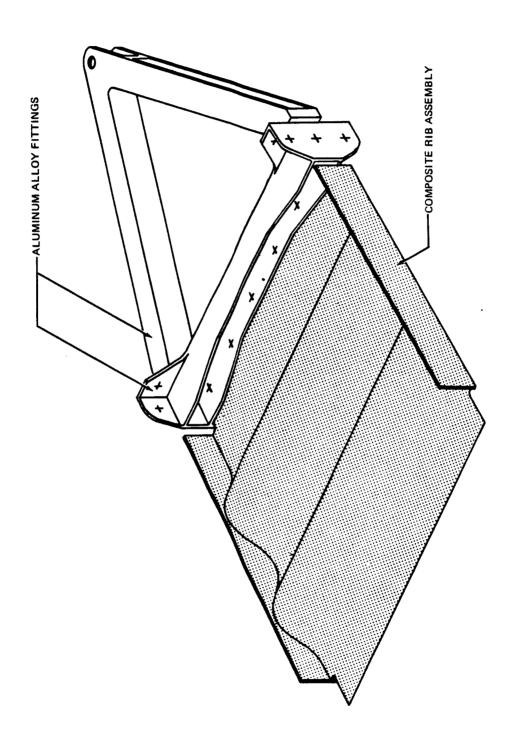


FIGURE 10. TIE-ROD RIB FITTINGS CONFIGURATION



19

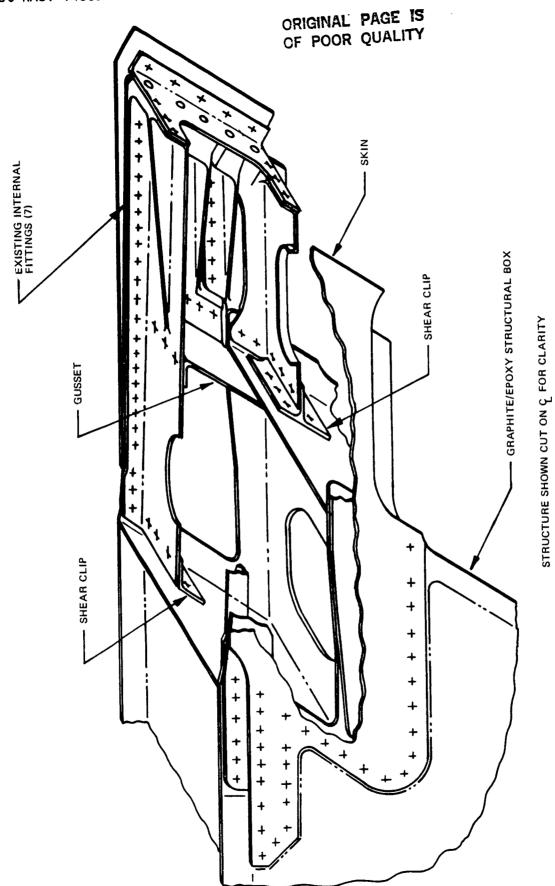


FIGURE 12. UPPER HINGE SUPPORT ASSEMBLY

Substructure Assembly

The spar and rib assemblies will be joined to form a substructure assembly as shown in Figure 13. The attachment at the intersecting corners will be effected by graphite-epoxy angles which will be co-cured and adhesively bonded in place.

Box Assembly

The skin panels will be attached to the substructure assembly with titanium bolts in accordance with normal DC-10 fastener usage policy. Nut-plates and channels will be used in all places where access does not permit the installation of nuts.

Access Doors

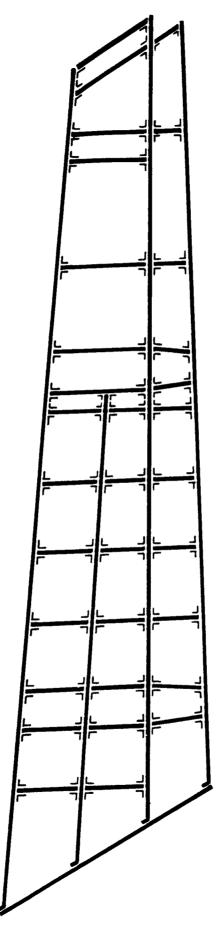
Access doors for the root-attach and actuator regions will be simple sandwich panels as shown in Figure 14. The basic construction will follow skin panel practice. The panel will be thick enough to sustain shear, compression, and lateral pressure loads without buckling. Solid laminate material will be used at the edges where the doors bolt to the skin panels.

Trailing Edge Panels

The design concept for the graphite-epoxy trailing-edge panels is shown in Figure 15. This type of composite panel has already been developed and several panels are presently in regular DC-10 airline service. The CVS panels will be hinged at the forward edges to permit opening for maintenance. The hinge-line was moved slightly aft in comparison with the metal stabilizer. When closed, the panels will be attached to supports mounted to the rudder hinge brackets or to the box skin panels.

Systems Installation

Mounting brackets and clips will be provided for installation of the hydraulic, control, electrical, and avionics systems. The avionics VOR/localizer antennas contained in the tip and door panels will be retained without change. To maintain satisfactory avionics performance, the outside surface of



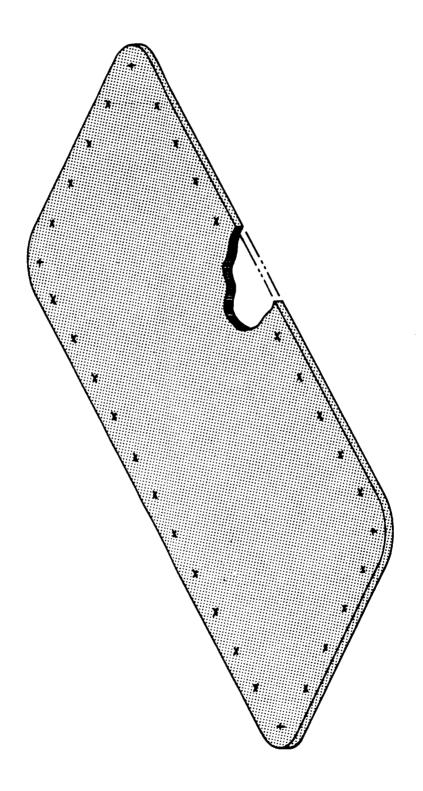
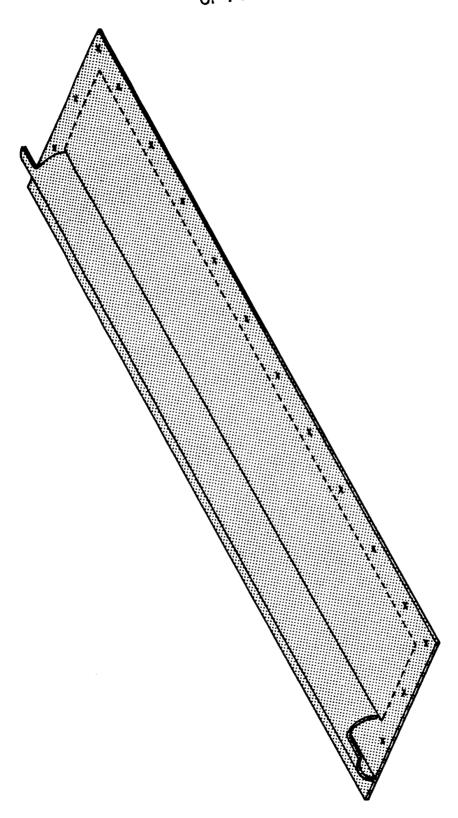


FIGURE 15. TYPICAL TRAILING-EDGE PANEL ASSEMBLY

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the graphite-epoxy skin panel and the inside surfaces of the antenna bay will be flame sprayed with a 5-mil aluminum coating. The exterior aluminum surface will also provide protection against the lightning travelling-stroke phenomena. Direct lightning paths will be provided across the tip, and down the metal leading-edge and rudder assemblies.

Special care will be taken to ensure electrical continuity across panel joints and at access doors. This continuity will be achieved at the access door by coating the inner rather than the outer surface of the door with the aluminum spray. Electrical continuity to the leading-edge will require local removal of the paint finish. The metal will then be specially treated to avoid galvanic corrosion.

Drawing Release Status

A total of 28 new drawings were released during the reporting period, and changes were made to four existing drawings to adapt them to the composite stabilizer configuration. Slightly more than 50 percent of the total required drawing effort has been completed. Released drawings are listed below.

```
Skin Panel Assembly
AMC 7840
           Substructure Assembly
AMC 7844
           Front Spar Assembly
AMC 7845
           Front Spar Attach Fitting
AMC 7846
           Forward Center Spar Assembly
AMC 7847
           Aft Center Spar Assembly
AMC 7848
           Lower Rear Spar Assembly
AMC 7849
           Hinge Support Fitting
AMC 7850
           Cant Rib Cap Fitting
AMC 7851
           Forward Center Spar Attach Fitting
AMC 7852
           Base Rib Installation
AMC 7853
            ZFR 295 Rib Installation
AMC 7854
            Hinge Bracket Assembly
AMC 7856
            Hinge Support Fitting
AMC 7857
```

```
Z<sub>FR</sub> 350 Rib Installation
AMC 7859
AMC 7862
           Rib Cap Fitting
AMC 7863 Rib Cap Fitting
           ZFR 375 Rib Installation
AMC 7865
           Z<sub>FR</sub> 424 Rib Installation
AMC 7869
           Hinge Support Fitting
AMC 7871
           Rib Cap Fitting
AMC 7872
            Rib Cap Fitting
AMC 7873
            Rear Spar Attach Fitting
AMC 7878
            Aft Center Spar Attach Fitting
AMC 7882
            Hinge Support Fitting
AMC 7892
            Upper Rear Spar Assembly
AMC 7893
            Laminated Graphite-Epoxy Angle
S00202
            Laminated Graphite-Epoxy Angle
S00203
```

The appropriate changes have been made to the following existing hinge and tie-rod bracket assemblies: AMC 7029, 7031, 7073 and 7074.

Drawing preparation is continuing on the remaining ribs and their associated fittings, on the access door assemblies, and on the trailing-edge installation.

STRUCTURAL ANALYSIS

The structural analysis effort has been devoted to completion of the NASTRAN internal loads analysis, derivation of test conditions for the Z5943454 box-beam verification test component, and strength analysis in conjunction with the engineering drawing release activities.

The NASTRAN model for the final internal loads analysis is illustrated in Figure 16. The rudder modules are complete and operational. The lower vertical stabilizer and aft fuselage module is complete and operational. This latter module will be also used to establish the support flexibilities for the Z5943454 box-beam verification test component. Physical and material properties for the upper stabilizer module are approximately 30 percent complete.

The fatigue loading spectrum for the Z5943454 box-beam component has been completed. The identical spectrum will be used on the Z5943452 spar root splice specimens. The spectrum is based on that used on the DC-10 aft-fuselage full-scale fatigue test with the loads modified to reflect the latest external and internal load distributions as developed for this program (Reference 1). The completed test spectrum load exceedance chart is shown in Figure 17. The gust exceedance theoretical line was plotted directly from the DC-10 aft fuselage test report, Reference 2, and the maneuver line was obtained from References 3 and 4.

The analysis method for flanged access holes was substantiated during the reporting period by successful testing of the Z5943434 sine-wave shear-web component (see Section 3 of this report). The analysis method was described in the prior quarterly Progress Report, Reference 5, Appendix B. Accordingly, standard access hole doubler arrangements were formulated for each sine-wave laminate patterns of interest. The doublers are capable of transmitting the maximum shear-flow in the most critical panel of each pattern, (see Figure 6).

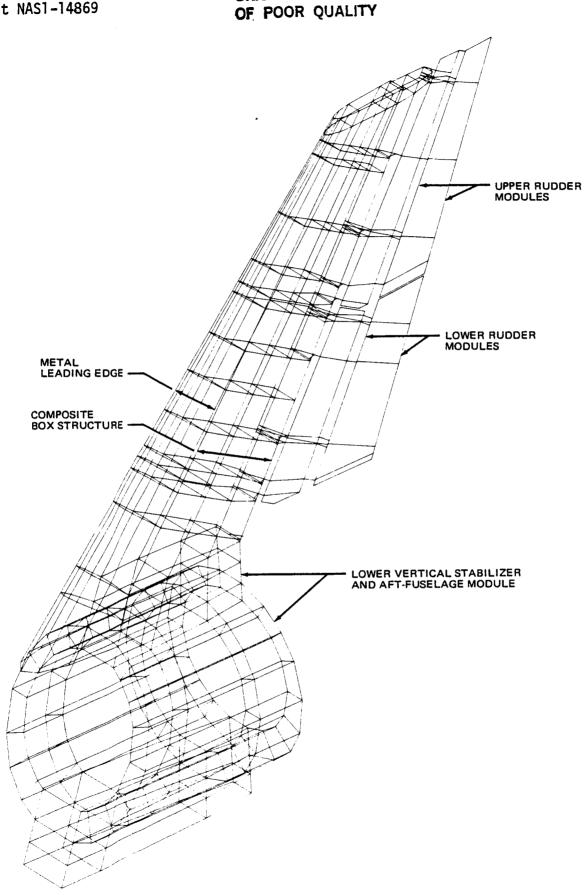


FIGURE 16. NASTRAN MODEL FOR THE COMPOSITE VERTICAL STABILIZER

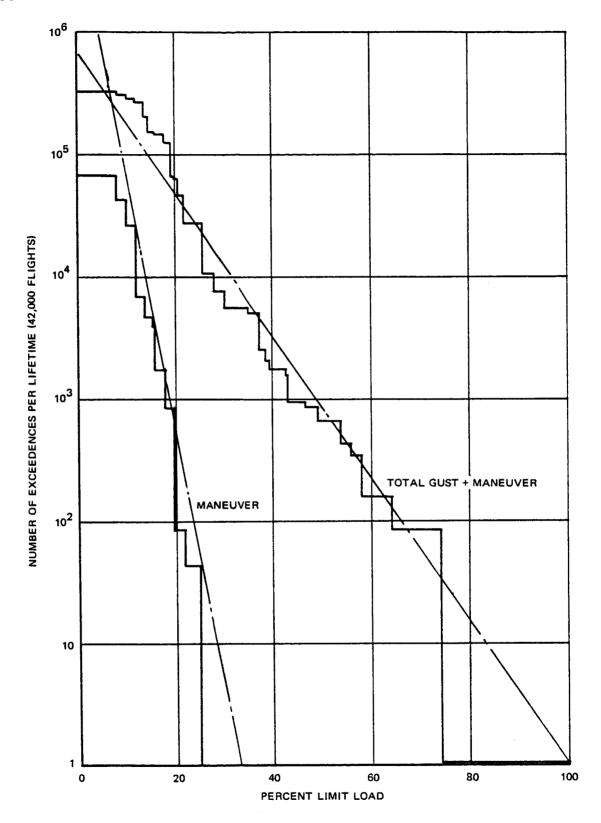


FIGURE 17. FATIGUE TEST LOAD EXCEEDENCE SPECTRUM FOR Z5943452 AND Z5943454 COMPONENTS

Analysis of designed components is proceeding concurrently with the engineering drawing release activity. A summary of analyses completed during the reporting period is presented in Table 1.

WEIGHT STATUS

The predicted weight of the composite stablizer was revised based on calculated weights for engineering drawings released to date. The revised weight comparisons are shown in Table 2. The current predicted weight saving is 21.7 percent.

A weight change summary for the current reporting period is shown in Table 3. A net weight increase of 0.2 kilograms (0.5 pounds), primarily in shear webs, resulted from recalculation of weights based on the released drawing configurations. The CVS weight distribution by material is summarized in Table 4. A weight-time history for the composite stabilizer is shown in Figure 18.

TABLE 1
SUMMARY OF CRITICAL MARGINS-OF-SAFETY
FOR DC-10 COMPOSITE VERTICAL STABILIZER COMPONENTS

AMC DRAWING NUMBER	PART DESCRIPTION	MINIMUM MARGIN- OF-SAFETY	CRITICAL MODE
7840	SKIN PANEL ASSEMBLY	0.15	SHEAR-COMPRESSION INTERACTION
7845 7847 7848 7849	FRONT SPAR ASSEMBLY FORWARD CENTER SPAR ASSEMBLY AFT CENTER SPAR ASSEMBLY REAR SPAR ASSEMBLY	0.52	SHEAR STRESS IN ADHESIVE BOND- LINE AT INTERFACE BETWEEN TITANIUM FITTING AND COMPOSITE SPAR CAP — INCLUDES 133-PERCENT FITTING FACTOR
7850	FITTING - STA Z _{FR} 316 HINGE	0.92	TENSION
7851	FITTING - STA Z _{FR} 316 CANT-RIB CAP	0.07 0.01	BENDING THERMAL BOLT LOAD BEARING
7853	BASE RIB INSTALLATION	0.07	BOLT BEARING
7854	STA Z _{FR} 295 RIB INSTALLATION	0.07	WEB SHEAR AT CUTOUT
7856	BRACKET - STA Z _{FR} 316 AND 325 HINGE	ОК	STRUCTURALLY IDENTICAL TO EXISTING DC-10 PART, AMC 7031
7857	FITTING – STA Z _{FR} 351 HINGE	0.24	TENSION
7859	STA Z _{FR} 350 RIB INSTALLATION	0.02	BOLT BEARING
7862	FITTING - STA Z _{FR} 351 RIB CAP	0.12	BOLT BEARING
7863	FITTING – STA Z _{FR} 351 RIB CAP	0.02	BOLT BEARING
7871	FITTING – STA Z _{FR} 423 HINGE	0.24	TENSION
7872	FITTING STA Z _{FR} 423 RIB CAP	0.02	BOLT BEARING
7873	FITTING - STA Z _{FR} 423 RIB CAP	0.12	BOLT BEARING

TABLE 2
PRELIMINARY WEIGHT COMPARISONS
COMPOSITE VERTICAL STABILIZER

		COMPOSITE STABILIZER	FABILIZER		I V T EIV	14
	PREVIOUS ESTIMATE	STIMATE	LATEST ESTIMATE	STIMATE	STABILIZER	LIZER
ITEM	KILOGRAMS	POUNDS	KILOGRAMS	POUNDS	KILOGRAMS	POUNDS
CPAR CAPS	140.5	309.7	132.7	292.6	158.4	349.2
INTERSPAR SKIN PANELS	64.3	141.8	48.3	106.5	87.5	192.8
SPAR WEBS	40.6	89.5	52.9	116.6	62.4	137.6
INTERSPAR RIBS	51.8	114.2	63.5	140.0	62.9	149.6
ACCESS DOORS	16.6	36.6	16.6	36.6	18.5	40.7
MISCELLANEOUS STRUCTURE	13.4	29.5	13.4	29.5	28.7	63.3
GROWTH/CONTINGENCY	4.5	10.0	4.5	10.0	1	-
BOX STRUCTURE	331.7	731.3	331.9	731.8	423.3	933.2
TRAILING-EDGE SKIN AND RIBS	25.3	55.7	25.3	55.7	32.7	72.1
TOTAL - BOX AND TRAILING EDGE	357.0	787.0	357.2	787.5	456.0	1005.3
WEIGHT REDUCTION	0.66	218.3	98.8	217.8	l	1
PERCENT REDUCTION	21.7	21.7	21.7	21.7	١	1

TABLE 3 WEIGHT CHANGE SUMMARY COMPOSITE VERTICAL STABILIZER

	WEIGHT CH	IANGE
ITEM	KILOGRAMS	POUNDS
SPAR CAPS RELEASE OF PRODUCTION DRAWINGS REFLECT CURRENT WEIGHTS	-7.8	-17.1
INTERSPAR SKIN PANELS DELETION OF ANTENNA PANEL FASTENERS (2.4 LB). ESTIMATED WEIGHT FOR PANEL HAS BEEN REPLACED BY COMPOSITE PANEL CORE (HONEYCOMB).	-16.0	-35.3
RELEASE OF SKIN PANEL DRAWING REFLECTS THE CURRENT WEIGHT. IN ADDITION, TRANSFER OF SOME SKIN PANEL TO INTERSPAR RIB WHERE APPLICABLE.		
SPAR WEBS RELEASE OF SPAR WEB PRODUCTION DRAWINGS REFLECT CURRENT WEIGHT	+12.3	+27.1
INTERSPAR RIBS PARTIAL RELEASE OF PRODUCTION DRAWINGS PROVIDES CURRENT DESIGN AND WEIGHTS FOR RIBS	+11.7	+25.8
TOTAL WEIGHT CHANGE	+0.2	+0.5

TABLE 4
WEIGHT DISTRIBUTION BY MATERIAL
COMPOSITE VERTICAL STABILIZER

									Σ	IATERI/	MATERIAL WEIGHT	ЭHТ				i				
	GRAPHITE- EPOXY	HITE.	TITAL	TITANIUM ADHE	ADHE	SIVE	NOMEX HONEYCOMB		SYNTACTIC FOAM	CTIC	ALUMINUM	NOM	STEEL		FASTENERS		EXTERIOR FINISH	H IOH	TOTAL	IAL
ITEM	KG	1.8	KG	EB.	KG	F.B	KG	LB	KG	87 FB	KG	F.B	KG	LB	KG	LB	KG	LB	KG	LB
SPAR CAPS	88.6	195.4	6.88	0.88	-										4.2	9.2			132.7	292.6
SKIN PANELS	26.2	57.7			9.9	14.6	5.3	11.6	2.3	5.1	7.9*	17.5*							48.3	106.5
SPAR WEBS	51.6	51.6 113.7		•	0.3	9.0	0.7	7.5	0.4	9.0									52.9	116.6
RIBS	52.0	52.0 114.7			0.2	0.5	0.1	0.2	0.1	0.2	7.3	16.2			3.7	8.2			63.5	140.0
ACCESS DOORS	10.5	23.1			7	2.5	0.5	1.2	4.0	6.0	1.6*	3.5*			2.4	5.4			16.6	36.6
MISCELLANEOUS STRUCTURE	4.6	10.1			9.0	£. 1					4.2	9.3	5.1	3.2	0.2	4.0	2.4	5.2	13.4	29.5
GROWTH/ CONTINGENCY																			*	:
BOX SUBTOTAL	233.5	514.7	39.9	88.0	8.8	19.5	9.9	14.5	3.2	7.0	21.0	46.5	1.5	3.2	10.5	23.2	2.4	5.2	327.4	721.8
TRAILING EDGE	13.8	30.4			4.0	8.0	0.1	0.3	0.3	9.0	8.2	18.0	0.7	1.6	1.5	3.3	0.3	0.7	25.3	55.7
TOTAL WEIGHT	247.3	247.3 545.1	39.9	88.0	9.2	20.3	6.7	14.8	3.5	7.6	29.2	64.5	2.2	4.8	12.0	26.5	2.7	5.9	352.7	5.777

*ALUMINUM SPRAY COATING **GROWTH/CONTINGENCY ALLOWANCE OF 10 POUNDS NOT INCLUDED

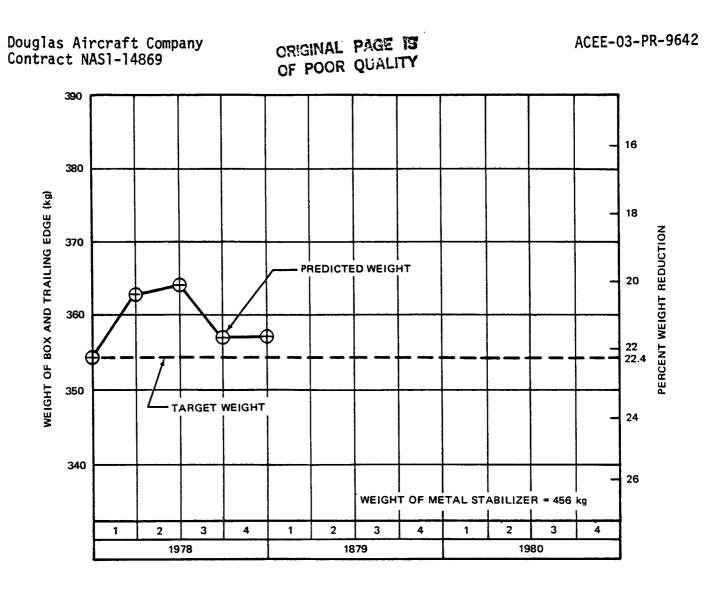


FIGURE 18. COMPOSITE VERTICAL STABILIZER WEIGHT TREND

SECTION 3 CONCEPT DEVELOPMENT COMPONENTS

The concept development component testing for the program was completed during the reporting period with completion of the Z5943434 sine-wave sparweb test and the Z3943451 lightning protection system tests. Previously completed tests in this component group included stiffened compression and shear panel tests (References 5 through 8) a honeycomb stiffened spar-web test (References 5 and 8), and galvanic effects tests (Reference 5).

Test setups and results of the sine-wave spar-web and lightning protection system tests are described in this section.

SPAR-WEB COMPONENT

The Z5943434 sine-wave shear web component was redesigned and remade as a result of previous test component failure (see Reference 5). The redesign provided for two flat areas in the web for incorporation of 11.4 cm (4.5 inch) diameter access openings. The component included one access opening having flanged edges as shown in Figure 19. The other flat area in the specimen web was left blank to permit subsequent test evaluation of an unflanged cutout, see Figure 20.

The testing was accomplished in two steps. The first test subjected the component to ultimate design shear loading (600 pounds per inch) in the web. Load was applied to the component as a simply supported beam as shown in Figure 19. The component successfully sustained the loading without failure. The maximum tensile strain in the flange of the cutout was 2495 microinches per inch at 152 percent test limit load (TLL).

The component was removed from the test fixture and a circular opening cut in the blank web (Figure 21). The component was re-installed in the test fixture and the test loading sequence repeated. The web failed at the unflanged cutout at 104 percent TLL (420 pounds per inch). Failure resulted from

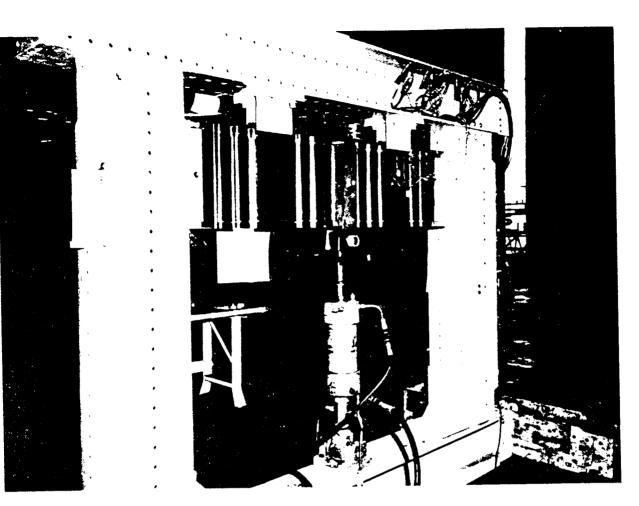
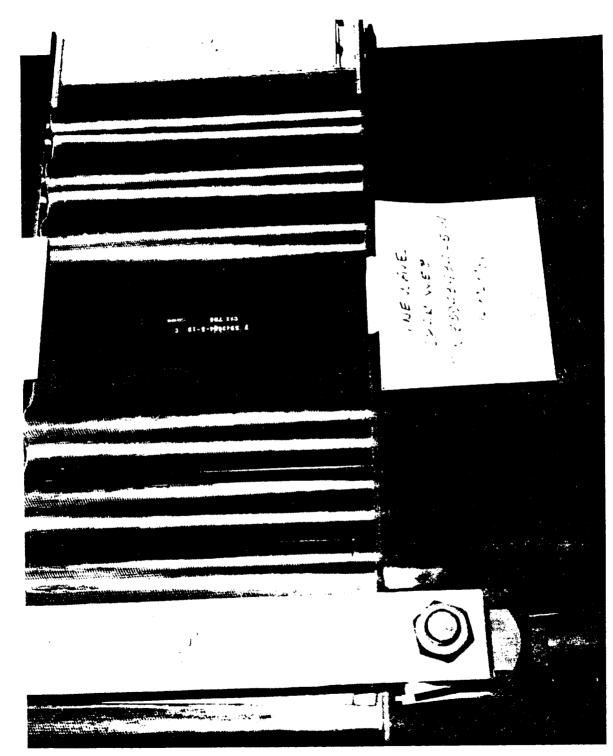


FIGURE 19. SINE-WAVE SHEAR WEB COMPONENT IN TEST FIXTURE

FIGURE 20. FLAT AREA IN Z5943434-501 SINE WAVE SPAR WEB



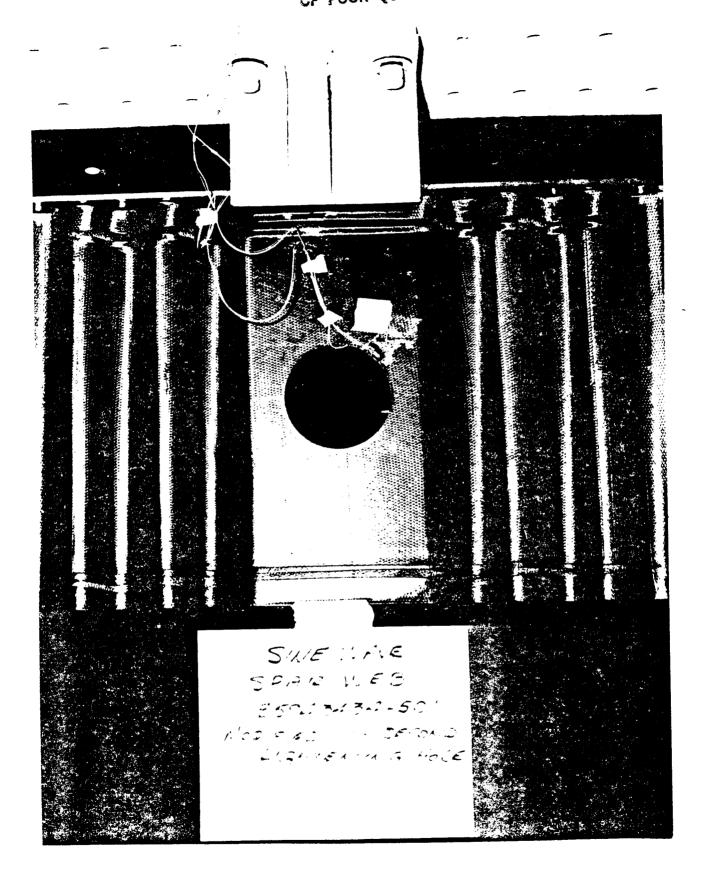


FIGURE 21. UNFLANGED ACCESS OPENING IN SINE-WAVE SHEAR WEB COMPONENT

excessive circumferential tensile strains at the edge of the cutout as shown in Figure 22. A tensile strain of 5925 microinches per inch was recorded at the edge of the unflanged cutout at failure. The test indicated the need for local reinforcing flanges at the shear-web cutouts to meet design shear load requirements.

LIGHTNING TEST PANEL

Current transfer and lightning restrike tests were completed on the Z3943451 lightning evaluation panel. The 5-mil aluminum spray coating was adequate for all lightning current transfer and restrike tests.

The test panel configuration is illustrated in Figure 23. The metal tabs at the left hand edge of the panel represented candidate joint configurations at the metal leading edge structure interface with the graphite stabilizer box-structure. The joint was effected with flush screws at normal installation torque and at two higher torque values. The metal tabs at the right-hand edge of the panel represented candidate joint configurations at the trailing edge structure interface with the graphite stabilizer box-structure in the vicinity of a rudder hinge bracket. The simulation of the metal pianohinge which will attach the trailing edge structure to the graphite box structure was attached with various rivet combinations. Tabs 1A, 1B, 3A, 3B, 5A and 5B all had metal-to-metal contact between the tab and the metal spray coating. The remaining tabs had a faying surface seal (PRC 1431G) between the tabs and the metal spray coating.

<u>Lightning Current Transfer Tests</u>

During a severe 200 kiloampere (KA) peak-current lightning-strike at the tip of the stabilier, the current transferred in the skins of the composite vertical stabilizer will be approximately 0.5 KA per centimeter of width. The 9.5 centimeter wide joints of the test panel, therefore, must be capable of transferring about 5 KA. The current transfer tests were made with the panel mounted in the test fixture as shown in Figure 24.

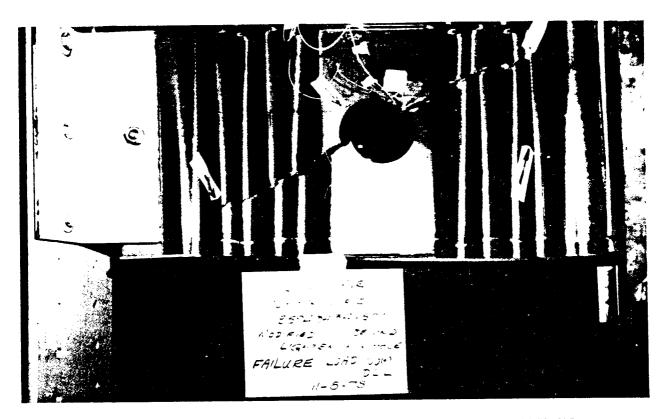


FIGURE 22. FAILURE OF SHEAR WEB AT UNFLANGED ACCESS OPENING

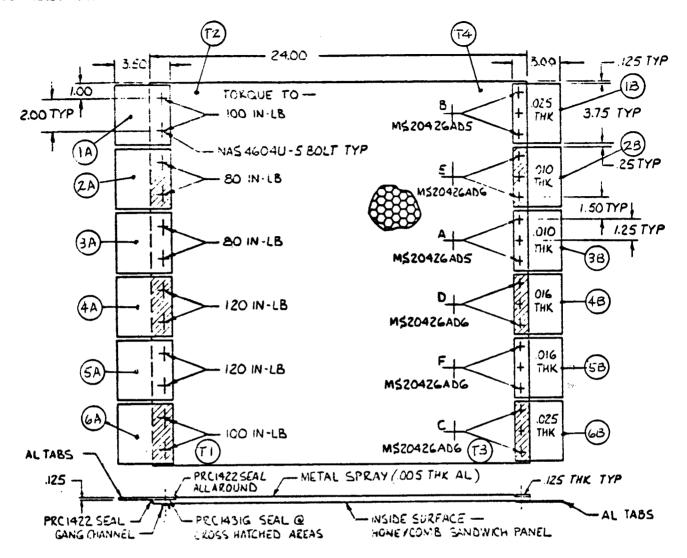


FIGURE 23. SKETCH OF LIGHTNING PANEL SHOWING TEST POINTS



FIGURE 24. TEST SETUP – SIMULATED LIGHTNING CURRENT TRANSFER TEST

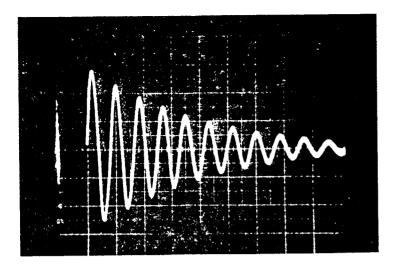
The initial test was a discharge of 5.9 KA peak current from tab 1A to 1B. The action integral was .0489 x 10^4 ampere squared seconds. The same test was made from tab 6A to 6B. Additional tests were made between the other tab pairs with peak currents of 11.6 KA and 16.9 KA. Typical current waveforms for the discharges made during the tests are shown in Figure 25. The discharge paths, current levels, and resistance values are tabulated in Table 5.

There was no measureable difference in the resistance values obtained before and after test or between the various joints. These tests indicate that all joint configurations tested are acceptable for transferring the necessary lightning current densities. The joints with the faying surface seal are preferred, because of the greater corrosion resistance.

Simulated Lightning Restrike Tests

Photographs of the test panel mounted in the test fixture before and after the 116 KA lightning restrike test are shown in Figure 26. The test waveform is shown in Figure 27. The action integral of 0.55 x 106 ampere squared seconds was greater than the required test value of 0.25 x 106 ampere square seconds and the peak current of 116 KA was greater than the required level of 100 KA. The discharge path was from the center of the panel to tabs 1B, 2B, 3B, 4B, 5B, and 6B which were clamped together. There were no changes in the resistances of the panel caused by the high current discharge (see Table 6). Photographs of the test panel showing the area where the metal spray was vaporized are shown in Figures 28 and 29.

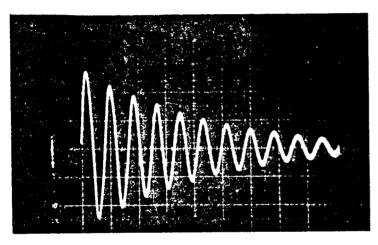
Upon completion of the lightning tests, the lightning panel was examined using x-ray and ultrasonic through transmission NDI techniques. This examination revealed that the high current lightning strike to the panel caused a delamination of approximately 7.5 centimeters (3 inches) diameter between the outer facing and the honeycomb core in the area where the metal spray was vaporized. The damage was repairable and considered to be acceptable for composite vertical stabilizer from a safety standpoint.



VERTICAL SCALE: 2.11 KA PER DIVISION HORIZONTAL SCALE: 10 µSEC PER DIVISION

PEAK CURRENT: 5.9 KA

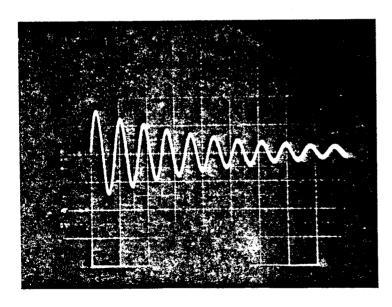
DISCHARGE PATH: POINT 1A TO POINT 1B



VERTICAL SCALE: 4.22 KA PER DIVISION HORIZONTAL SCALE: 10 µSEC PER DIVISION

PEAK CURRENT: 11.6 KA

DISCHARGE PATH: POINT 2A TO POINT 2B



VERTICAL SCALE: 10.55 KA PER DIVISION HORIZONTAL SCALE: 10 µSEC PER DIVISION

PEAK CURRENT: 16.9 KA

DISCHARGE PATH: POINT 1A TO POINT 1B

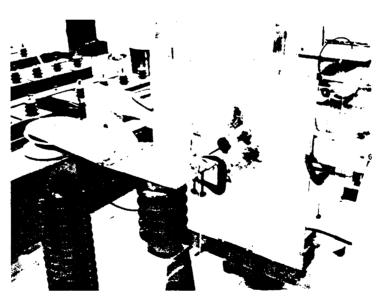
FIGURE 25. TYPICAL TEST WAVEFORMS - CURRENT TRANSFER TESTS

TABLE 5
PANEL RESISTANCE MEASUREMENTS
CURRENT TRANSFER TESTS

DISCHARGE		RESISTANCE	RESISTANCE (MILLIOHMS)
CURRENT (KILOAMPERES)	DISCHARGE PATH	CHECK POINTS	BEFORE TEST	AFTER TEST
5.9	1A-1B	1A-1B 2A-2B 3A-3B T1-T2 T3-T4 T2-T4 1A-T2 1B-T4	10.0 6.0 4.5 6.5 5.0 6.0 3.0 5.0	6.0 5.0 5.0 7.0 9.0 7.0 2.0 2.0
	6A-6B	4A-4B 5 A-5B 6A-6B T1-T2 T3-T4 T1-T3 6A-T1 6B-T3	9.0 13.0 27.0 7.0 9.5 8.0 8.0 20.0	4.5 4.0 5.0 6.5 5.0 7.0 1.0 3.0
11.6	2A-2B	1A-1B 2A-2B 3A-3B T1-T2 T3-T4 2A-T2 2B-T4	6.0 5.0 5.0 7.0 9.0 3.0 4.0	6.0 4.5 4.5 6.5 7.5 3.0 3.5
	3 A-3B	2A-2B 3A-3B 4A-4B T1-T2 T3-T4 3A-T2 3B-T4	4.5 4.5 4.5 6.5 7.5 4.0 4.5	4.5 4.0 4.5 6.5 8.0 4.0 4.5
16.9	1A-1B	1A-1B 2A-2B 3A-3B T1-T2 T2-T4	6.0 5.0 4.5 7.0 7.0	5.5 5.0 4.5 6.5 7.0
	6A-6B	4A-4B 5A-5B 6A-6B T1-T2 T1-T3	4.5 4.0 5.0 8.5 7.0	4.0 4.0 5.0 6.5 6.0
	2 A-2B	1A-1B 2 A-2B 3A-3B T1-T2	6.0 4.5 4.5 6.5	6.0 4.5 4.5 7.0
	3A-3B	2A-2B 3A-3B 4A-4B T1-T2	4.5 4.0 4.5 6.5	4.5 4.0 4.0 2.0
	4 A-4B	3A-3B 4A-4B 5A-5B T1-T2	4.0 4.5 4.5 6.5	4.0 4.0 4.5 6.5
	5A-5B	4A-4B 5A-5B 6A-6B T1-T2	4.0 4.5 5.5 6.5	4.0 4.0 5.5 4.0

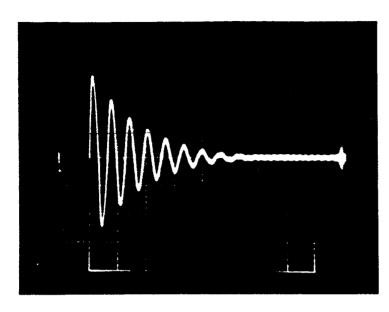


BEFORE TEST



AFTER TEST

FIGURE 26. TEST SETUP — SIMULATED LIGHTNING RESTRIKE TEST TO COMPOSITE PANEL (116-KA PEAK)



VERTICAL SCALE: 40 KA PER DIVISION HORIZONTAL SCALE: 100 μSEC PER DIVISION PEAK CURRENT: 116 KA

FIGURE 27. TEST WAVEFORM - SIMULATED LIGHTNING RESTRIKE TEST TO CENTER OF PANEL

TABLE 6 PANEL RESISTANCE MEASUREMENTS LIGHTNING RESTRIKE TEST

		RESISTANCE	RESISTANCE	(MILLIOHMS)
DISCHARGE CURRENT (KILOAMPERES)	DISCHARGE PATH	CHECK POINTS	BEFORE TEST	AFTER TEST
116.0	CENTER	1A-1B	6.0	6.0
110.0	OF	2 A-2B	5.0	5.0
	PANEL	3A-3B	4.5	5.0
	то	T1-T2	7.0	7.0
	TABS	T3-T4	9.0	7.5
	1B, 2B,	T2-T4	7.0	4.5
	3B, 4B,	1B-T4	2.5	3.0
	5B, 6B	4 A-4B	4.5	3.0
	35, 35	5A-5B	4.0	5.5
		6 A-6B	5.0	5.5
		T1-T3	7.0	7.0
		6B-T3	3.0	3.0

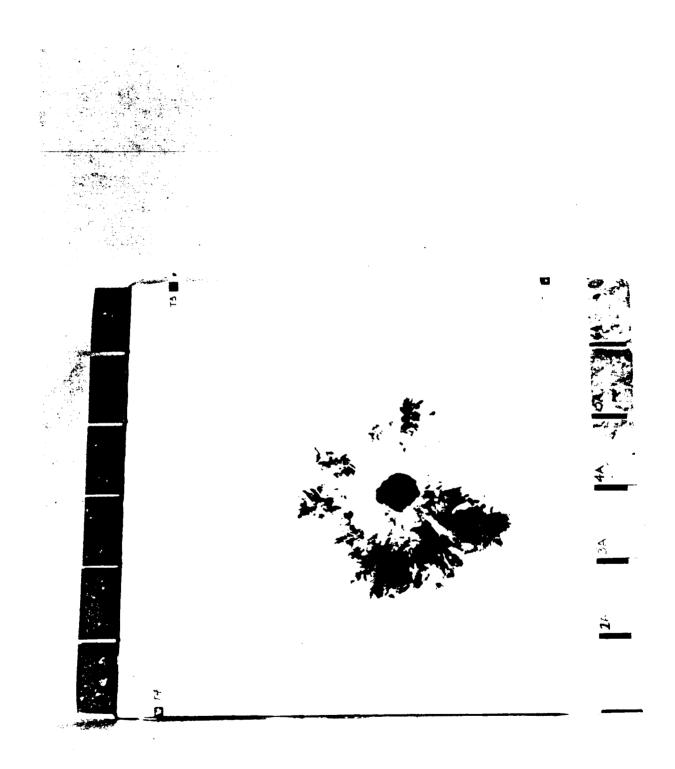


FIGURE 28. COMPOSITE PANEL AFTER 116-KA SIMULATED LIGHTNING RESTRIKE TEST

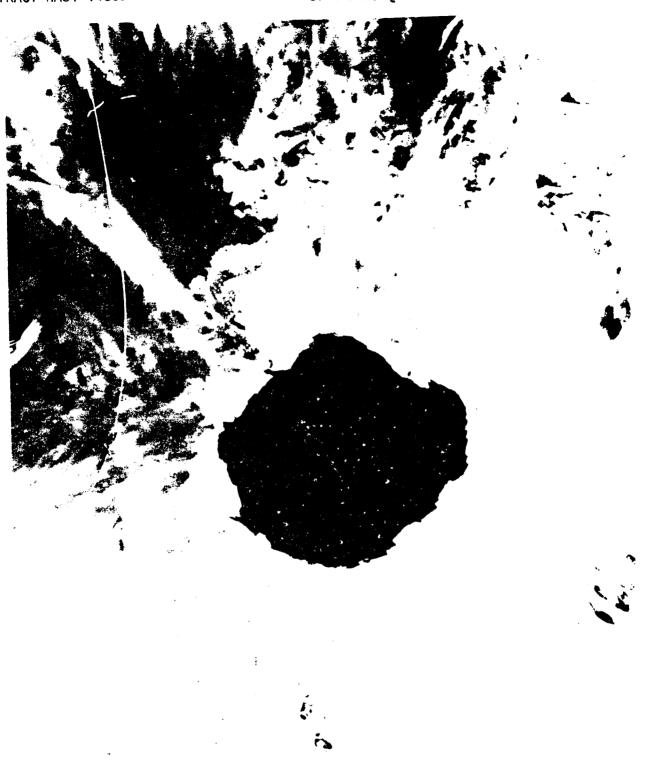


FIGURE 29. COMPOSITE PANEL AFTER 116-KA SIMULATED LIGHTNING RESTRIKE TEST (CLOSE-UP VIEW)

The metal tabs were all removed from the lightning panel for visual examination. There was no burning or other evidence of lightning damage at any of the joints. One joint that had only an edge seal was filled with fluid. The fluid entered the joint through a break in the seal when the panel was submerged for ultrasonic test. This incident emphasized the importance of using a faying surface seal to eliminate moisture intrusion and the resulting corrosion.

SECTION 4 JOINT DEVELOPMENT COMPONENTS

The joint development component testing for the program will be completed on successful testing of the Z5943453 actuator (-1) and tie-rod (-501) rudder fitting components. These tests will simulate critical load conditions in the actuator ribs and tie-rod ribs discussed previously in Section 2. Detail Design. Previously completed tests in this component group included the leading edge splice tests, leading edge attachment fatigue tests, spar cap to cover panel attachment tests, and major attach fitting tests (see References 7 through 9).

Detail part fabrication was completed for both the Z5943453-1 and -501 specimen configurations during the reporting period and final assembly was completed for the -1 (actuator) component. The completed -1 component is shown in Figures 30 and 31. The -501 component during setup for final assembly is shown in Figure 32. Test fixture installation and testing for both components will be completed in January 1979. Fabrication of the full-size rib tooling for the CVS will not be started until these tests are successfully completed.

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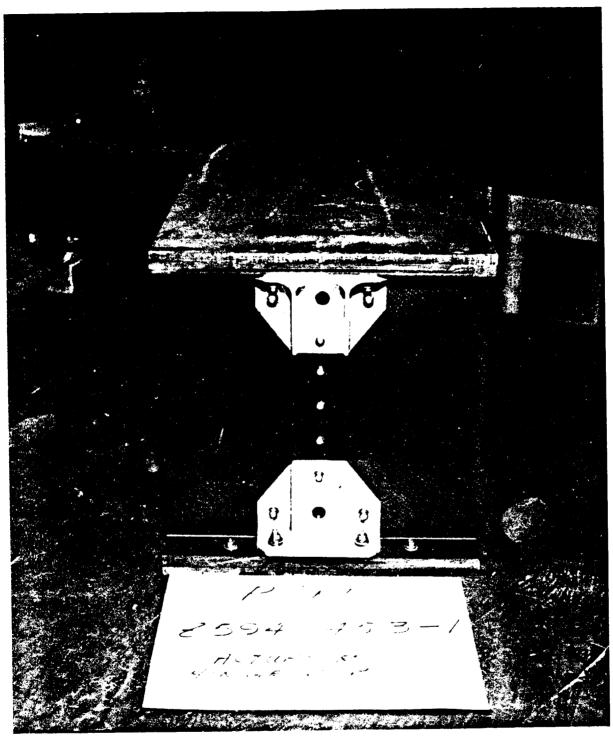


FIGURE 30. Z5943453-1 ACTUATOR HINGE RIB COMPONENT — VIEW LOOKING FORWARD AT SIMULATED REAR SPAR



FIGURE 31. Z5943453-1 ACTUATOR HINGE RIB COMPONENT — VIEW SHOWING INTERNAL FITTING AND SIMULATED SINE-WAVE RIB WEB

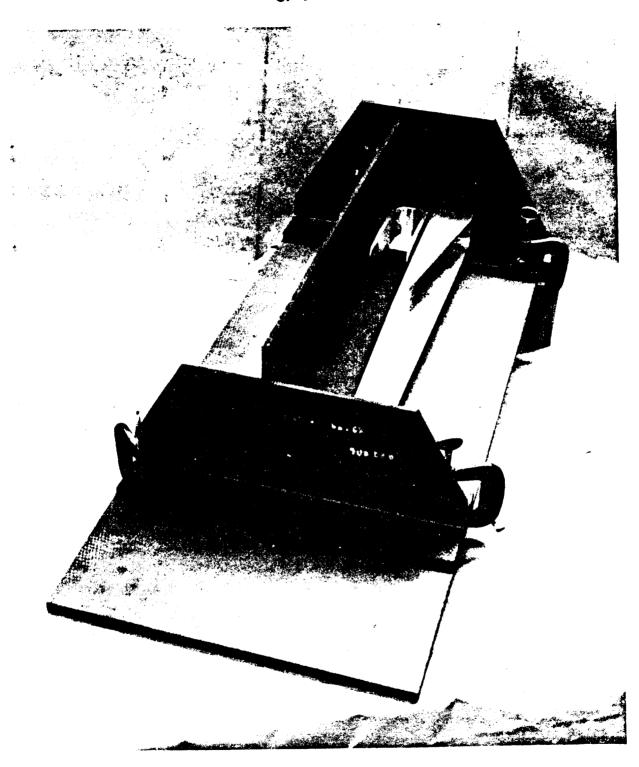


FIGURE 32. Z5943453-501 TIE-ROD RUDDER FITTING COMPONENT DURING SETUP FOR FINAL ASSEMBLY

SECTION 5 MECHANICAL PROPERTY TESTING

Mechanical property testing for the program was completed during the reporting period with the completion of laminate fatigue testing and fracture mechanics data testing on a variety of damaged and debonded specimens. Previously completed laminate property tests for static tension and compression allowables, fatigue data, and bolt bearing and shear-out data were reported in References 5 and 8.

Test conditions and results for the completed fatigue tests and the fracture mechanics tests are described in this section.

LAMINATE PROPERTY TESTS

Fatigue testing was completed on the Z3943432-505 quasi-isotropic laminate sandwich beam specimens. The results of these tests are plotted in Figures 33 through 36. The test data are tabulated in Tables B-1 and B-2 in Appendix B.

A plot of the fatigue data at a stress ratio of R = -1.0 is shown in Figure 33. Tests were conducted at temperatures of $219^{\circ}K$ ($-65^{\circ}F$), ambient, and $350^{\circ}K$ ($170^{\circ}F$). All specimens included a central hole of 0.635 cm (0.250 in) diameter providing a width-to-hole-diameter ratio of 6. The average fatigue strength exhibited by the specimens tested at ambient and $350^{\circ}K$ was approximately $185^{\circ}M$ megapascals ($26800^{\circ}M$) at the one-life equivalent of $130,000^{\circ}M$ load cycles. The specimens tested at $219^{\circ}K$ exhibited a somewhat higher average fatigue strength of $225^{\circ}M$ megapascals ($32700^{\circ}M$) or about a $22^{\circ}M$ percent increase in fatigue strength.

A plot of the fatigue data at a stress ratio of R = 0.05 is shown in Figure 34. Tests were conducted at temperatures of 219°K, ambient and 350°K as before. These specimens also included a central hole of 0.635 cm. The data at this

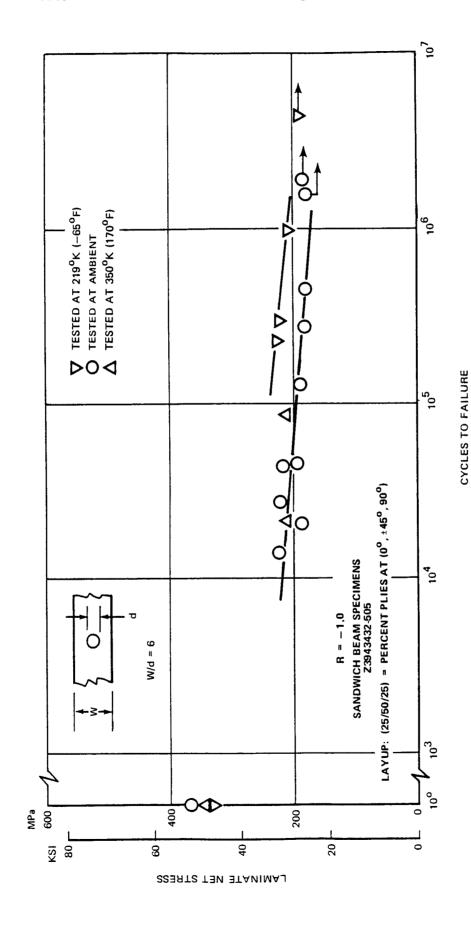


FIGURE 33. FATIGUE CHARACTERISTICS OF T300/5208 GRAPHITE/EPOXY LAMINATES

FIGURE 34. FATIGUE CHARACTERISTICS OF T300/5208 GRAPHITE/EPOXY LAMINATES

400

8

MPa 600 F

> KSI 80 g

61

200

40

LAMINATE NET STRESS

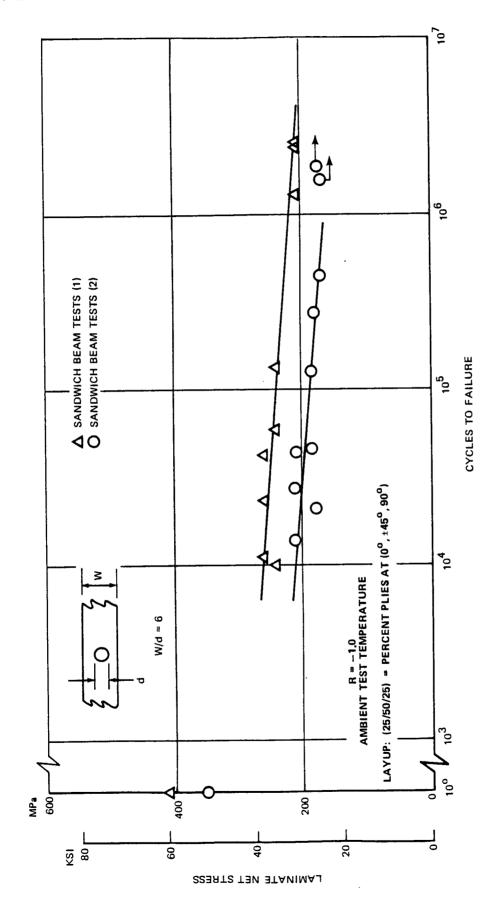
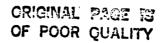


FIGURE 35. FATIGUE CHARACTERISTICS OF T300/5208 GRAPHITE/EPOXY LAMINATES

UNIDIRECTIONAL TAPE, DATA FROM REFERENCE 10. B1-WOVEN BROADGOODS, DATA FROM CVS PROGRAM (TABLE B-1, APPENDIX B)

(2)



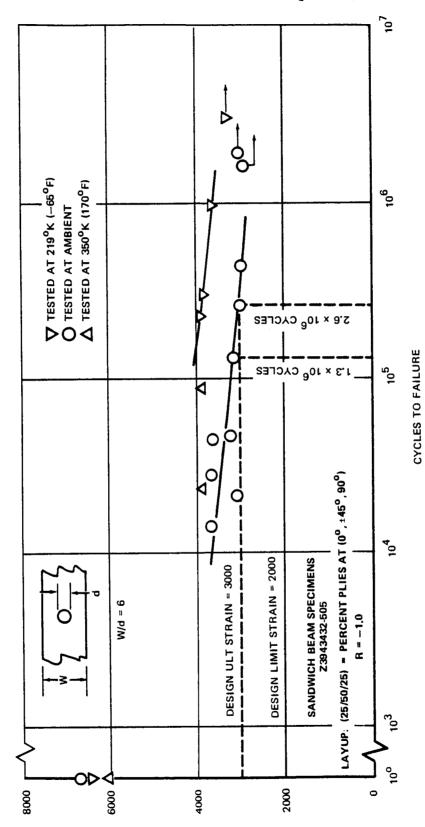


FIGURE 36. FATIGUE CHARACTERISTICS OF T300/520B GRAPHITE/EPOXY LAMINATES

stress ratio do not show the drop in fatigue strength with increasing load cycles that was evident at R = -1.0. The data all fall within the scatter band of the static tensile strength data obtained on similar sandwich beam specimens tested in the same environments (Reference 5). It is apparent that cyclic loading at a stress ratio of R = 0.05 has a negligible effect on laminate strength.

The results of the ambient fatigue tests and the results of tests on similar sandwich beams from the DC-10 composite rudder program (Reference 10) are compared in Figure 35. The test results from the rudder program are for T300/5208 laminates made from unidirectional tape as opposed to the bi-woven broadgoods used in the composite stabilizer sandwich beams. The data show a difference in fatigue strength between the two types of prepreg of approximately 39 percent at 130,000 cycles in favor of the unidirectional tape. Unidirectional tape material has now been specified for the CVS spar cap laminates.

The same data in terms of strain levels for each beam specimen at failure are shown in Figure 36. No failures occurred below the design limit microstrain of 2000 even for those cases where the load cycles exceeded the one-life equivalent of 130,000 cycles. Strain levels were computed using the modulus values from the static tensile strength data (Reference 5).

It was concluded that the selected quasi-istropic laminate of T300/5208 unidirectional tape will provide more than adequate fatigue life for the expected in-service loads and environments of the CVS.

FRACTURE MECHANICS TESTS

Fatigue testing was completed on the Z3943442-1 debond tension specimens and on the Z3943442-505 damaged tension specimens. The test data are tabulated in Tables B-3 and B-4 in Appendix B.

Photographs showing a typical test set-up and a close-up of the anti-buckling plates used to prevent lateral instability at the laminate under compression load are shown in Figures 37 and 38.

A plot of the fatigue data for the Z3943442-1 debond specimens is shown in Figure 39. All testing was conducted at a stress ratio of R = -1.0 and at temperatures of 219°K, ambient, and 350°K. All specimens included a laminate debond area of approximately 1.27 cm (0.50 in.) diameter located in the center of the test region. All specimens were subjected to 130,000 load cycles (equivalent to one life-time of structural loading) at the design limit microstrain of 2000 prior to undergoing the additional fatigue loading shown. No failures occurred during the first life cycle tests and no changes to the debond area were noted.

The second life cycle tests were run at higher load levels to establish the fatigue characteristics of the laminate in the presence of debonds. None of the debond specimens failed through the delaminated area but through the minimum section adjacent to the tangent point of the 4.0 inch shoulder radius. The X-ray, Figure 40, clearly indicates a significant stress concentration in this area. An opaque liquid was applied to the specimen edges to accent the damage. Figure 41 shows that no significant, consistent difference exists between the results for the moisture conditioned specimens and those tested "dry".

A plot of the fatigue data for the Z3943442-505 damaged specimens is shown in Figure 42. All testing was conducted at a stress ratio of R = -1.0 and at temperatures of 219°K, ambient, and 350°K as before. All specimens included a damaged area in the center of the test region to provide a w/d ratio (specimen width to damage size) of 6.0. This ratio was selected to provide a similar stress concentration effect to that in the sandwich beam fatigue test specimens. The tests were conducted at strain levels ranging from 2000 microstrain (design limit strain) down to 1500 microstrain (84 occurrances in one lifetime). None of the specimens failed at this strain level even when tested for 260,000 cycles (two lifetimes).

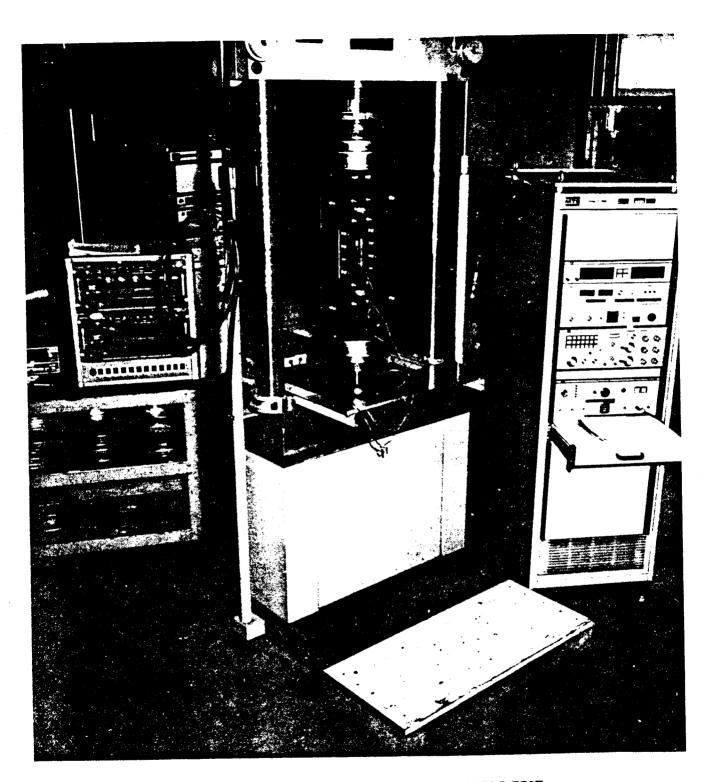


FIGURE 37. TEST SETUP FOR DAMAGE AND DEBOND TEST

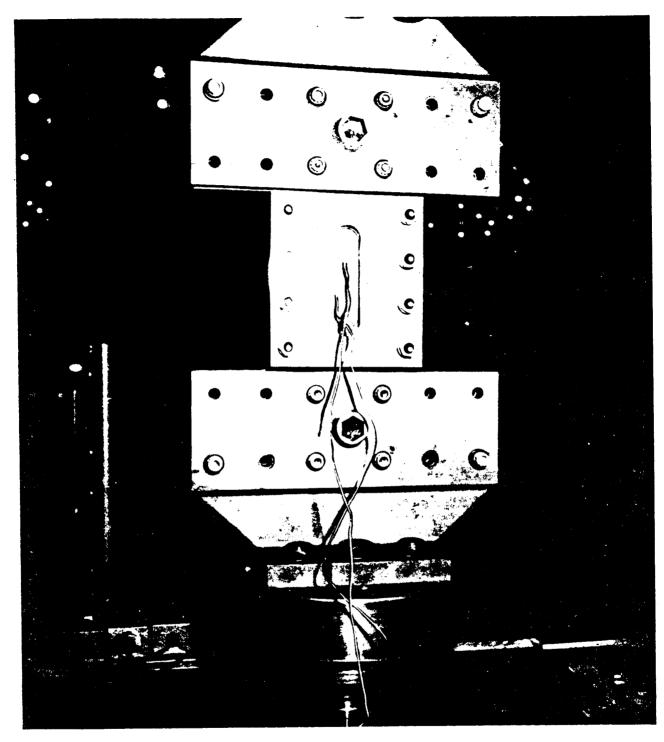


FIGURE 38. CLOSE-UP VIEW OF PLATES USED TO PREVENT BUCKLING OF DAMAGE AND DEBOND SPECIMENS

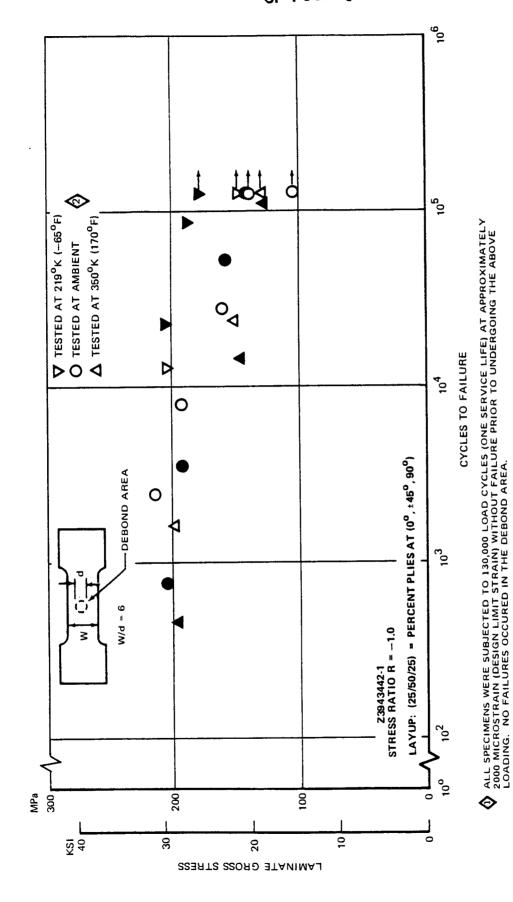


FIGURE 39. FATIGUE TEST RESULTS FOR LAMINATE TENSION SPECIMENS WITH DEBONDS

SHADED SYMBOLS INDICATE THOSE SPECIMENS THAT WERE PRETEST MOISTURE CONDITIONED. REMAINDER WERE TESTED DRY.

③

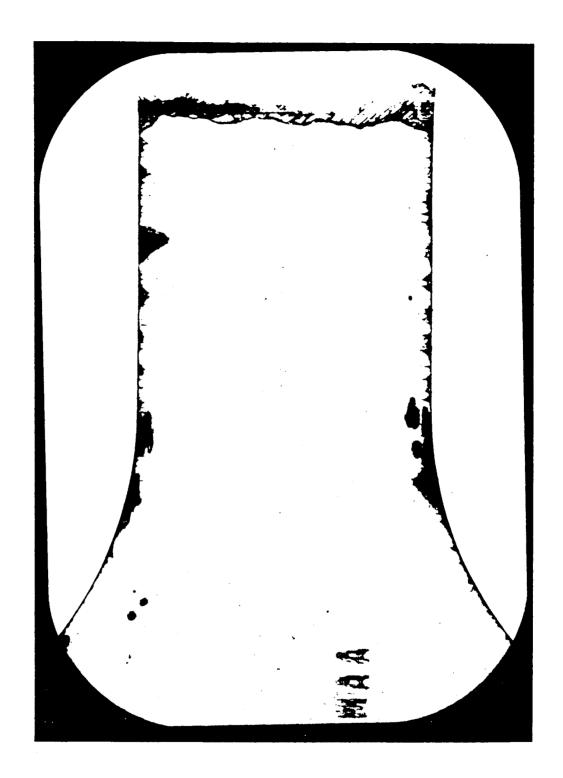


FIGURE 40. X-RAY OF DEBOND SPECIMEN SHOWING FATIGUE DAMAGE

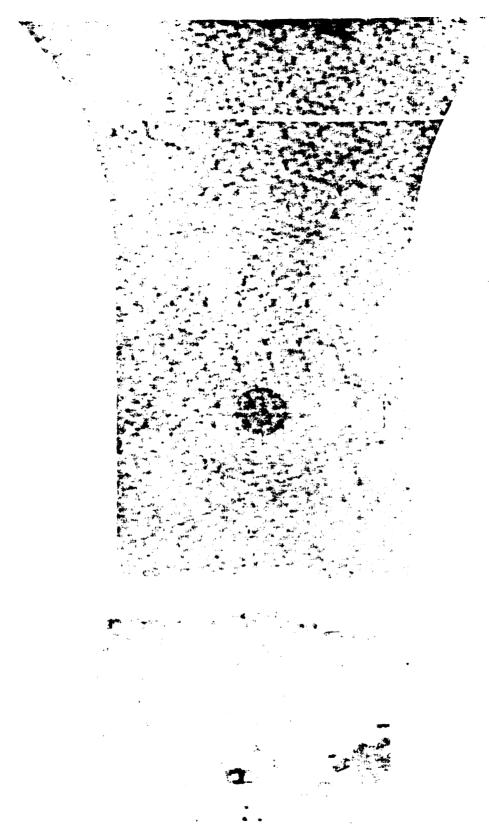


FIGURE 41. X-RAY OF DEBOND SPECIMEN AFTER FATIGUE TEST SHOWING STATIC FAILURE

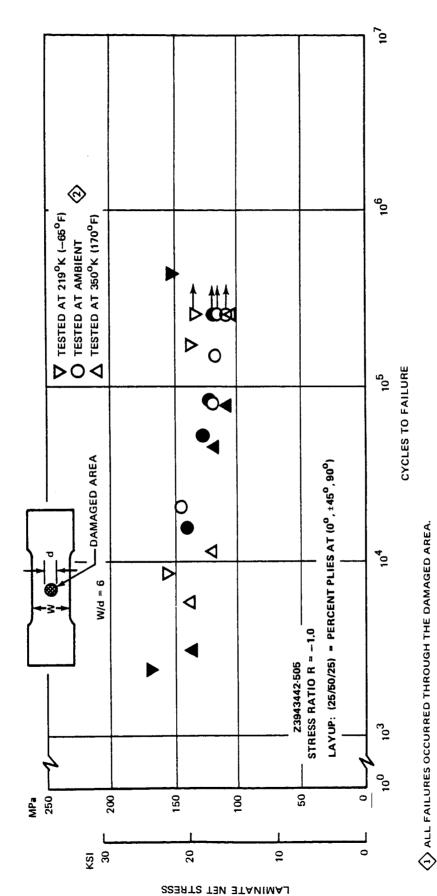


FIGURE 42. FATIGUE TEST RESULTS FOR DAMAGED LAMINATE TENSION SPECIMENS 💠

SHADED SYMBOLS INDICATE THOSE SPECIMENS THAT WERE PRETEST MOISTURE CONDITIONED. REMAINDER WERE TESTED DRY.

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Trial impact energy tests were conducted on a piece of 0.052 inch thick quasi-isotropic laminate to determine what energy levels would be required to produce the desired damage level (Figure 43). These tests were conducted using a standard Gardner impact tester with a 0.50 inch diameter anvil. The desired level of damage was a 0.50 inch diameter area on the impact side with complete penetration through the laminate. The energy levels were varied from 10 inch-pounds to 70 inch-pounds. An energy level of 40-inch-pounds produced the desired damage and this level was used in damaging the Z3943442-505 test specimens.

Fatigue damage sustained by the damage specimens progressed fairly rapidly under cyclic load with failures occurring through the minimum net section of the damage area. Damage propagation initially appeared as surface pitting accompanied by a brownish discoloration of the epoxy with subsequent localized buckling of the graphite woven fabric. Figures 44 and 45 show typical fatigue effects. Figures 46, 47, and 48 show typical failures of the damage and debond specimens together with C-scan records after test.

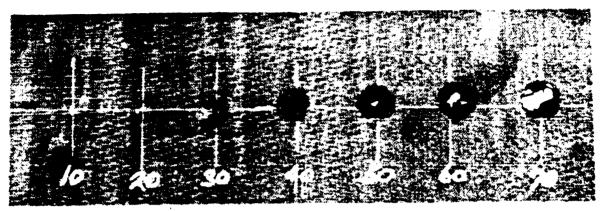
The results of the damaged specimen fatigue tests and the sandwich beam fatigue tests are compared in Figure 49. An apparent 36.6 percent reduction in strength is indicated at 130,000 cycles from 185 megapascals (26800 psi) for the sandwich beams (drilled holes) to 117 megapascals (16990 psi) for the impact damaged specimens. The theoretical stress concentration factor from Reference 11 for the sandwich beams is calculated as follows:

$$k_{t_e} = 2 + (1 - d/w)^3 = 2 + (1 - 0.25/1.50)^3 = 2.58$$

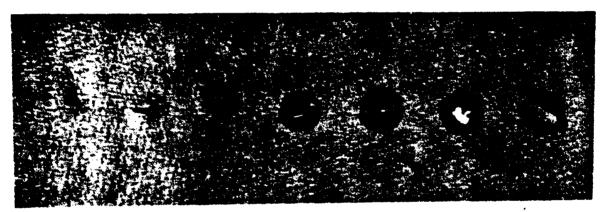
The composite stress concentration factor from Reference 11 is:

$$k_{t_c} = 0.73 + 0.27 k_{t_e} = 0.73 + 0.27 \times 2.58 = 1.43$$

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A. IMPACT SIDE



B. REVERSE SIDE

FIGURE 43. TRIAL IMPACT ENERGY TESTS

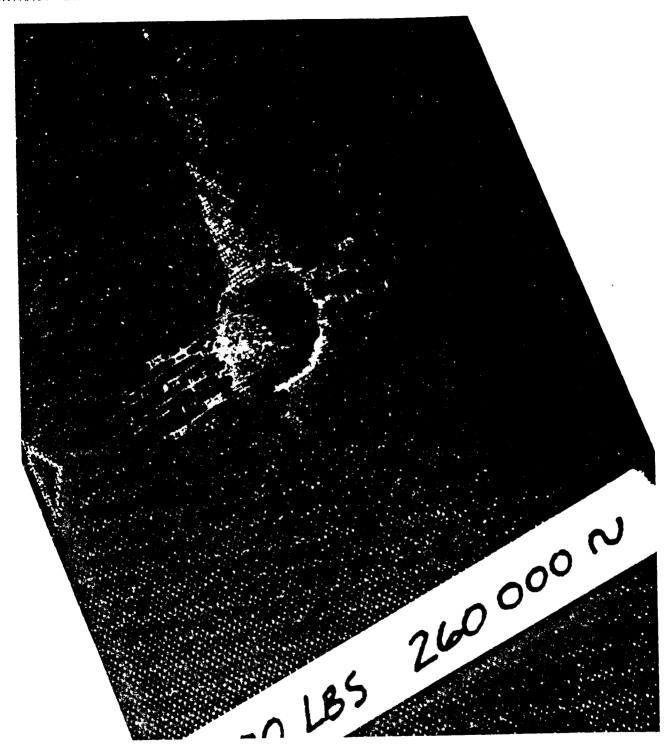


FIGURE 44. CLOSE-UP VIEW SHOWING FATIGUE DAMAGE IN IMPACT-DAMAGED SPECIMEN

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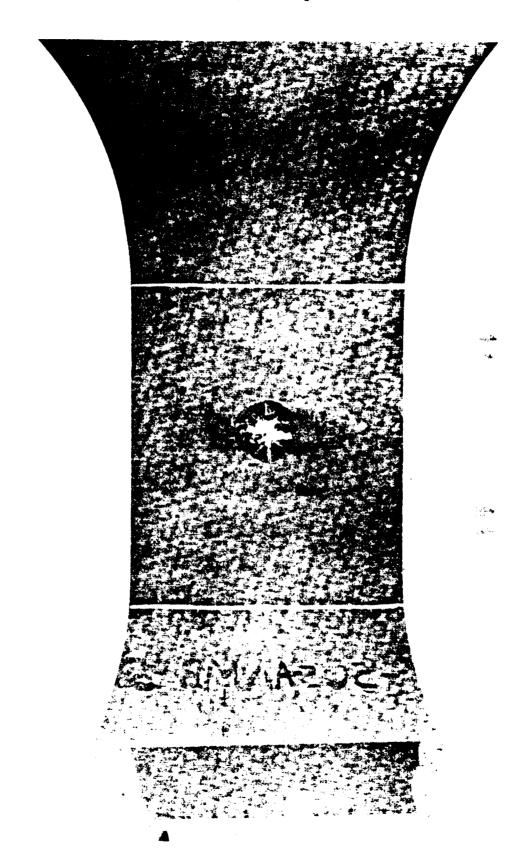


FIGURE 45. X-RAY OF IMPACT-DAMAGED SPECIMEN SHOWING FATIGUE DAMAGE

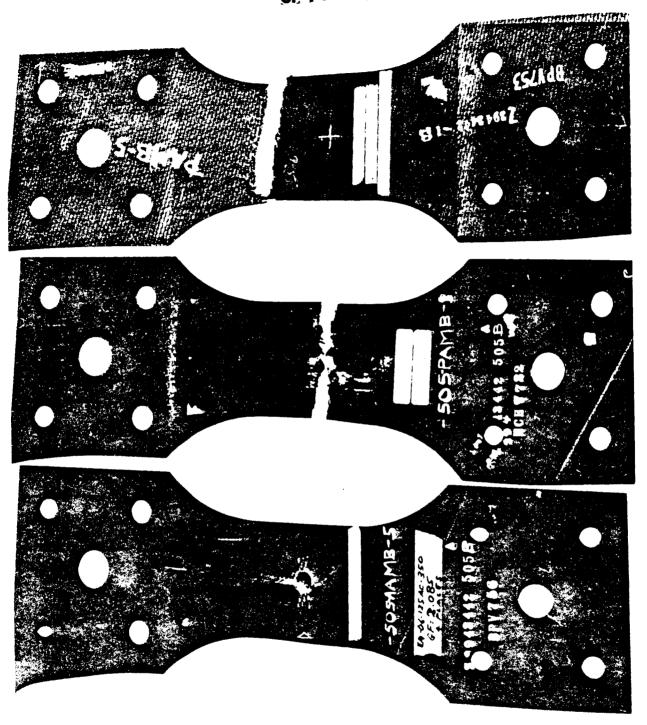


FIGURE 46. TYPICAL FAILURE MODES OF DAMAGE AND DEBOND SPECIMENS

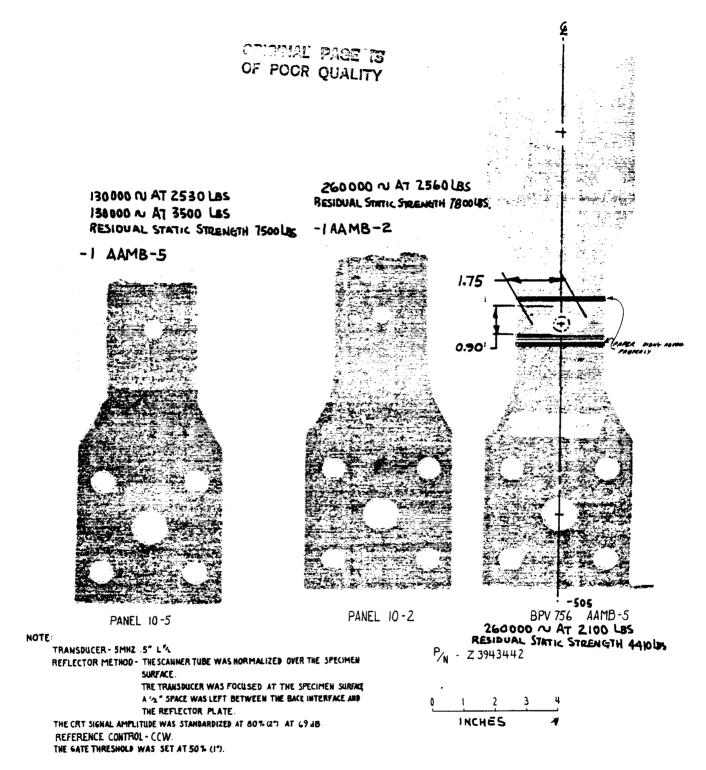


FIGURE 47. TYPICAL C-SCANS SHOWING FATIGUE DAMAGE SUSTAINED BY DAMAGE AND DEBOND SPECIMENS

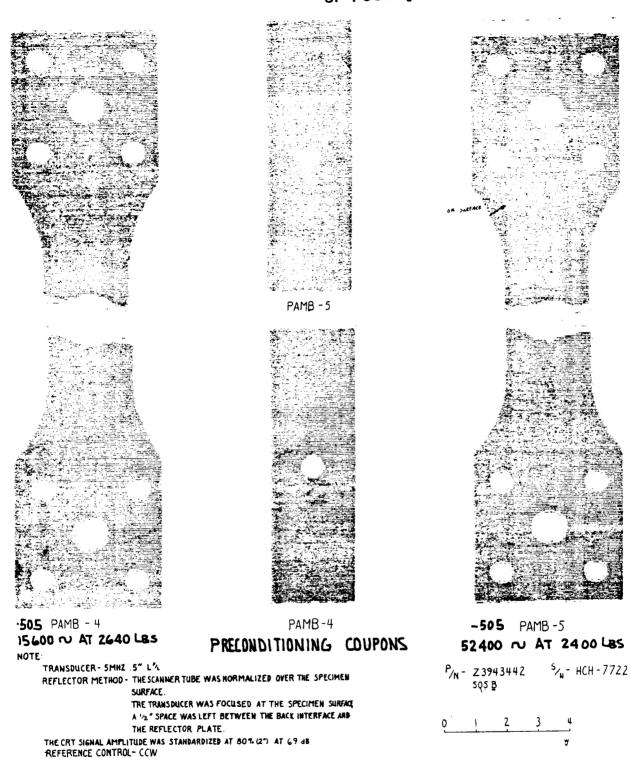
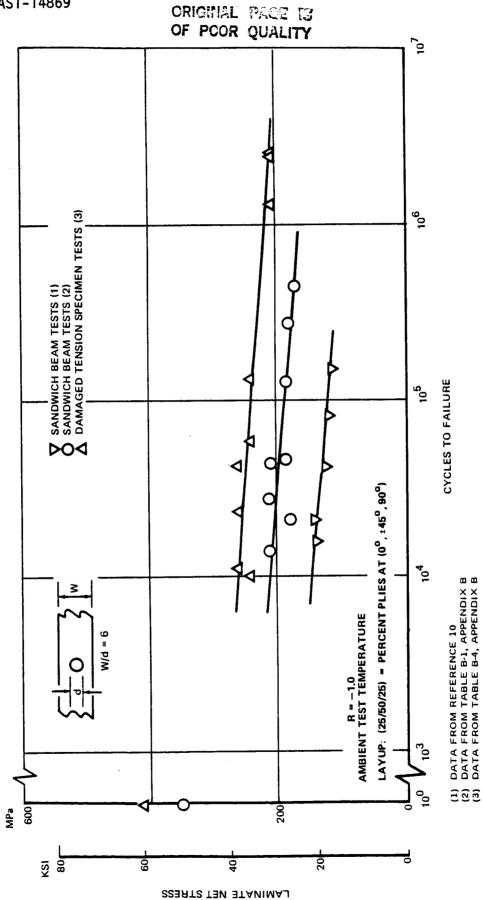


FIGURE 48. C-SCANS OF DAMAGE SPECIMENS AND MOISTURE CONTROL COUPONS

THE GATE THRESHOLD WAS SET AT 50 % (1").

FIGURE 49. FATIGUE CHARACTERISTICS OF T300/5208 GRAPHITE/EPOXY LAMINATES



79

The average measured stress concentration factor for the damaged specimens from Table B-4, Appendix B, is 1.75. From these factors, the strength reduction from the drilled hole to the damage-induced hole is 18.3 percent (1 - 1.43/1.75). The damage is apparently more severe than is indicated by the strain gage readings. This is probably due to local invisible damage to the epoxy around the edge of the damaged area.

Testing of the three Z3943443 axial load cover panels with a transverse center slit was completed. The data obtained from the tests are tabulated in Table 7. One of the specimens mounted in the MTS test machine is shown in Figure 50. The photograph also shows the environmental chamber used for testing at 350°K (170°F) or 219°K (-65°F). Figure 51 is a photograph of the ambient specimen after test indicating crack growth as a function of the number of cycles. Final rupture occurred at 10,000 cycles. A similar failure occurred in the specimen tested at 350°K at 4670 cycles. The specimen tested at 219°K did not fail after two lifetimes of load cycling. The first life cycle test was run at a load level of 10231 Newtons (2300 pounds). This load level was doubled for the second life cycle test in an attempt to induce crack propagation. No failures occurred so the specimen was tested statically to determine residual strength. Failure occurred at 194.04 MPa (28143 psi) across the net section. It is evident from these tests that crack propagation will not be a problem at low temperature (219°K) but will be a significant consideration at room and elevated temperatures. These tests indicate the need for further test and evaluation in this area.

Testing was completed on the three Z5943428-501 damaged shear panels. The data obtained from the tests are tabulated in Table 8.

The specimens were load cycled to the equivalent of two lifetimes (260,000 cycles) and then tested to failure to determine residual strength. The testing was stopped after each 130,000 load cycles and the specimens visually and sonically inspected to determine damage growth. Neither specimen exhibited

FATIGUE TEST RESULTS ON SANDWICH SKIN TENSION PANELS WITH TRANSVERSE CENTER SLIT Z3943443-1 TABLE 7

STRESS RATIO R = -1.0

Slitted speacing

										CALC	CALCULATED STRESSES	STRESSE	S		
	 	ļ			SKIN LAN	AMINAIES		TECT	•	SSUED	250	Z	. :		
	22	TEST	MOISTURE	WIDTH	ТН	THICKNESS ⁽¹⁾	VESS ⁽¹⁾	LOAD LEVEL	EVEL	SECTION	NOI	SECTION ⁽²⁾	ON ⁽²⁾	CYCLES	
SPEC NO.	y Y	e.	PERCENT	CM	ż	₩.	ż	NEWTONS POUNDS	POUNDS	MPa	PSI	MPa	<u>s</u>	FAILURE	REMARKS
P065-1	219	-65		25.418	10.007	0.1092	0.0430	10,231 20,462	2300	36.85	5,345	40.94	5,938	130,000	NO FAILURE NO FAILURE (3)
PAMB-1	AMB	AMB		25.438	10.015	0.1054	0.0415	791,7	1746 (4)	28.96	4,201 (4)	32.18	4,667	10,000	FAILED THROUGH SLIT
P170-1 350	350	170		25.420	10.008	0.1016	0.0400	6,731	1513 (4)	26.06	3,780 (4)	28.96	4,200	4,670	FAILED THROUGH SLIT

⁽¹⁾THICKNESS IS NET LAMINATE THICKNESS FOR BOTH FACES BASED ON MEASURED PANEL THICKNESS LESS 0.762 CM (0.300 IN.) FOR CORE.

 $^{(2)}$ NET SECTION STRESS BASED ON MEASURED PANEL WIDTH LESS 2.54 CM (1.00 IN.) FOR SLIT.

(3) STATIC RESIDUAL STRENGTH OF SPECIMEN WAS 48,486 NEWTONS (10,900 LB) OR 194.04 MPa (28,143 PSI) BASED ON NET SECTION. FAILURE OCCURRED THROUGH CENTER SLIT.

(4) TEST LOAD LEVELS FOR THE AMBIENT AND 350 ⁰K (170 °F) CASES WERE NOT OBTAINED BECAUSE OF EQUIPMENT CALIBRATION ERROR. LOAD LEVELS AND STRESSES SHOWN ARE BASED ON MEASURED STRAINS.

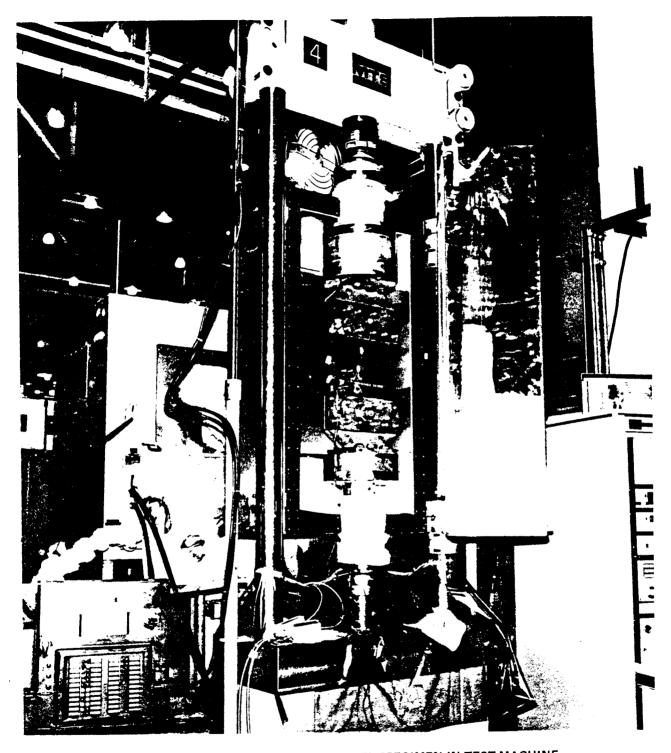


FIGURE 50. Z3943443 CENTER SLIT PANEL SPECIMEN IN TEST MACHINE

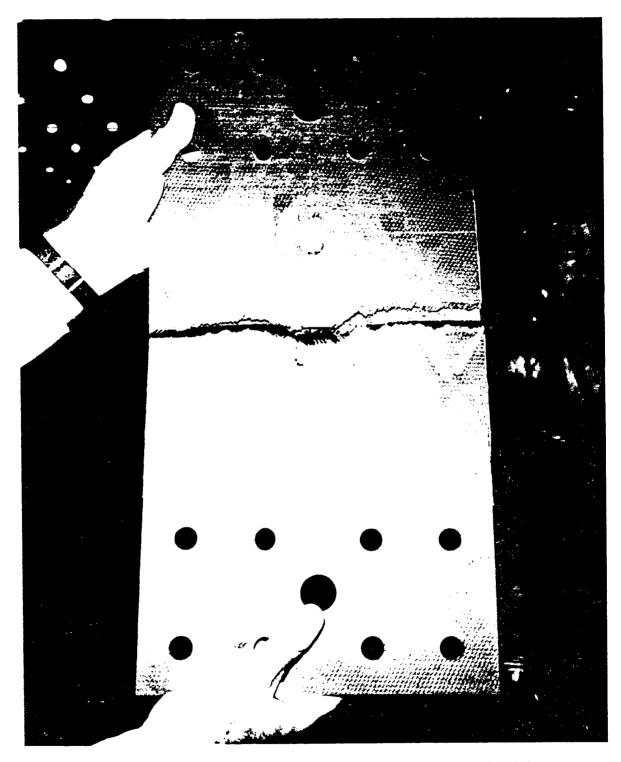
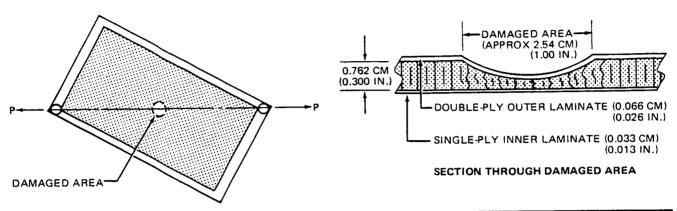


FIGURE 51. FAILURE OF Z3943443 CENTER SLIT PANEL SPECIMEN

TABLE 8

DAMAGED SHEAR PANEL TEST RESULTS
Z5943428-501



	PERCENT	TES TEMPER		RESIDUAL STRENGTH AT FAILURE p(1)		SHEAR FLOW AT FAILURE N _{XY}		SHEAR STRESS IN LAMINATE AT FAILURE		PERCENT OF DESIGN ULTIMATE
PANEL NUMBER	MOISTURE CONTENT	°к	°F	NEWTONS	POUNDS	N/M	LB/IN.	MPa	PSI	LOAD(2)
1	1.31	219	-65	131,303	29,518	200,870	1147	202.77	29,410	151
2	1.66	AMBIENT	AMBIENT	105,103	23,628	160,766	918	162.29	23,538	121
3	0.99	350	170	125,066	28,116	191,239	1092	199.05	28,000	144

⁽¹⁾ RESIDUAL STRENGTH AFTER THE EQUIVALENT OF TWO LIFETIMES OF CYCLIC LOADING AT A LOAD RATIO OF R = -1.0.

⁽²⁾ DESIGN ULTIMATE SHEAR FLOW = 132,746 N/M (758 LB/IN.)

any change or growth in size of the damaged area. The residual strengths were approximately 42 percent below the undamaged (and uncycled) panel at ambient temperature and 22 percent below the undamaged panel at 350°K. A typical failure mode for these panels is shown in Figure 52. All failures occurred through the damaged area.



FIGURE 52. FAILURE OF Z5943428-501 DAMAGED SHEAR PANEL

SECTION 6 DESIGN VERIFICATION TEST COMPONENTS

Tooling and detail part fabrication were continued on all elements of this component group. Final assembly operations were started as detail part availability permitted. The verification tests components are:

- Z5943445 Concept Verification Panels
- Z5943446 Concept Verification Spars
- Z5943452 Attach Fitting Splice Specimens

The fabrication status of these components is discussed in this section.

CONCEPT VERIFICATION PANELS

Fabrication and assembly were continued on the Z5943445 combined load test panel (-1) and the acoustic test panels (-501). Current fabrication emphasis is on parts for the -1 configuration, illustrated in Figure 53, because this activity is on the critical schedule path of the program. The -1 panel will be tested in combined compression, in-plane shear, and lateral pressure to verify the CVS skin panel design concept. Fabrication of the CVS skin panel tooling will not begin until this test is successfully completed.

Tool fabrication for the combined load test panel was completed on 6 November 1978. Some joggled regions were omitted on the sine-wave rib laminating mold and the initially cured rib-elements were rejected as a result. The mold was modified per engineering drawing requirements and replacement parts were fabricated.

Assembly of the Z5943445-1 combined loads panel is in progress as shown in Figure 54. The cured skin panel, protected by peel plies on the outer surfaces, has been fit to the metal parts which will connect it to the test fixture. The sine-wave rib and spar-web elements have been trimmed to size and fit together. Subsequent operations will cocure and bond the rib and spar-web junctions using graphite-epoxy attach angles, and mechanically fasten the skin panel to the sine-wave substructure elements. Finally, the graphite-epoxy panel will be mechanically attached to the test frame.

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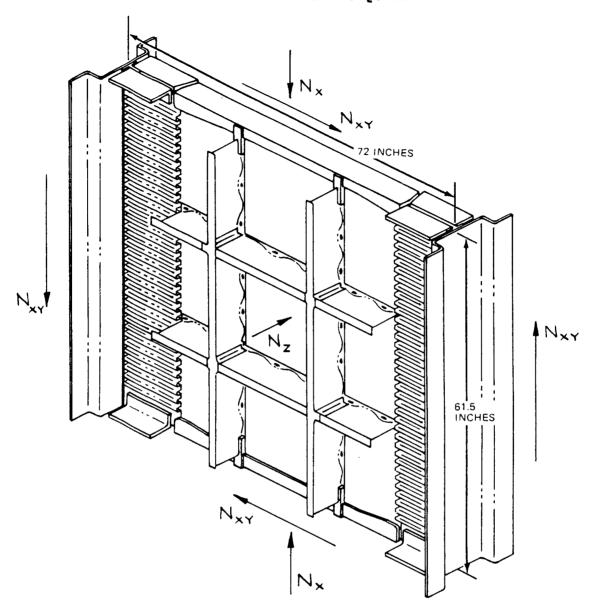


FIGURE 53. Z5943445 COVER PANEL COMBINED LOAD TEST SPECIMEN

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FIGURE 54. FINAL ASSEMBLY OF Z5943445 COMBINED LOAD TEST SPECIMEN

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Following completion of final assembly and instrumentation, the component will undergo a four-day moisture exposure period to achieve a minimum moisture content of one percent in the honeycomb sandwich skin facings. The component will then be tested under combined compression, in-plane shear, and lateral pressure. Test completion is presently projected at 23 February 1979.

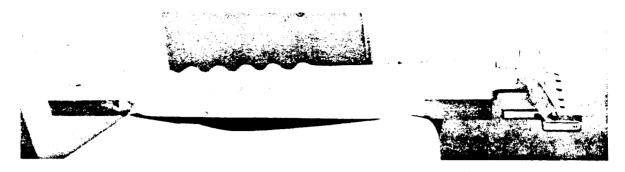
CONCEPT VERIFICATION SPARS

Tool fabrication for the Z5943446-1 concept verification spar component was essentially completed during the reporting period. The machined aluminum alloy lower half of the laminating mold together with the mold end piece is shown in Figure 55a. The upper half of the laminating mold is shown in Figure 55b. The upper mold half consists of a rigid aluminum alloy component (which in conjunction with the lower half will maintain the spar cap contours and bevels) and a cast rubber facing. Side pressure plates will also be provided to transmit autoclave pressure to the spar cap flanges during the cure cycle. This tooling concept is further illustrated in Section 7, Tool Design. Fabrication of the spar tooling for the CVS will not be started until the Z5943446 spar component is successfully tested.

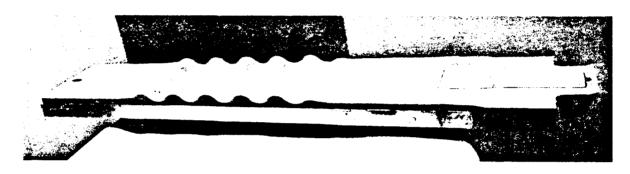
ATTACH FITTING SPLICE SPECIMENS

Fabrication of the Z5943452-1, -501, and -503 splice specimens, three each, was completed during the reporting period and the specimens were placed in a moisture chamber to start a 30-day exposure period. The completed -1 and -501 specimens are shown in Figure 56. The specimens represent the bonded/ bolted splice joint at the root-end of a CVS spar between the laminates and the titanium alloy fitting as shown in Figure 4.

The thicker -l specimens, Figure 57, represent the joint interface between the thick laminated spar-web and the titanium alloy fitting. The thinner -501 specimen, Figure 58, represents a double-lap scarf-joint at the skin flange of the spar in the same region. The -503 specimen represents a single-lap scarf-joint at the skin flange of the spar. This single-lap scarf-joint configuration is being used in the CVS spar drawings to simplify the spar layup and processing.



(a) LOWER MOLD HALF AND END-PIECE



(b) UPPER MOLD HALF

FIGURE 55. Z5943446 SPAR COMPONENT LAMINATING MOLD

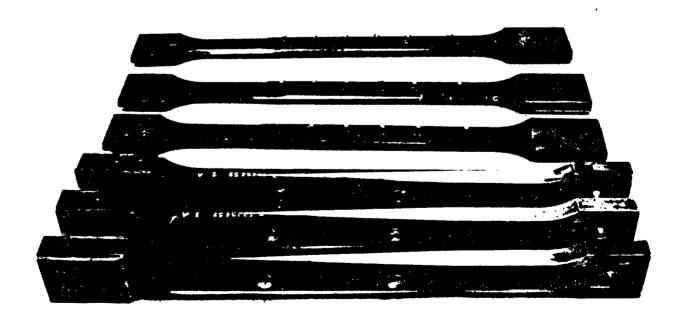


FIGURE 56. Z5943452-1 AND -501 SPECIMEN

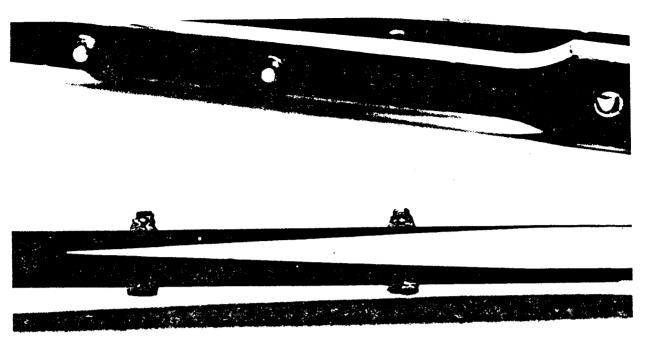


FIGURE 57. Z5943452-1 SPAR WEB SPLICE

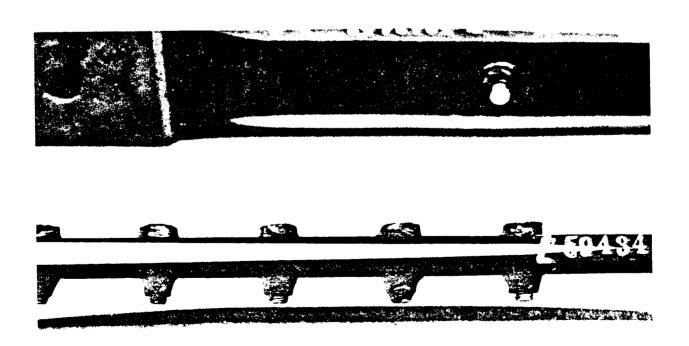


FIGURE 58. Z5943452-501 SPAR SKIN-FLANGE SPLICE

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Resin and void content samples indicated good laminate quality in all specimens as shown in the following table.

Specimen Number	Nominal Thickness- Layers (inches)	Sample Number	Resin Content Weight %	Void Content Volume %
Z5943452 - 1	60 (0.780)	1 2 Average	25.49 26.77 26.13	1.70 1.30 1.50
Z5943452 - 501	28 (0.364)	l 2 Average	28.02 28.41 28.21	0.90 0.90 0.90
Z5943452 - 503	34 (0.422)	1 2 Average	26.34 27.13 26.74	0.90 0.57 0.74
DPS Requirement	all		27 ± 5	2.00 max

Computer runs have been made using an infinite series solution to Fick's Second Law of Diffusion to show moisture distribution in different laminate thicknesses for various exposure times. Average moisture content vs. thickness is plotted in Figure 59 for various times of exposure to 170°F and 100 percent relative humidity. The plot indicates that an average moisture content over one percent will be achieved in laminates up to a thickness of about 0.35 inches during the 30-day exposure period. Additional studies indicate that discernible moisture will not penetrate to the center of the thicker section (-1) in the lifetime of the DC-10 aircraft.

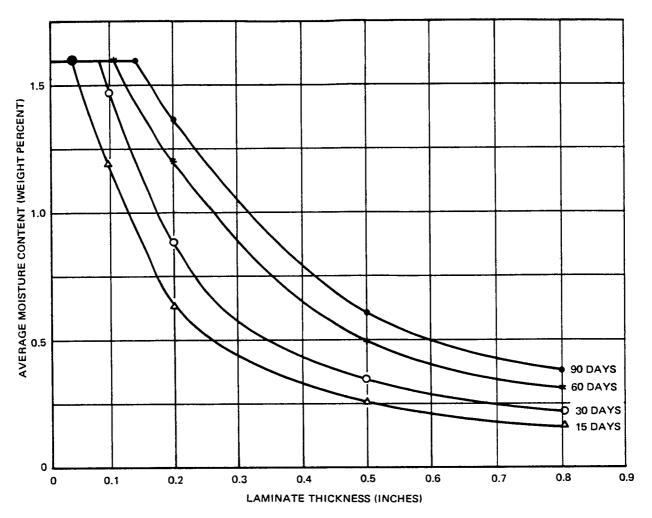


FIGURE 59. AVERAGE MOISTURE CONTENT VERSUS LAMINATE THICKNESS FOR VARIOUS TIMES OF EXPOSURE TO 170°F AND 100 PERCENT RELATIVE HUMIDITY

SECTION 7 TOOL DESIGN

Tool design activities are proceeding on the released engineering drawings. With the exception of the station 295 rib tools, fabrication of hard tooling will not be started until appropriate joint development or concept verification tests are completed. The station 295 rib tools will be fabricated directly on completion of the tool design. One set of station 295 rib parts for the box-beam verification component will be fabricated to prove the rib tooling design concept.

SKIN FABRICATION TOOLING

The skin panel tool design is in progress based on the requirements of drawing number AMC7840. The tool concept is illustrated in Figure 60. The outer surface of the skin panel will be the tooled surface to facilitate net molding of the recesses for access doors, antenna bays, and leading and trailing edge attachments. The basic mold surface will be a 1/4 inch thick steel plate rolled to contour. The plate will be stud-welded at the back surface and bolted to an egg-crate supporting structure. The supporting structure will be designed to facilitate air circulation and rapid heat-up in the autoclave. Honeycomb locators which index to the mold surface will be used to facilitate placement of the core segments during the skin panel buildup. A caul plate will be used during the cure cycle to maintain a smooth inner surface on the cured part.

SPAR FABRICATION TOOLING

The design of spar tools is in progress based on the requirements of drawing numbers AMC7845, AMC7847, AMC7848, AMC7849, and AMC7893. The spar tool concept is illustrated in Figure 61. A machined aluminum alloy tool for the upper surface will be used. The titanium fittings of the spar assembly will be held in position by locating bolts through the end piece of the tool. The lower half of the laminating mold will consist of a cast rubber mold

FIGURE 60. TOOLING CONCEPT FOR COMPOSITE VERTICAL STABILIZER SKIN PANELS

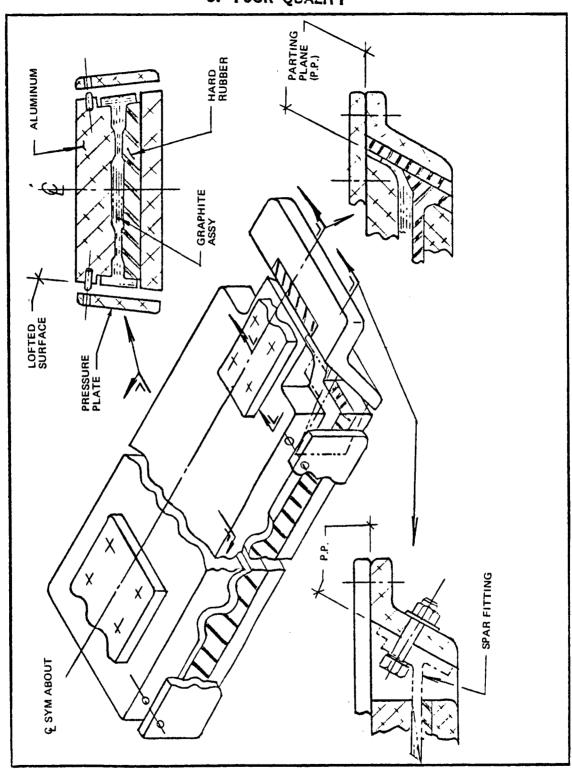


FIGURE 61. TOOLING CONCEPT FOR COMPOSITE VERTICAL STABILIZER SPARS

backed up by a rigid aluminum alloy plate. The rubber mold will be Teflon coated to preclude laminate sticking after a cure cycle. The side pressure plates will maintain the lofted bevels and contours of the spar cap flanges which interface with the skin panels.

RIB FABRICATION TOOLING

The design of the base rib tool is in progress based on the requirements of drawing number AMC7853. The tooling concept is illustrated in Figure 62. An aluminum female mold will be used to control the outer loft surface of the part. A fiberglass caul plate will be used to control the internal surfaces to insure a good fit with the spar ends. A split base rib was designed to provide a tolerance take-up feature at a centerline overlap splice joint.

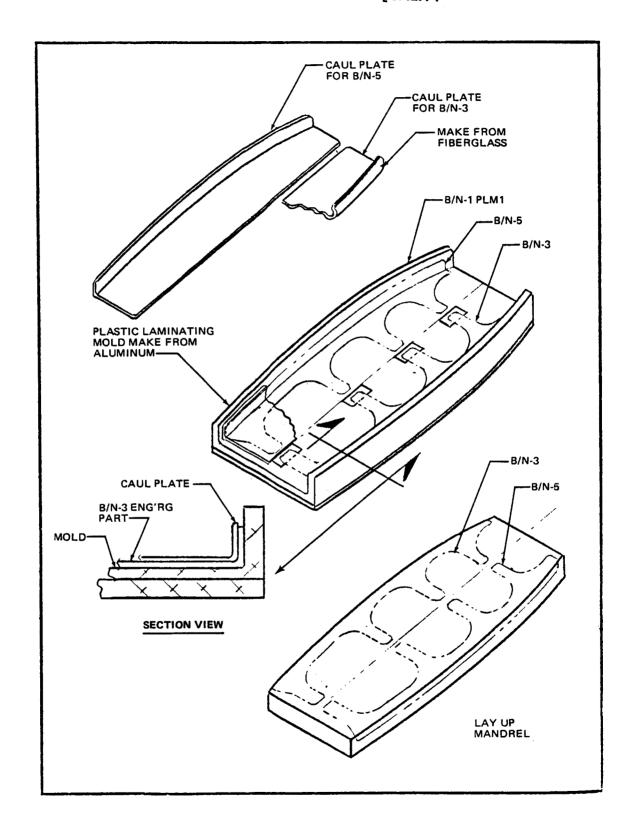


FIGURE 62. TOOLING CONCEPT FOR COMPOSITE VERTICAL STABILIZER BASE RIB

SECTION 8 COST ANALYSIS

An economic analysis was completed using the technical and manufacturing information developed for the current structural configuration (e.g., mechanically attached skin panels, sine-wave stiffened shear panels). The assumptions and analysis guidelines reported previously (Reference 9) were applied in this current analysis. Experience and information obtained in the composite upperaft rudder program (Reference 6) were also applied.

The economic analysis results are expressed in terms of the number of composite stabilizers to be manufactured in a production mode to achieve cost parity with the current unit costs of the metal stabilizer. This cost cross-over point was estimated at 100 composite stabilizer units at the start of the program. The current analysis indicates that the cost cross-over point will be achieved after approximately 32 units are produced (see Figure 63).

New labor estimates were made to reflect the current design concept. The estimates associated with the economic analysis were selectively extracted from the overall program labor estimates. Development costs were excluded. The manufacturing data shown in Table 9 represents estimates of the eight stabilizers incorporating the present design concept. These estimates will be updated as actual cost data are accrued. In Table 9, the manufacturing labor hours and the planning hours were derived from the current estimates. The recurring tooling was allocated arbitrarily to the eight units as sustaining tooling. Sustaining Engineering and Inspection/NDT were assumed to be the same as shown in the prior analysis, Reference 9. Manufacturing T1 labor hours (recurring labor hours for the first production unit) derived from the base estimate are shown in Table 10 together with production progress (learning) curve assumptions used in the cost cross-over analysis.

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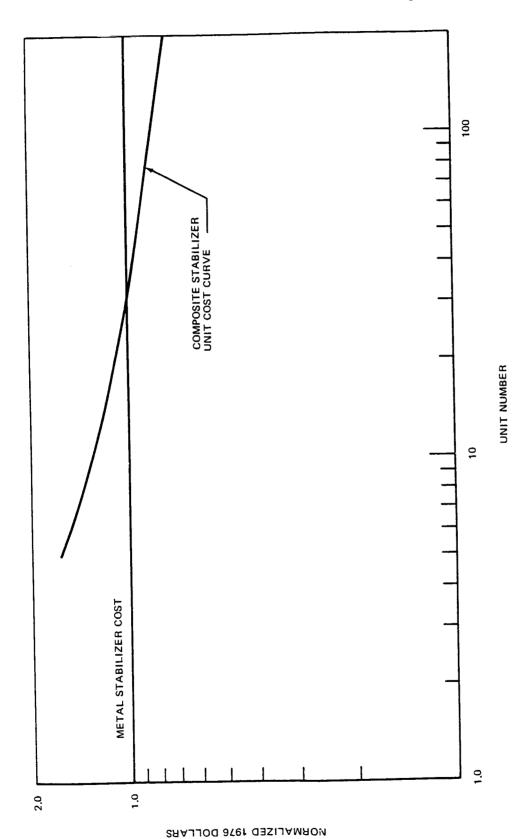


FIGURE 63. PROJECTED COST CROSS-OVER POINT FOR DC-10 COMPOSITE VERTICAL STABILIZERS

Douglas Aircraft Company Contract NAS1-14869

TABLE 9
RECURRING LABOR HOURS FOR DC-10 COMPOSITE VERTICAL STABILIZERS

MANUFACTURING FUNCTION	NO. 1	NO. 2	NO. 3	NO. 4	NO. 5	NO. 6	NO. 7	NO. 8	TOTAL
COMPOSITE STRUCTURE	11,153	9,050	10,073	9,348	8,968	8,692	8,475	8,296	74,055
LE/TIP/RUDDER	(102)	(99)	133	133	133	132	132	132	795
METAL DETAILS	(387)	(383)	512	512	511	511	511	511	3,068
STRUCTURE ASSEMBLY	286	286	286	286	286	286	286	285	2,287
BOX ASSEMBLY	2,377	2,377	2,377	2,377	2,377	2,377	2,377	2,377	19,016
FINAL ASSEMBLY	1,164	1,164	1,164	1,164	1,164	1,164	1,164	1,164	9,312
PLANNING	542	462	_499	473	459	449	440	432	3,756
INSPECTION/NDT	1,770	1,600	1,410	1,300	1,250	1,220	1,190	1,170	10,910
TOOLING	700	700	700	700	700	699	699	699	5,597
ENGINEERING	260	250	240	230	220	190	150	140	1,680
TOTAL	18,252	15,889	17,394	16,523	16,068	15,720	15,424	15,206	130,476

TABLE 10
PROJECTED T1 LABOR HOURS FOR
DC-10 COMPOSITE VERTICAL STABILIZER

MANUFACTURING FUNCTION	T1 AVERAGE DERIVED FROM UNITS 1 THROUGH 8	ESTIMATING CURVE AND FACTORS
COMPOSITE STRUCTURE	9257	80/84%*
LE/TIP/RUDDER	133	90%
METAL DETAILS	511	90%
STRUCTURE ASSEMBLY	286	80%
BOX ASSEMBLY	2377	80%
FINAL ASSEMBLY	1164	80%
PLANNING	FACTORED	6.7%
INSPECTION/NDT	FACTORED	13.6%
TOOLING	8	к
ENGINEERING	126	91%

^{*}ASSUMES AN 80 PERCENT LEARNING FROM T1 TO T100 AND AN 84 PERCENT FROM T101 TO T200

SECTION 9 QUALITY ASSURANCE

Non-destructive inspections (NDI) were performed on the Z5943434-501 sine-wave shear web component and the Z5943445-21 combined load test panel using ultrasonic, radiographic, and Fokker Bondtester NDI methods for the various types of construction (e.g., flat laminates, sine-wave laminates, and honeycomb sandwich regions).

The Z5943434-501 sine-wave shear web component was evaluated using an ultrasonic C-scan reflector technique to test the flat areas of the web, an ultrasonic C-scan pulse-echo technique to evaluate the solid laminate caps, and a pulse-echo digital thickness gage to contact scan the convoluted web areas. These ultrasonic tests indicated the test component to be of acceptable quality.

The Z5943445-21 combined load test panel was evaluated using X-radiography to view core quality and ultrasonic C-scan to detect discontinuities in the solid laminate and the skin-to-core bond lines. The solid laminate areas that could not be reached by the thru-transmission fixture were evaluated using an ultrasonic pulse-echo thickness gage. Skin-to-core bond areas were evaluated using the Fokker Bondtester. The core closures and skin-to-core bonds were judged to be of acceptable quality by the X-ray and Fokker Bondtester inspections.

Interpretation of the ultrasonic inspections was difficult because of the presence of peel plies on both surfaces of the panel and the lack of applicable inspection standards. The peel plies will not be removed until hole preparation is complete during final assembly of the panel and test fixturing. The panel was therefore examined ultrasonically for consistent homogeniety in both the solid laminate and honeycomb core regions. The inspection indicated several areas of porosity on the solid laminate. These regions were sufficiently porous to attenuate digital thickness gage readouts.

Resin and void samples will be obtained from these regions after the panel tests are completed to help define QC acceptance levels in the CVS parts. These test results are also being considered during planning of the necessary CVS inspection standards.

Incoming quality assurance tests were conducted on 23.7 kilograms (52.2 pounds) of bi-woven material. The material met all specification requirements as shown in Table 11.

TABLE 11
PREPREG QUALITY CONTROL RECEIVING INSPECTION RESULTS

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DATE REC'D	10.30.78	COLANTITY REC'D.		52.2	CCN		,	/ DATE REC'D		OUA	QUANTITY REC'D	ام	NS				
		PREP	PREPREG PROPERTIES	TIES							LAMIN	LAMINATE PROPERTIES	TIES				
UNNIT	CONTENT	VOLATLE	GEL THE	FIBER AREAL WT.		TENSULE STRENGTH OF PSI	TENSILE MODULUS	SULLOOM HESSION STREET	SULTON SULTON SULTON	FLEXURAL STRENGTH 10° PSI	PLEXURAL MODILUS 10° PSI	SHEAR STRENGTH 10 PSI	RESIN CONTENT WT. %	VOID CONTENT VOL. %	THICOGESS PER PLY MILS	FIBER	COMMENTS
DAIS REQUIRENENTS	39-45	2.0 MAX	17.28	345-385						125.0	10.0	9.0	N/A	2 MAX	N/A	N/A	
AVERAGE RESULTS																	
UNIT	41.0		30.6	367.8						175.9	12.8	12.6	28.1	8.0	13.0	94.6	
₩ 7 '	40.9	0.0		366.5						175.5	13.3	13.1			13.1		
AV6.	⊥.		-	367.2						176.1	13.0	13.1			13.1		
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SECTION 10 REFERENCES

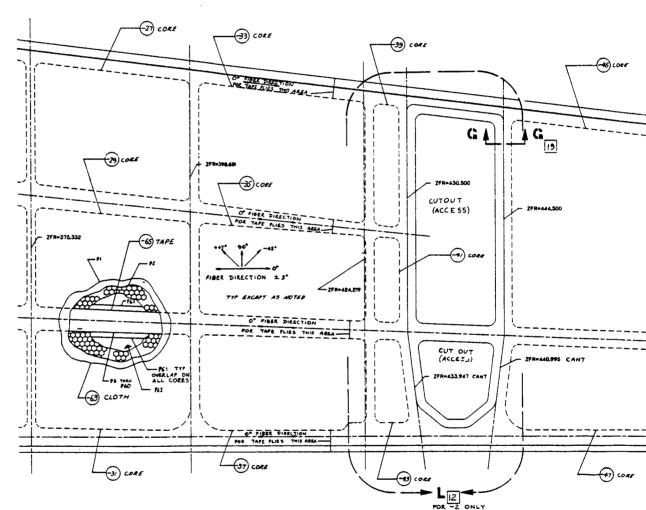
- 1. Grubb, D. W., "DC-10 Composite Vertical Stabilizer Design Criteria and External Loads," McDonnell Douglas Corporation Report Number MDC J7718, September 1977.
- 2. Abelkis, P. R., "DC-10 Aft Section Fatigue Test, Volume I, Spectrum Derivation," McDonnell Douglas Corporation Report Number DAC 67725, 23 July 1970.
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- 5. "Advanced Composite Vertical Stabilizer for DC-10 Transport Aircraft,"
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- 6. "Advanced Composite Vertical Stabilizer for DC-10 Transport Aircraft,"
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- 8. "Advanced Composite Vertical Stabilizer for DC-10 Transport Aircraft,"
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 Douglas Corporation, Report Number ACEE-03-PR-8484, 26 July 1978.
- 9. "Advanced Composite Vertical Stabilizer for DC-10 Transport Aircraft,"
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- 10. Lehman, G. M., et al, "Advanced Composite Rudders for DC-10 Aircraft Design, Manufacturing, and Ground Tests," NASA CR-145068, April 1976.
- 11. Hart-Smith, L. J., "Bolted Joints in Graphite-Epoxy Composites," Report NASA CR-144899, Contract NAS1-13172, January 1977.

APPENDIX A

ENGINEERING DRAWINGS



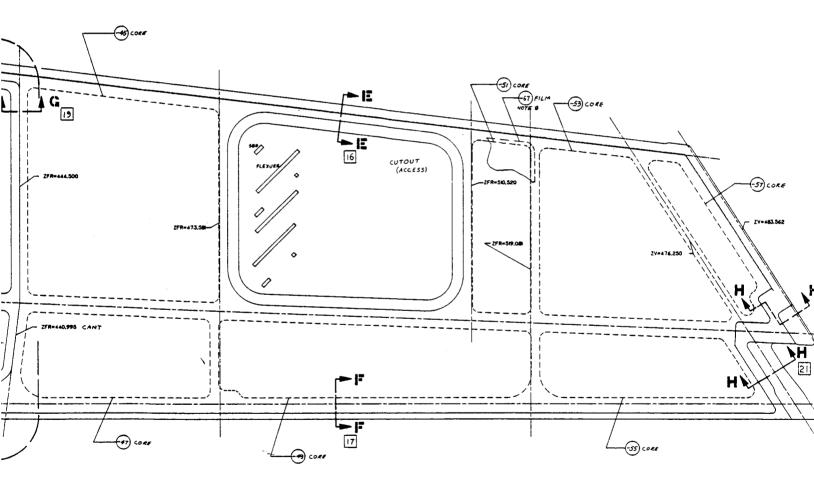
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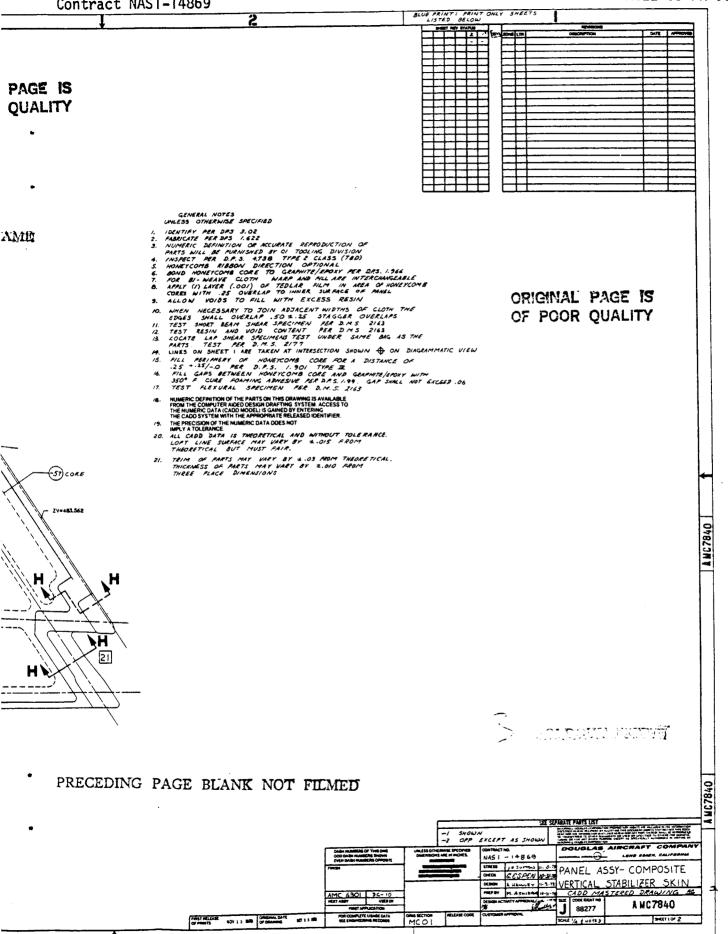
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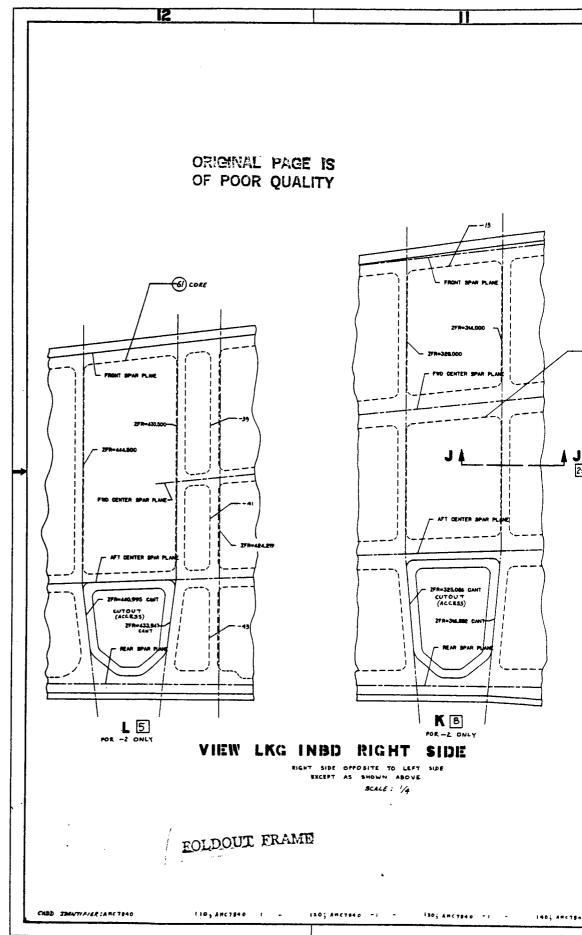
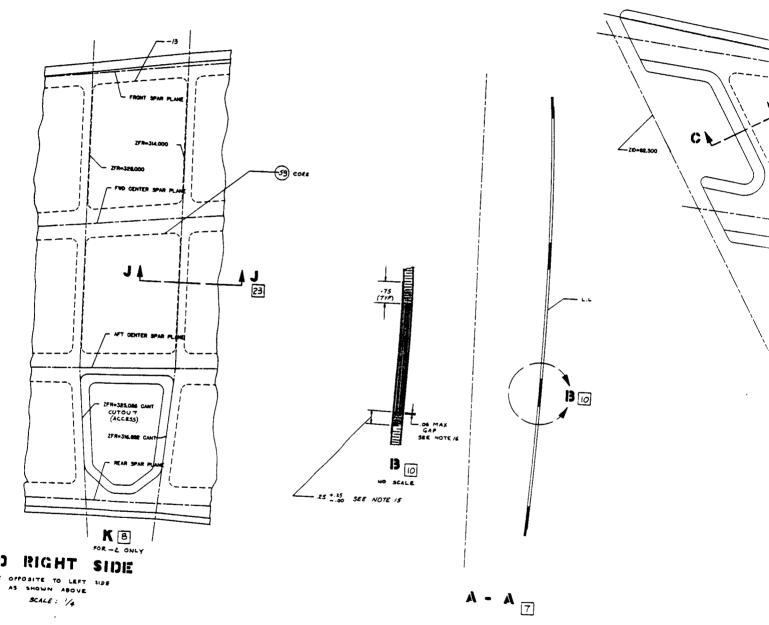


FIGURE A1. DRAWING AMC7840 - SKIN PANEL ASSEMBLY (SHEET 2 OF 4)

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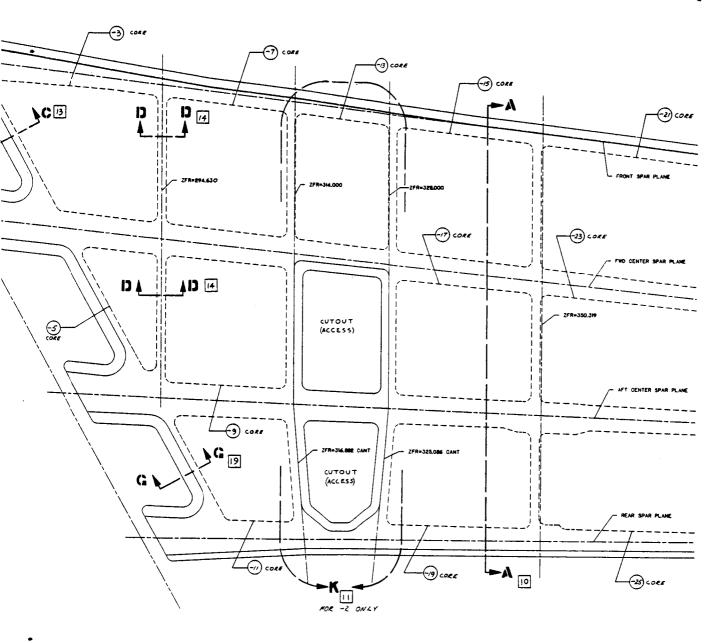
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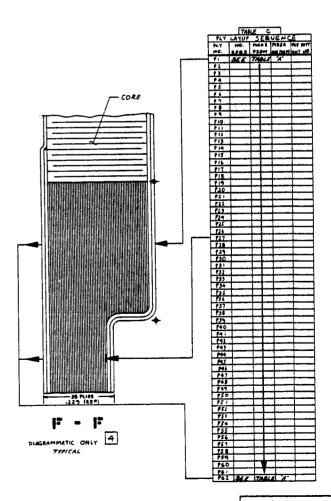
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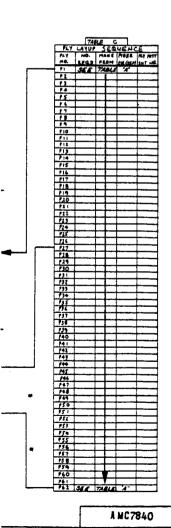
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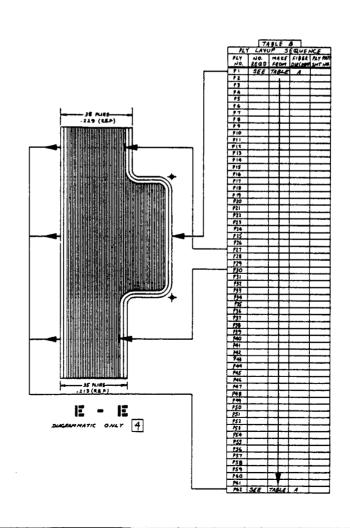


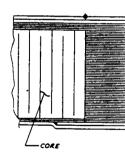
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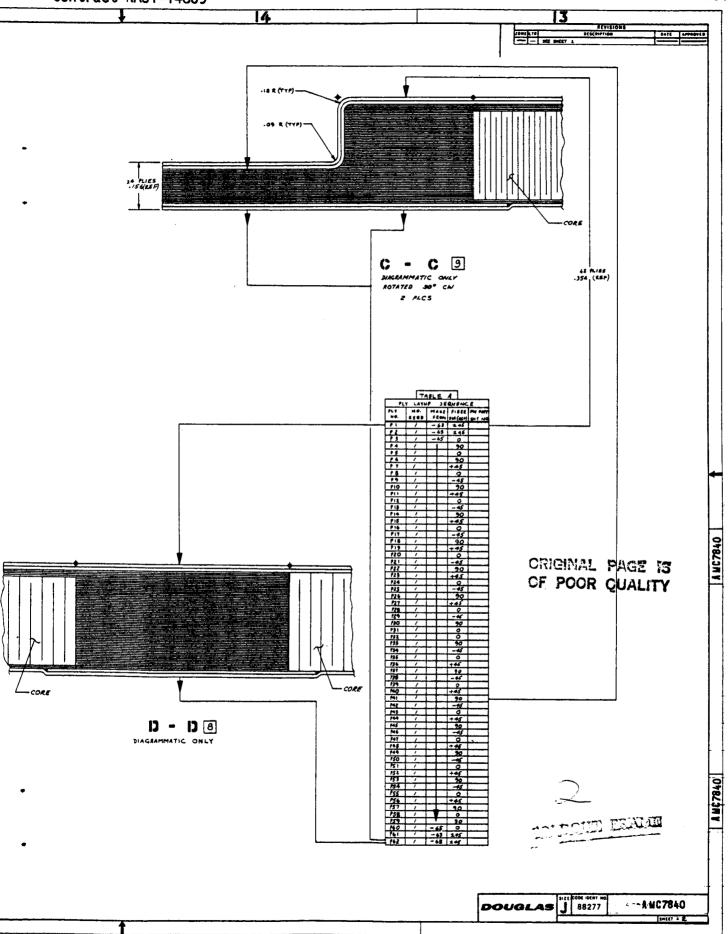


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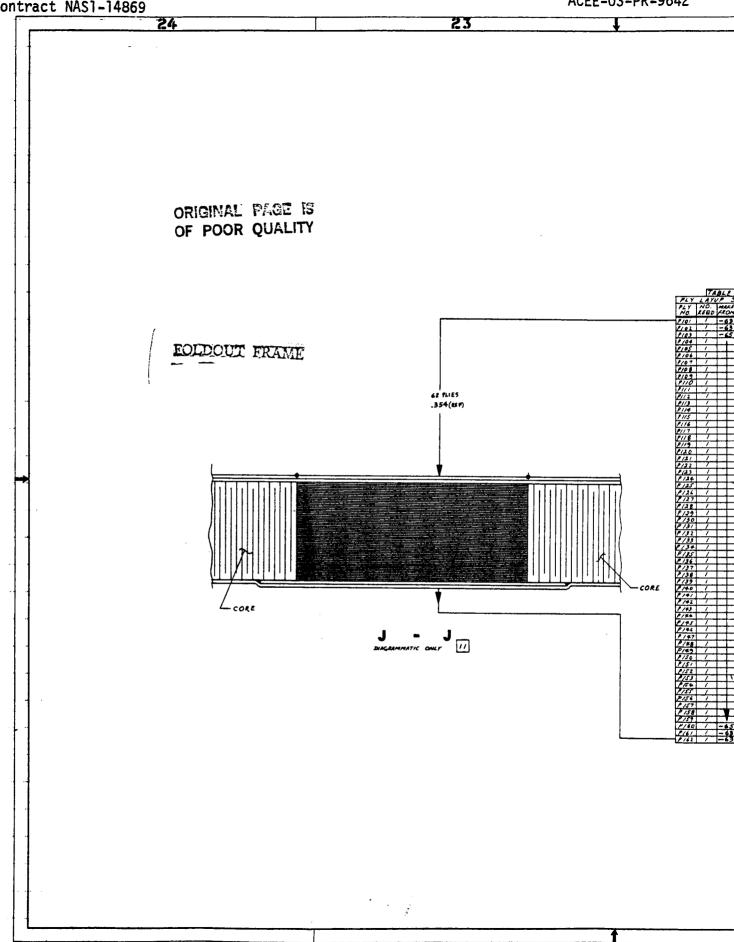
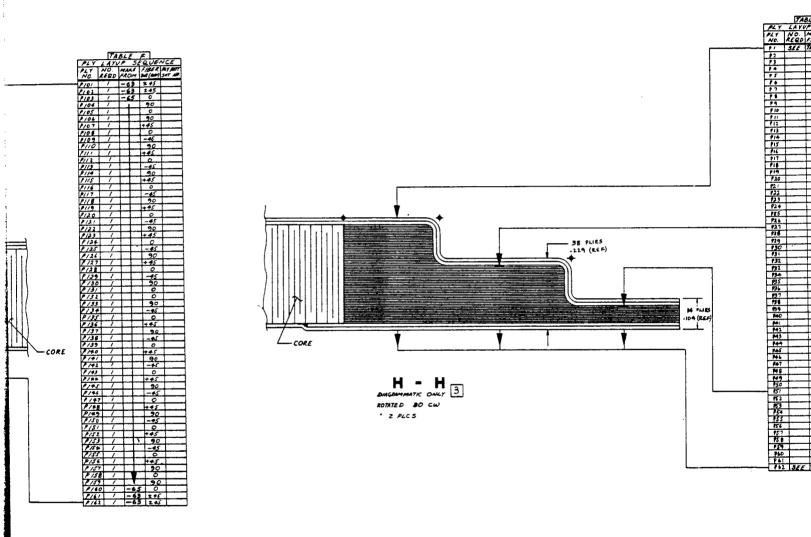


FIGURE A1. DRAWING AMC7840 - SKIN PANEL ASSEMBLY (SHEET 4 OF 4)

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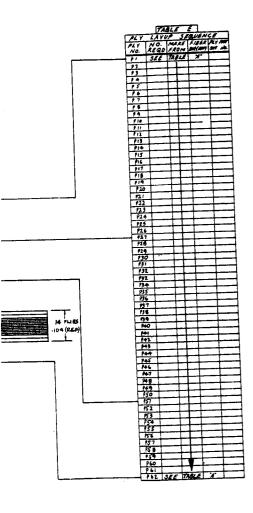


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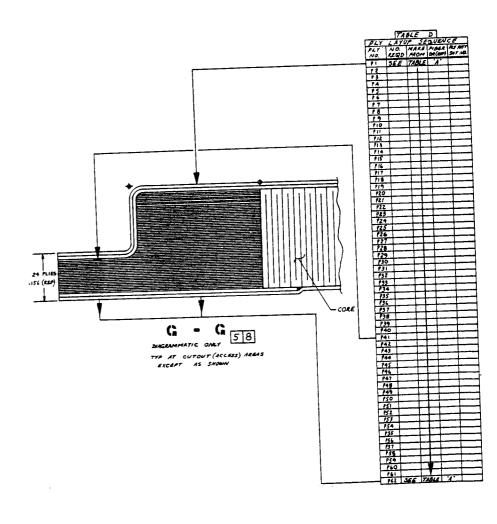
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AMC 7859-1 RIB INSTL AT Zm . 350.319

AMC 7859-1 RIB INSTL AT Zm . 528.000

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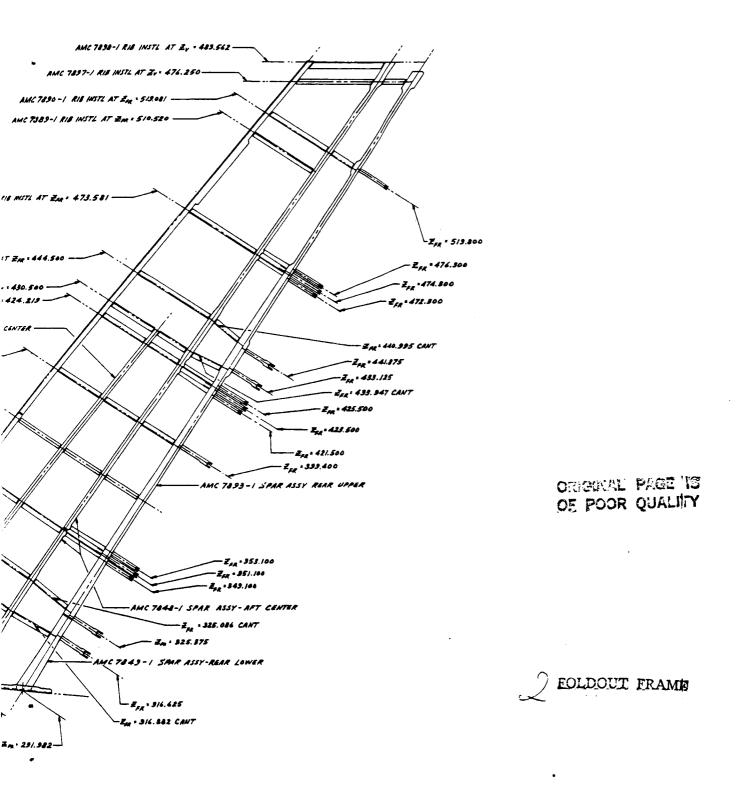
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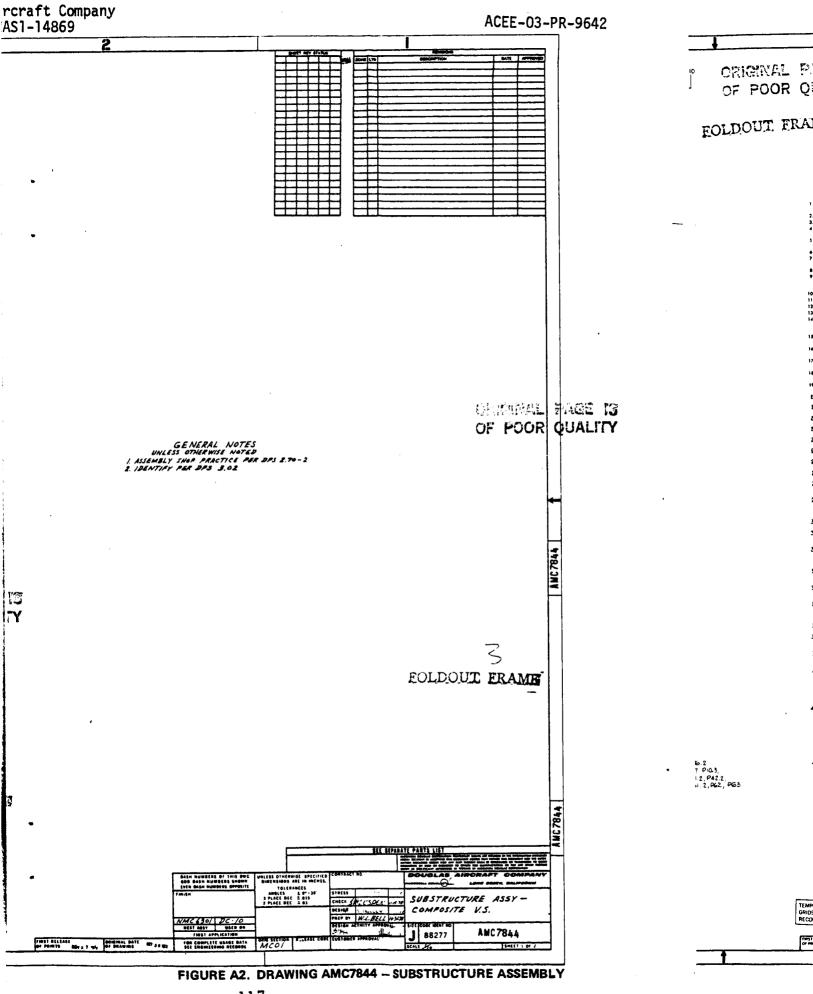
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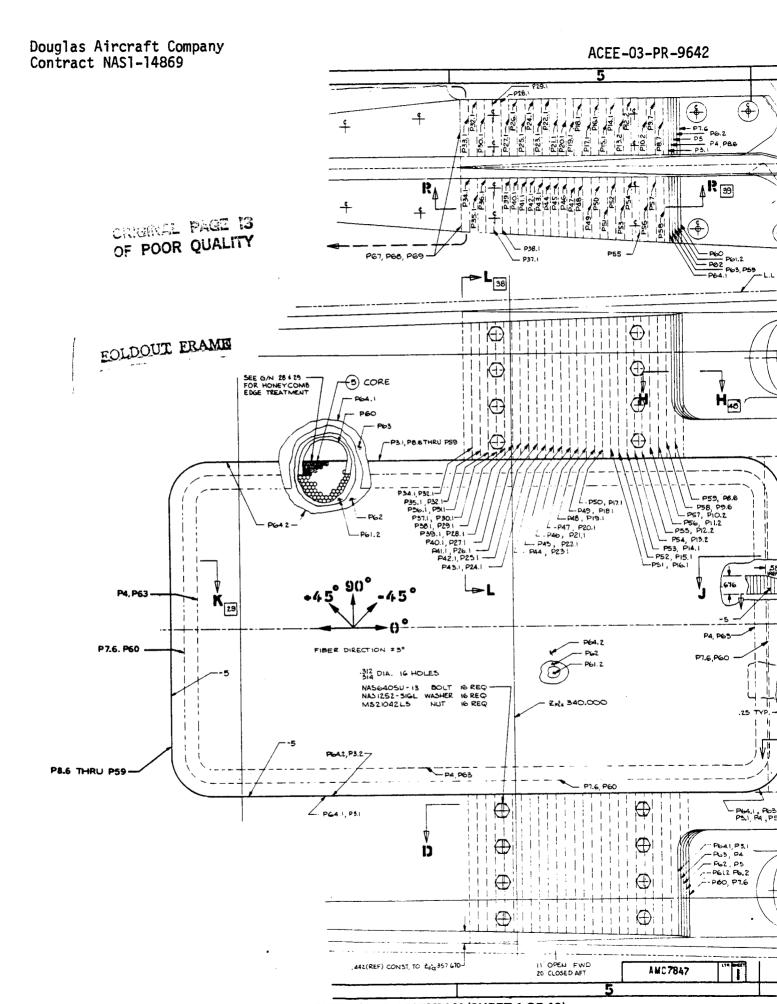
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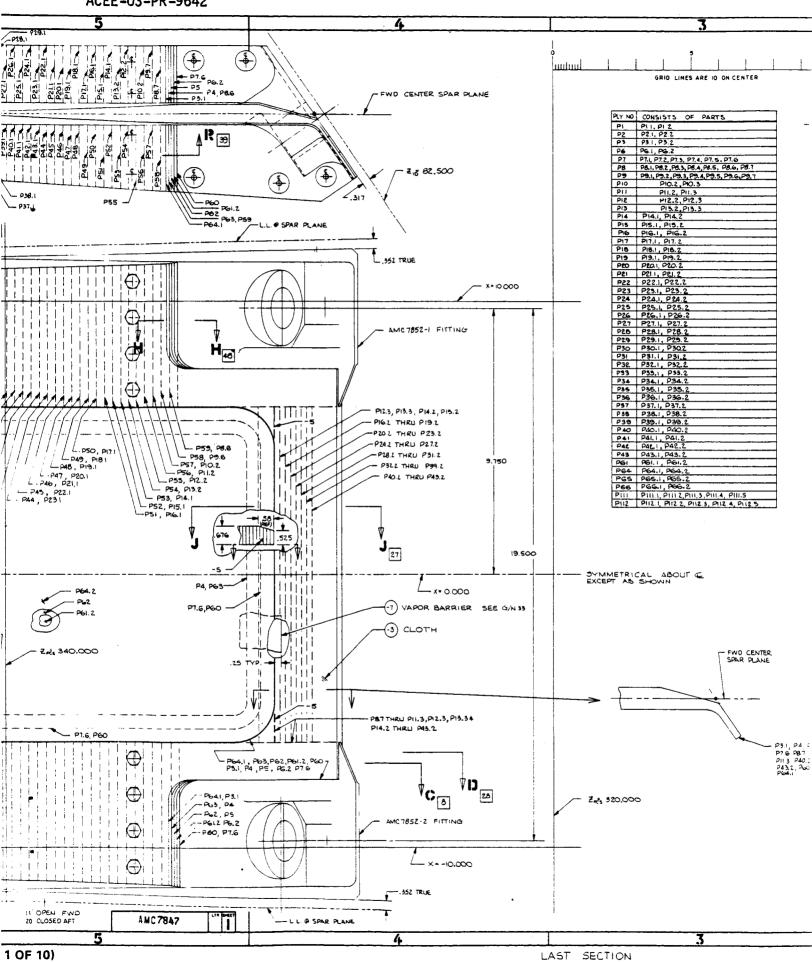
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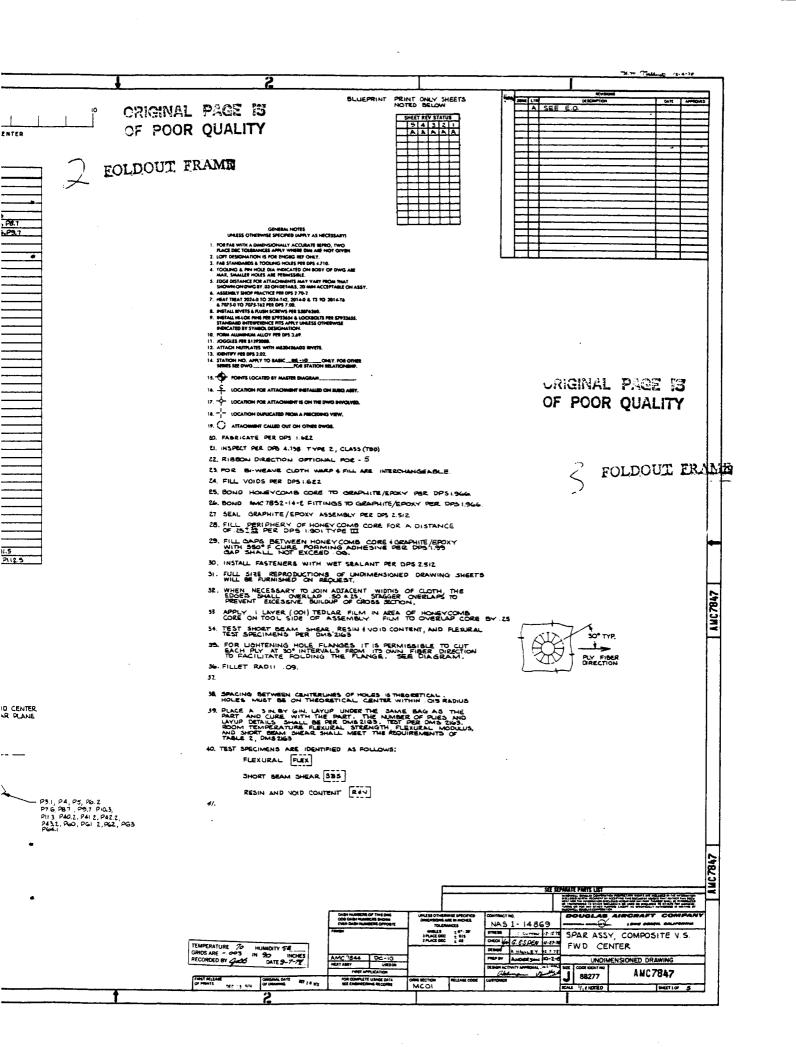


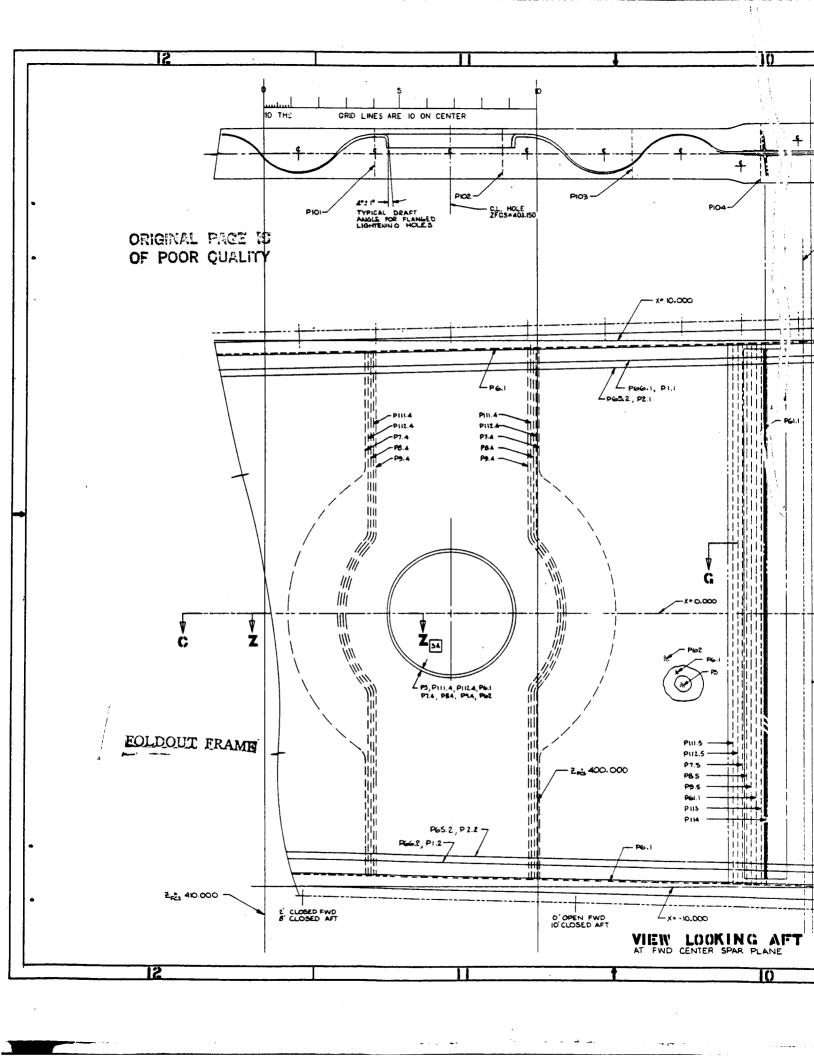
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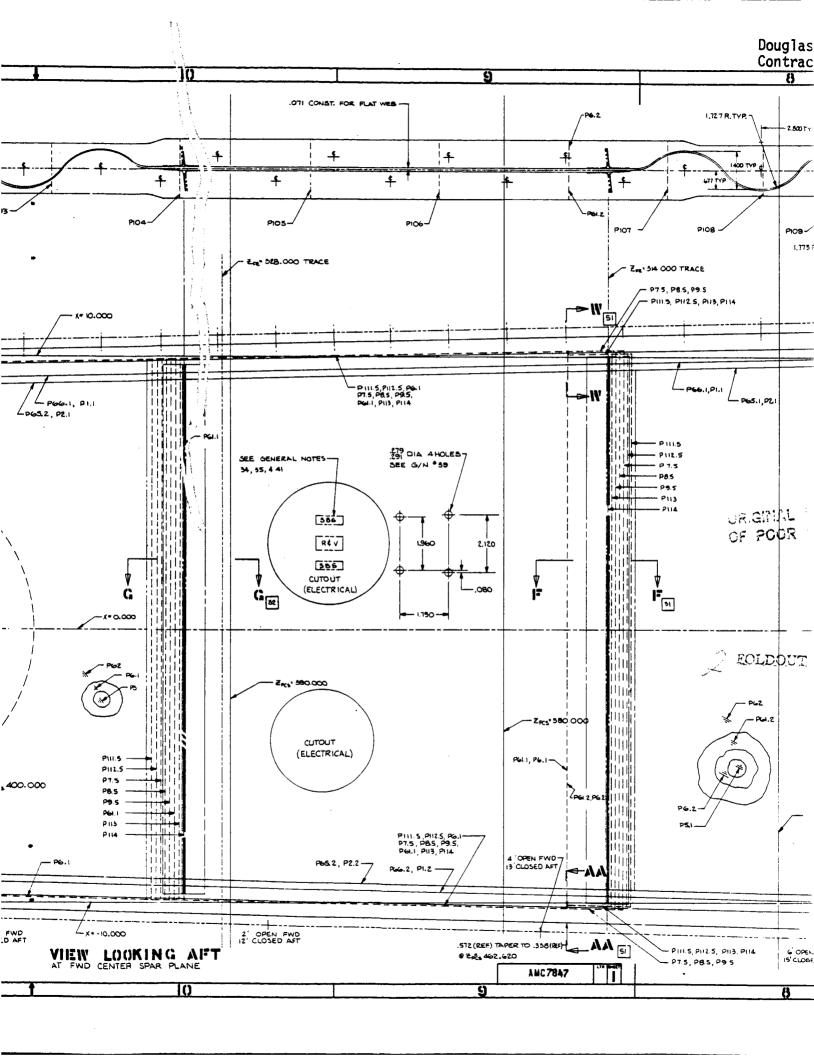












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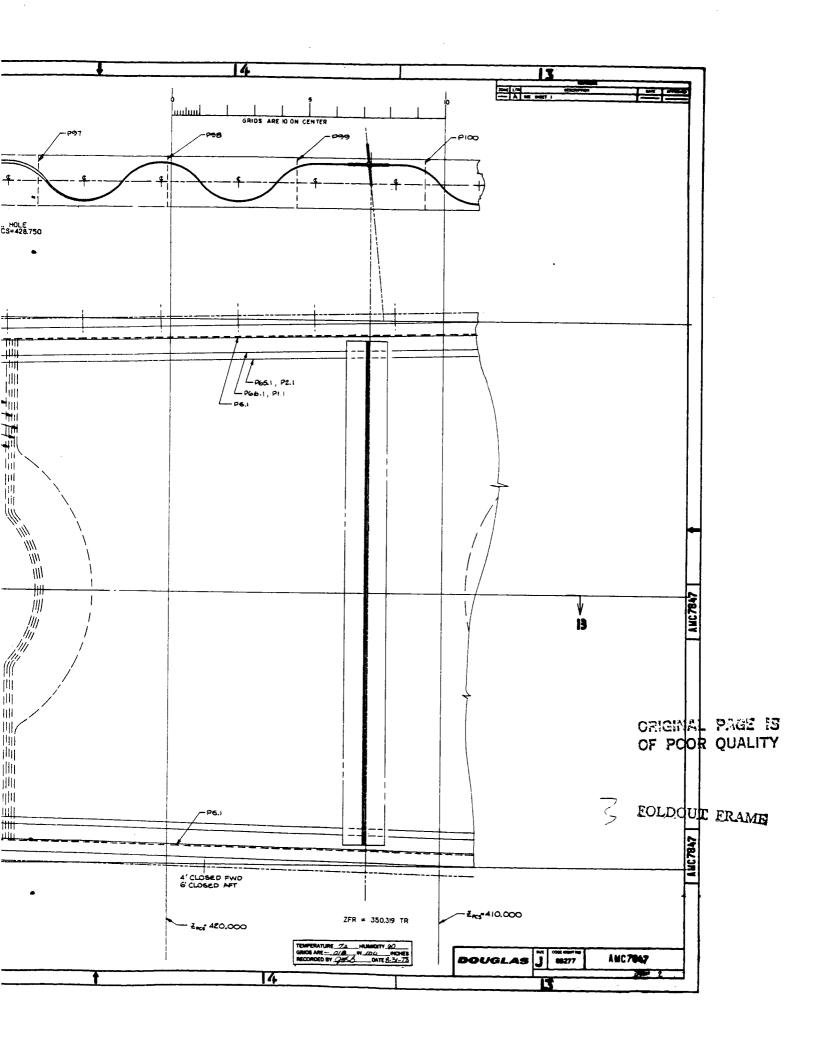
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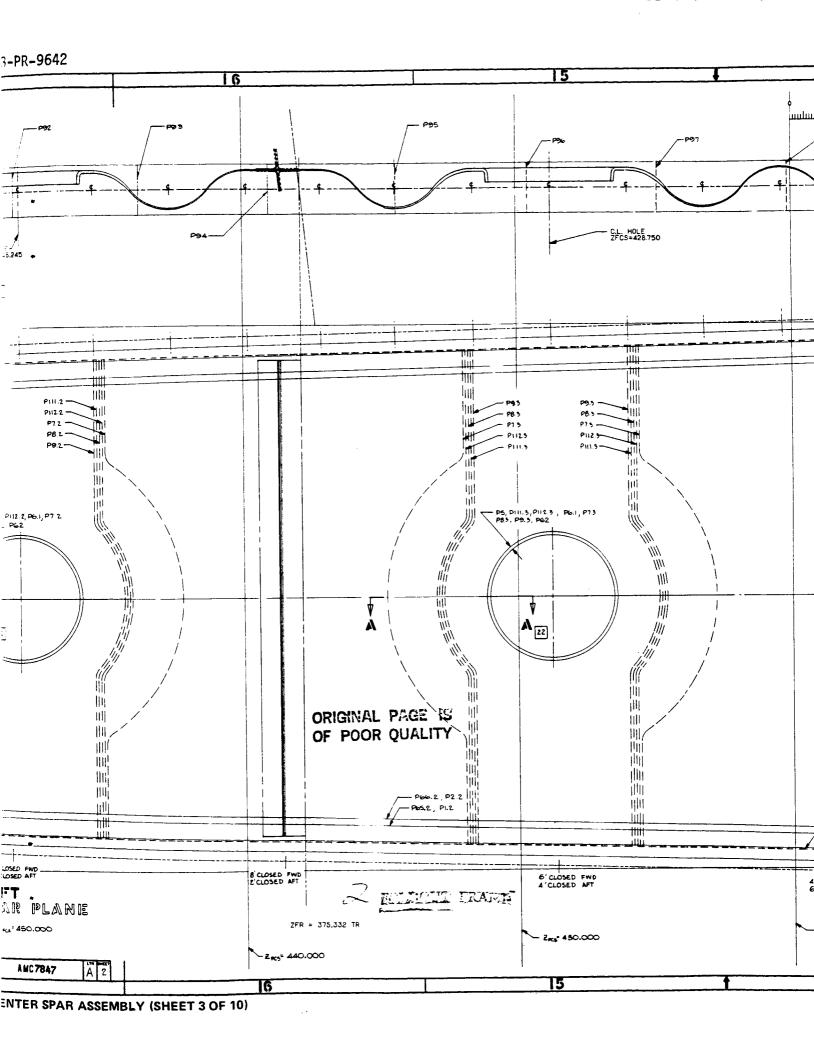
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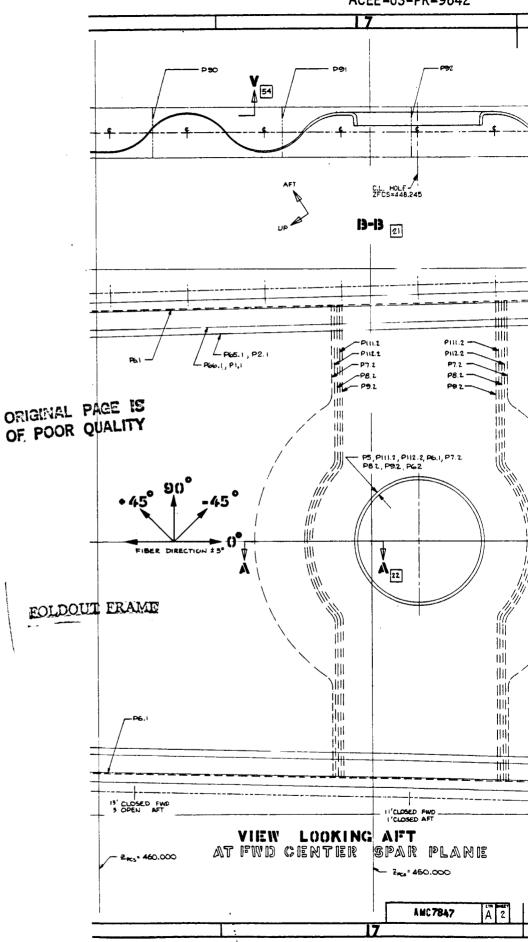
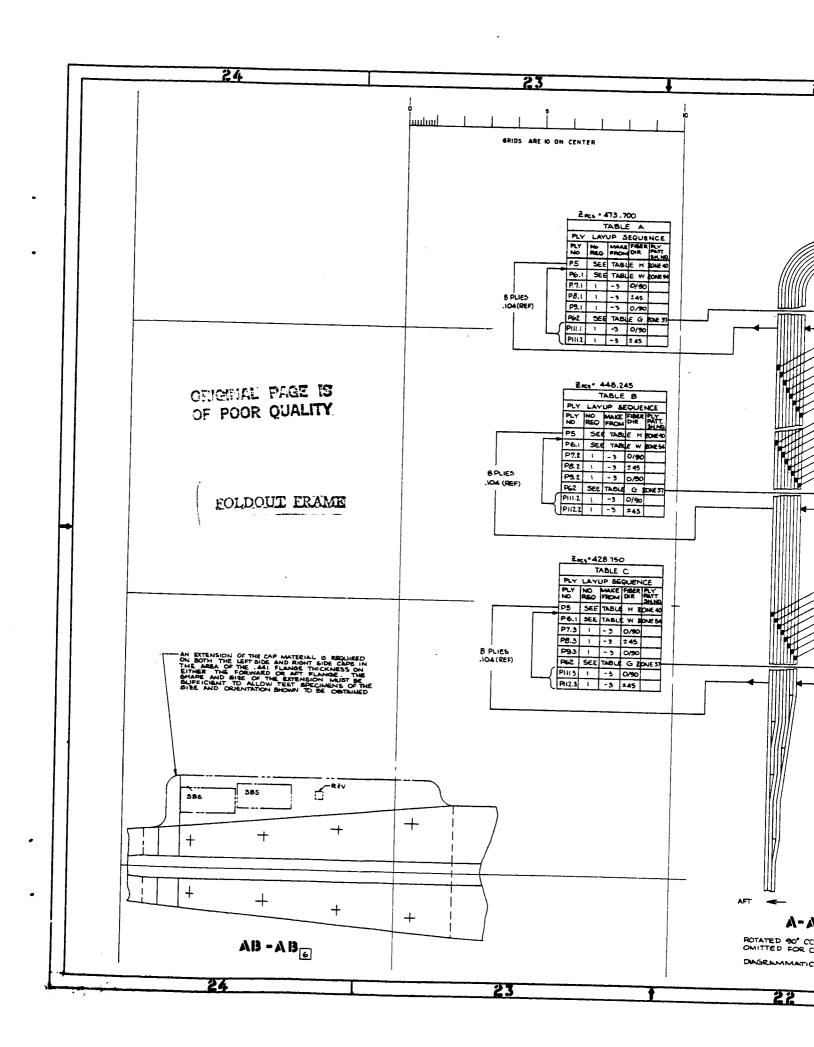
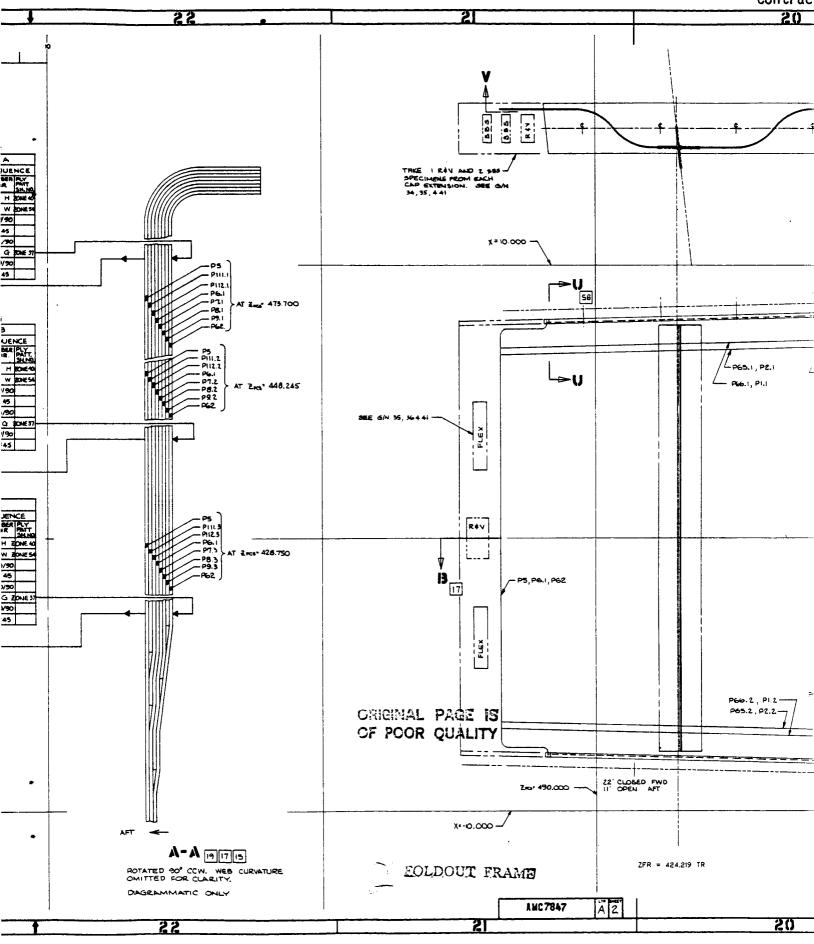
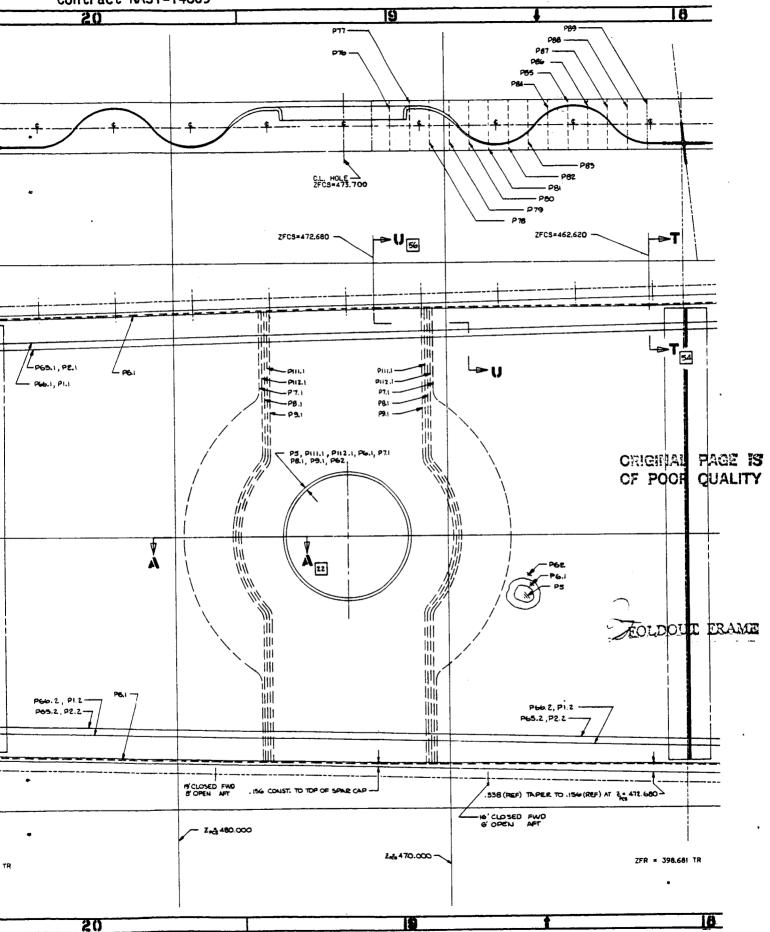


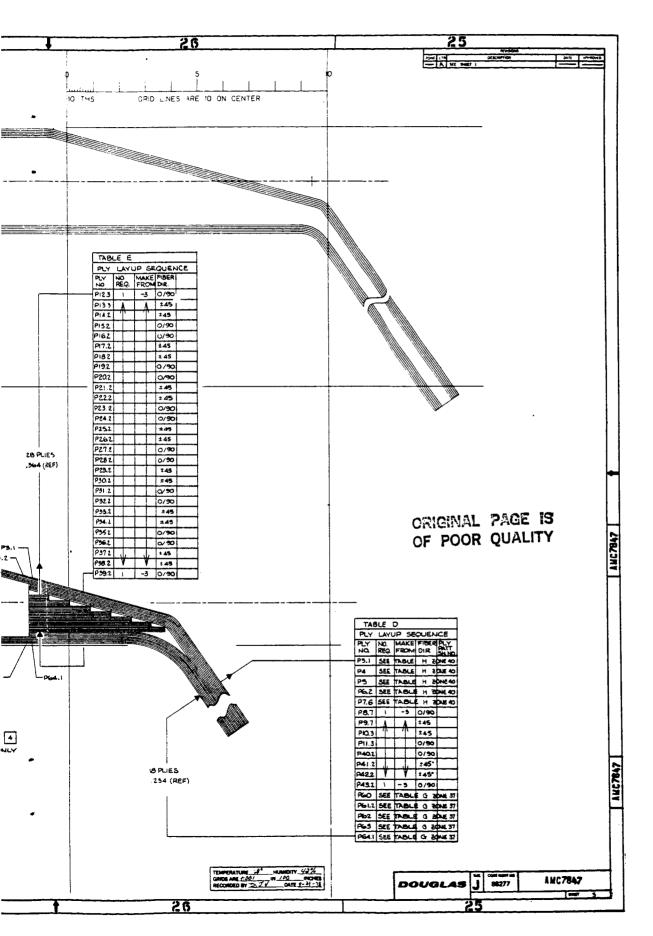
FIGURE A3. DRAWING AMC7847 — FORWARD CENTER SPAR ASSEMB







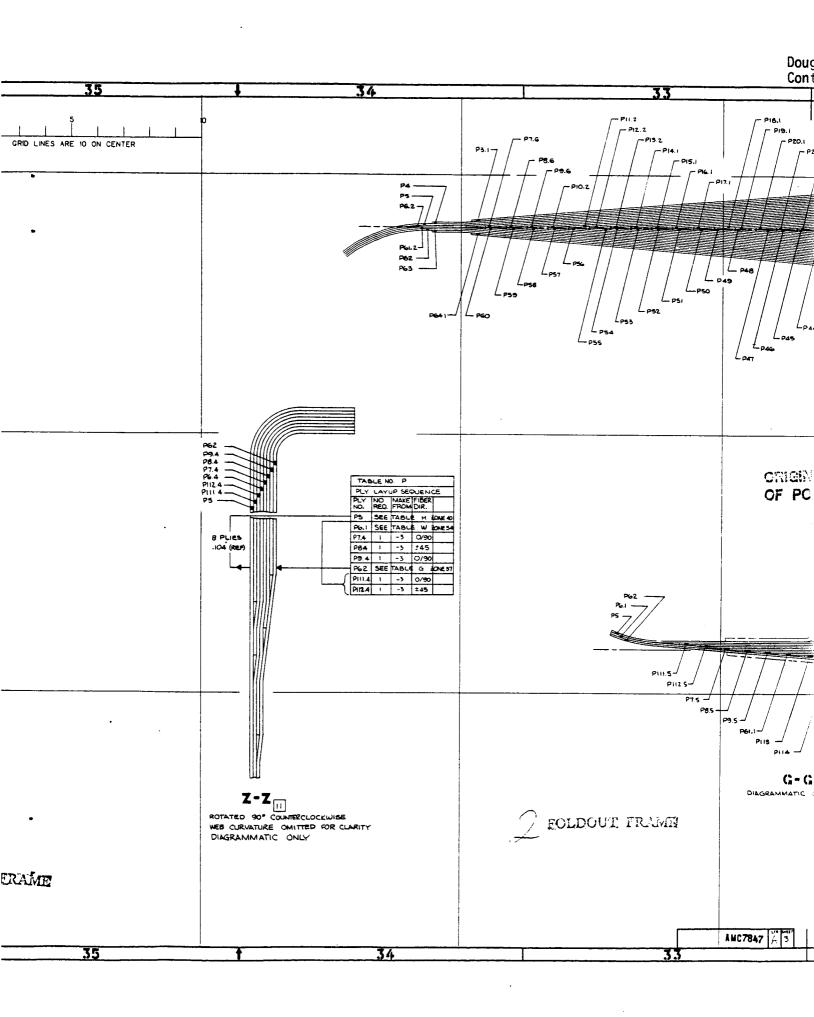
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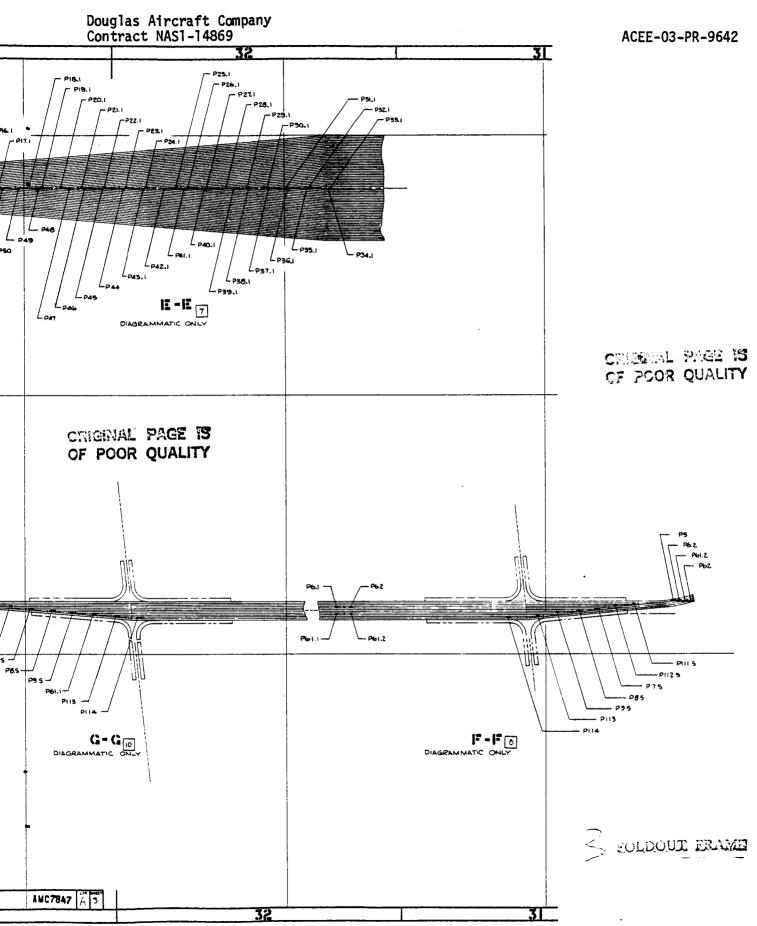


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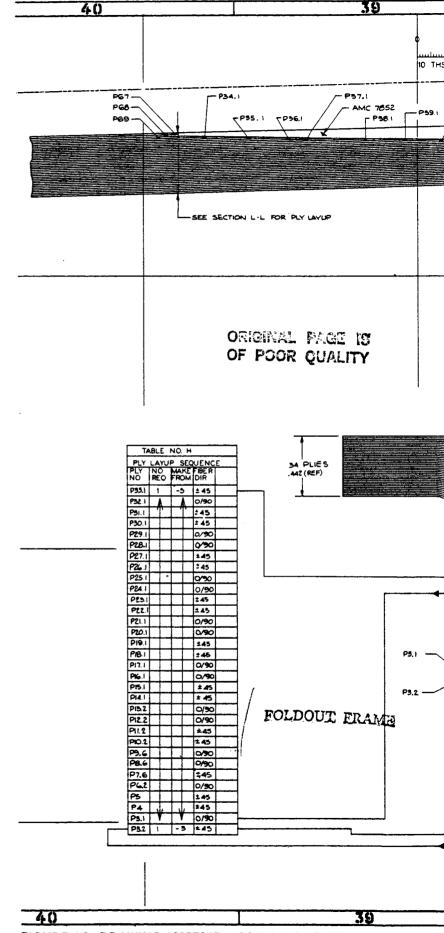
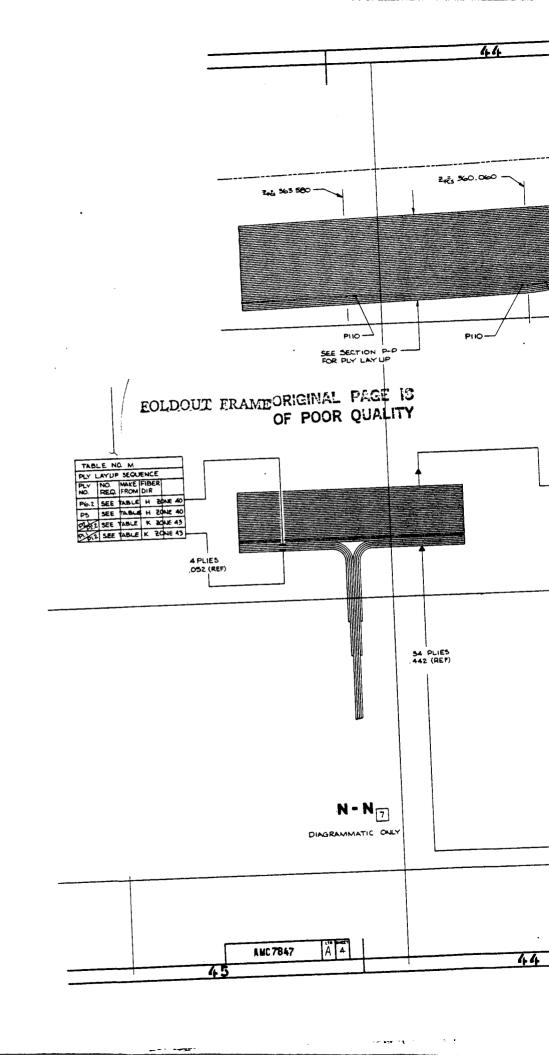
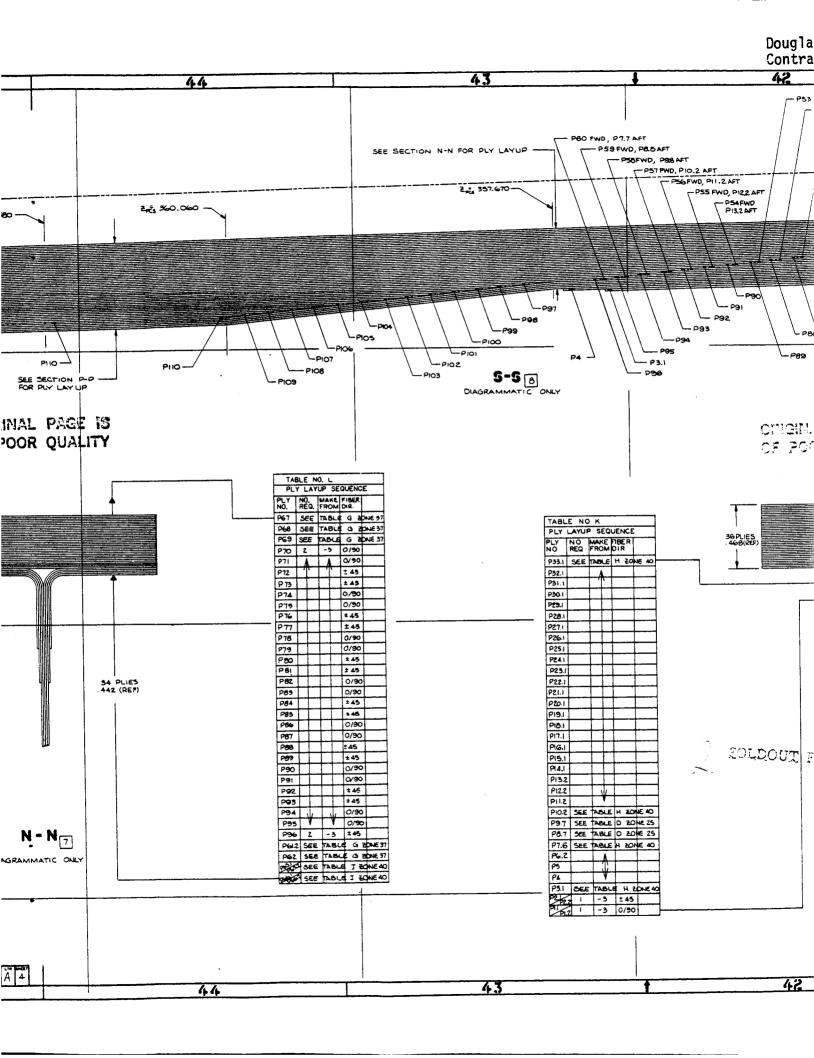
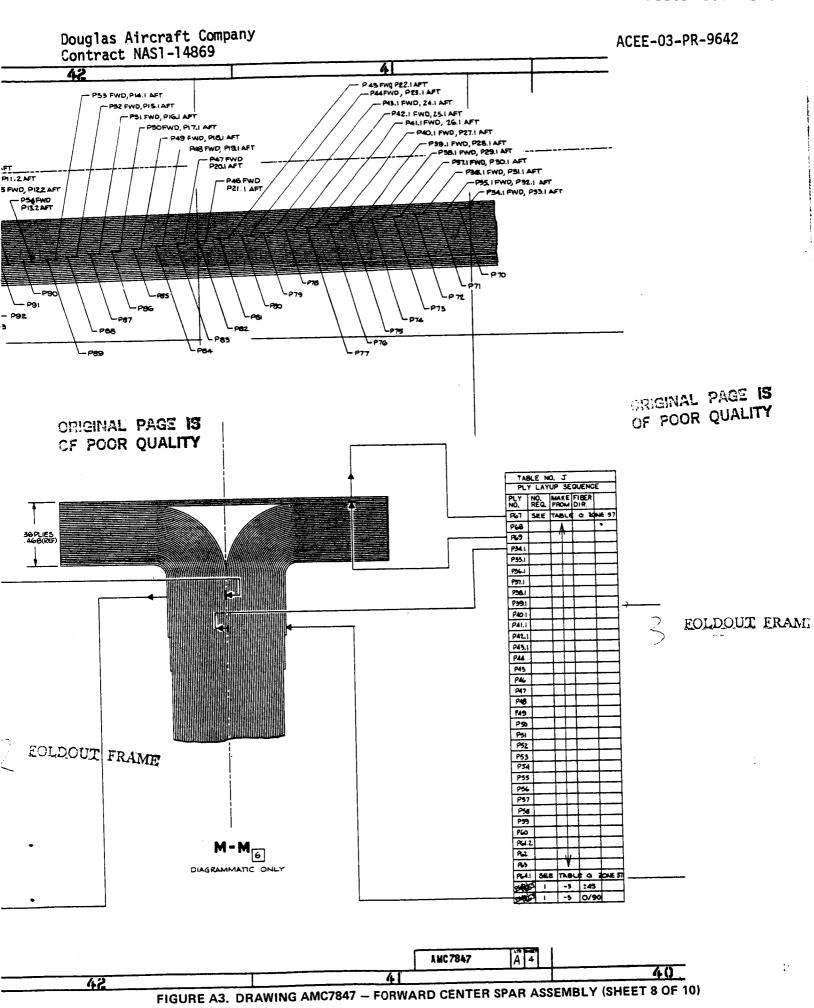


FIGURE A3. DRAWING AMC7847 — FORWARD CENTER SPAR ASSEME

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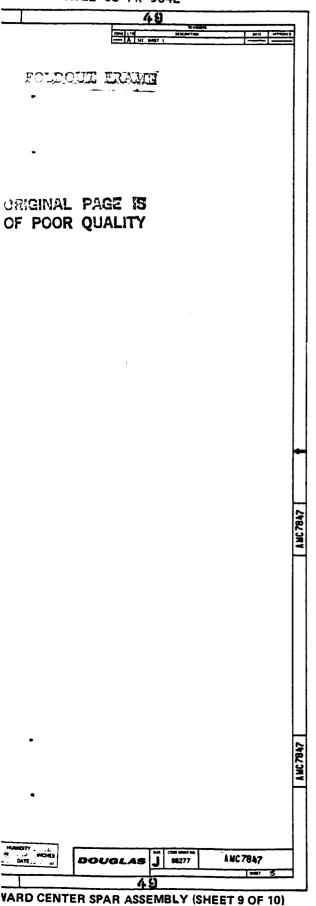
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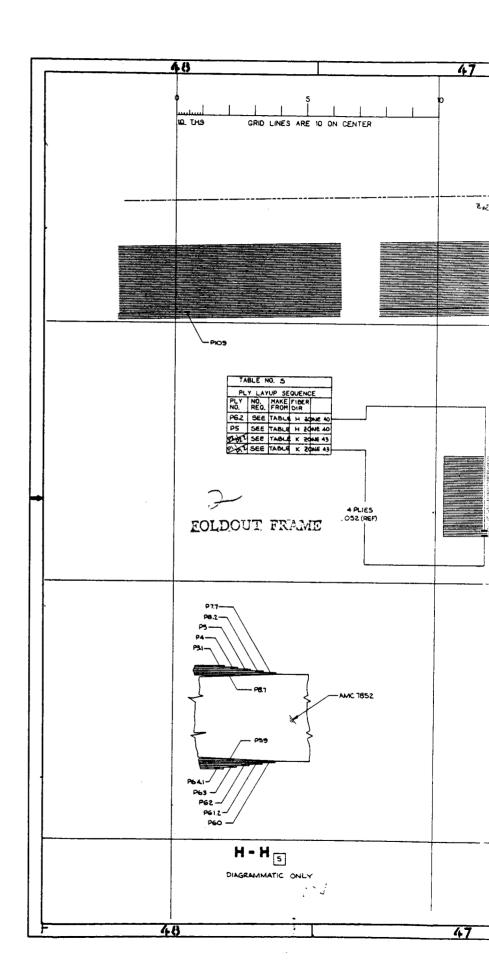
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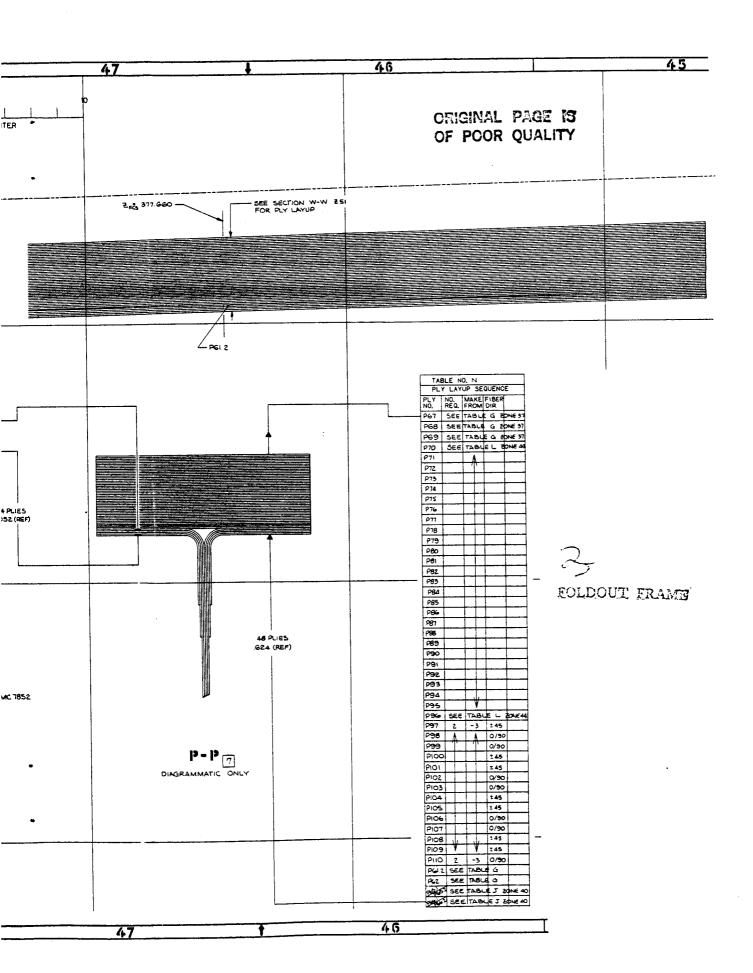
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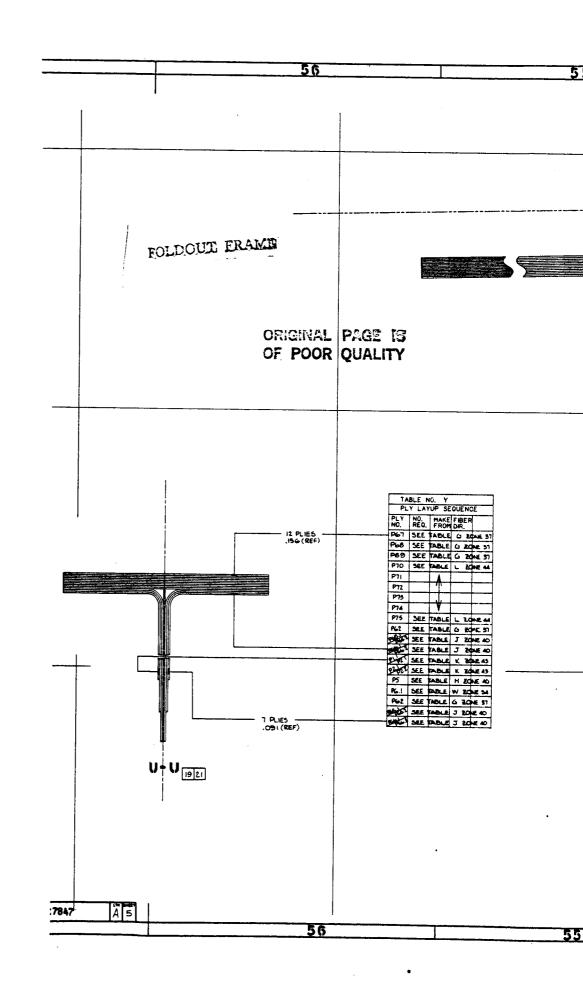
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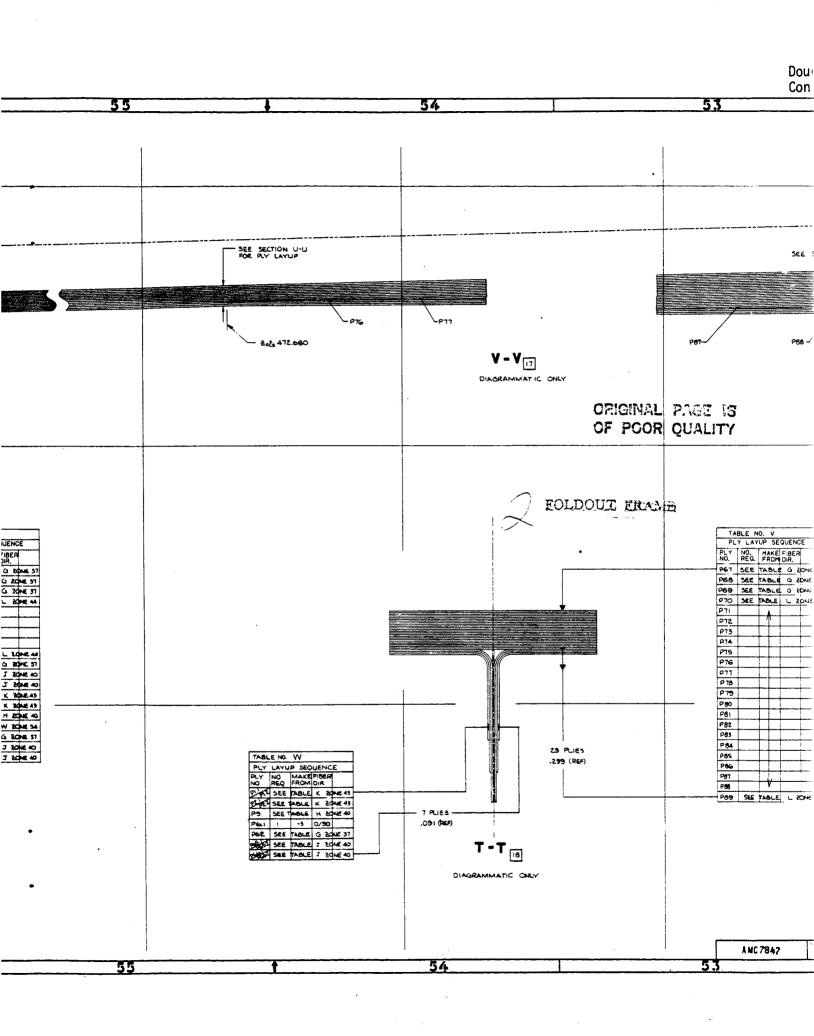
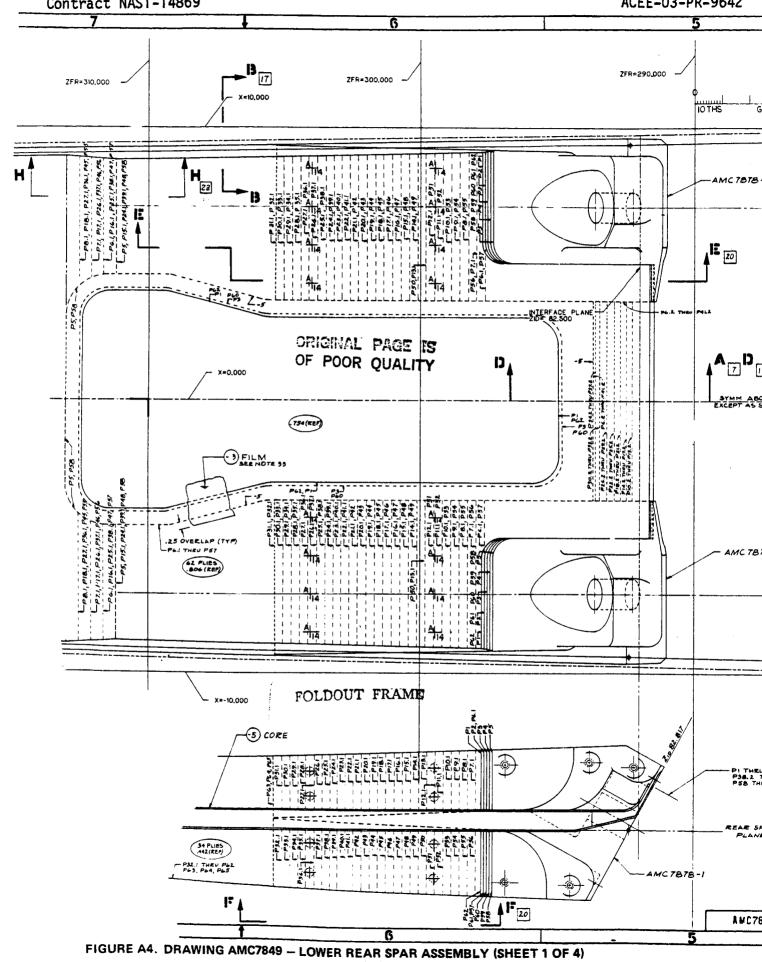
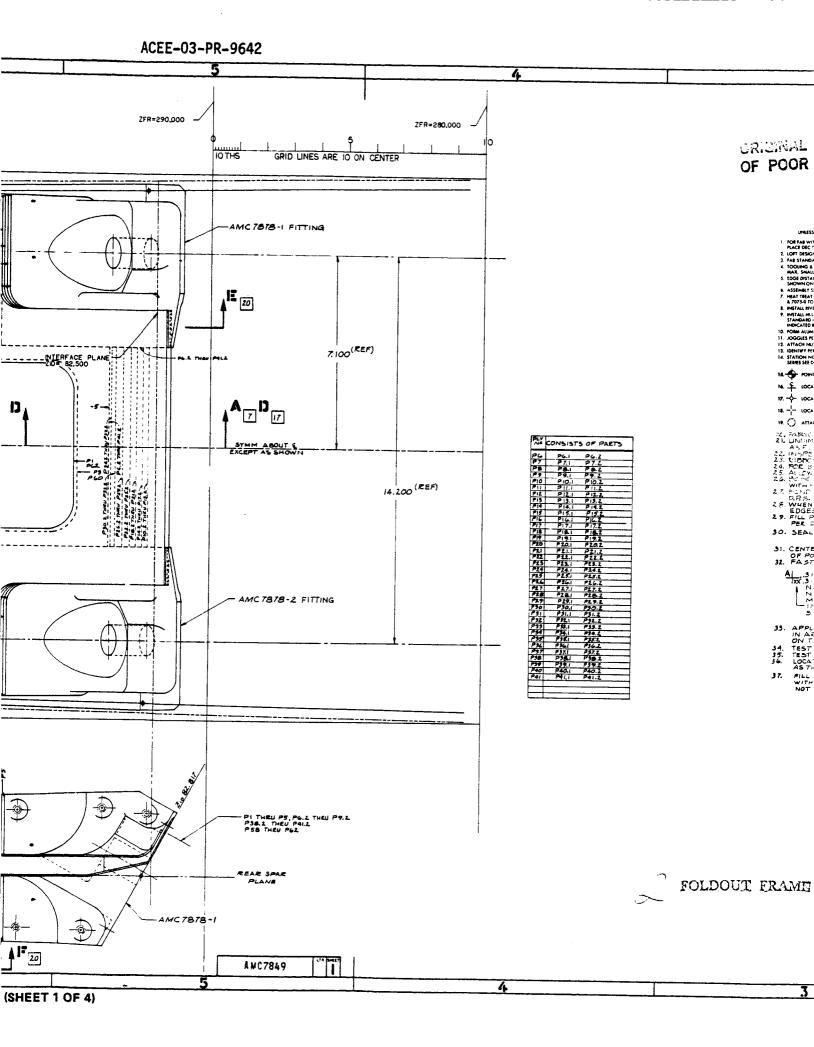
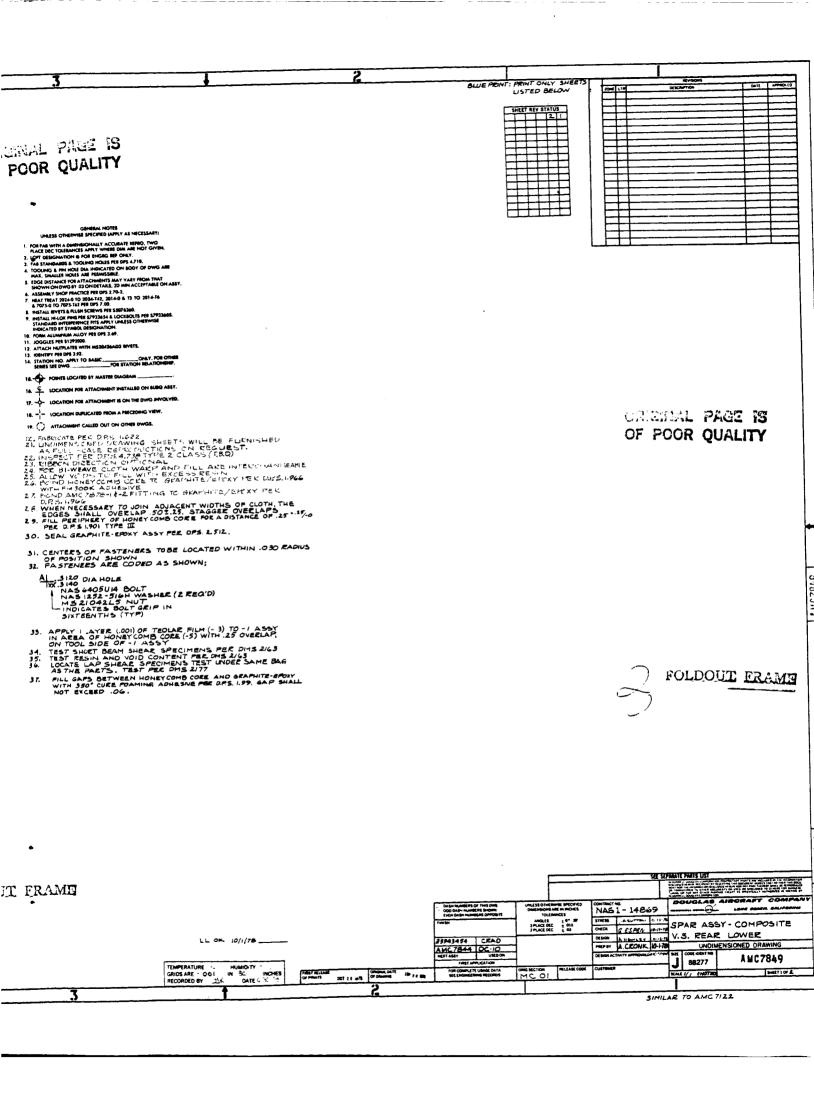


FIGURE A3. DRAWING AMC7847 - FORWARD CENTER SPAR ASSEMBLY (SHEET 10 OF 10)







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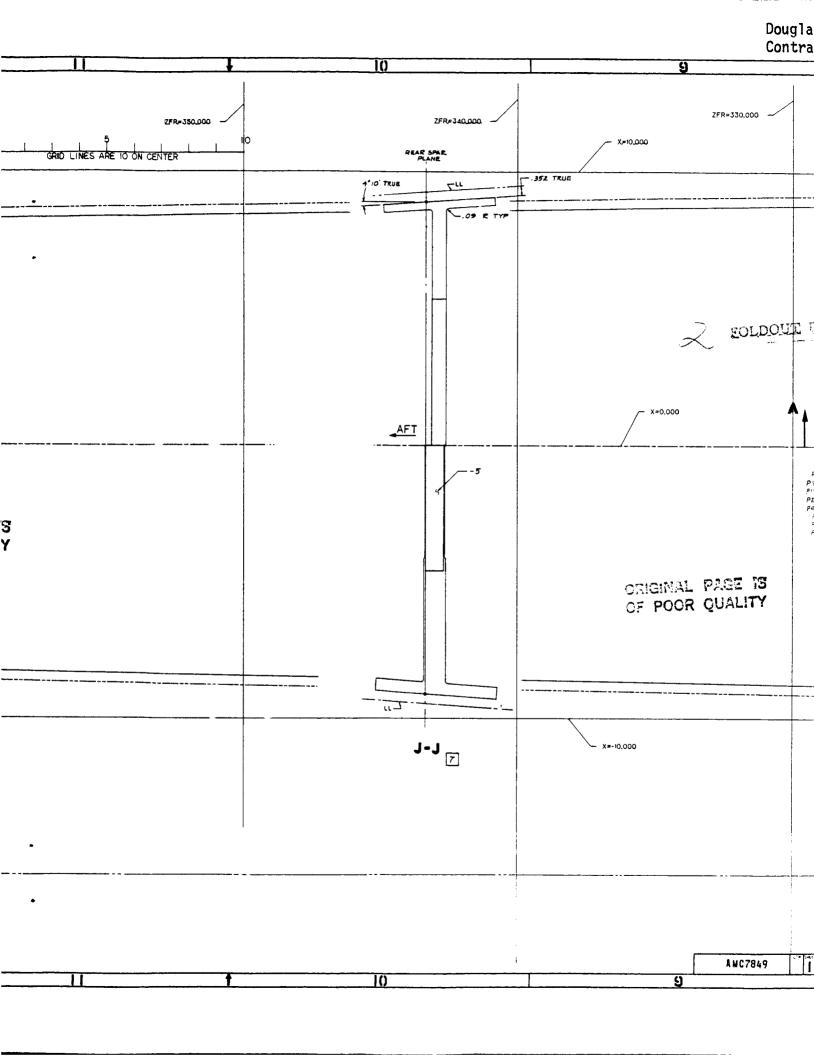


FIGURE A4. DRAWING AMC7849 - LOWER REAR SPAR ASSEMBLY (SHEET 2 OF 4)

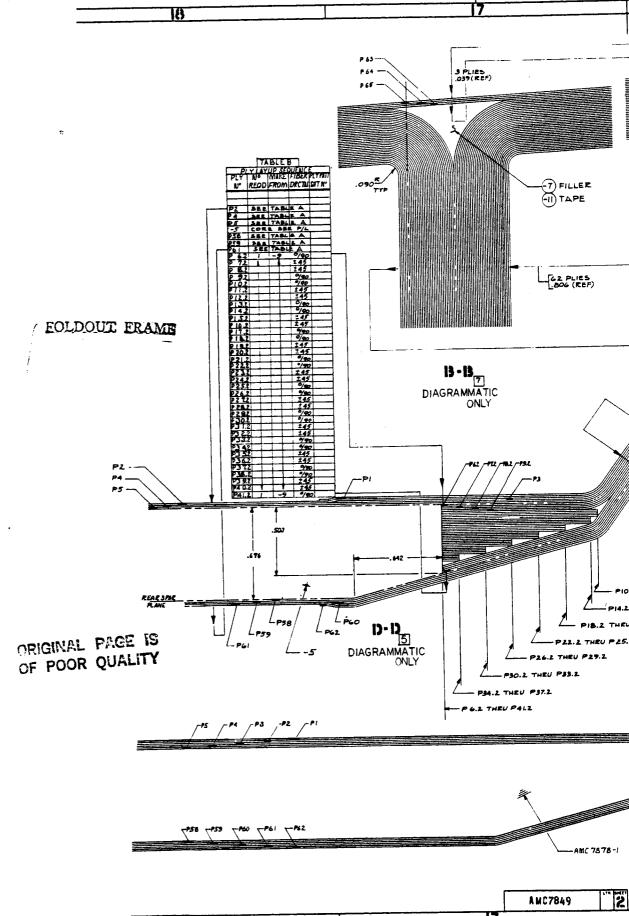
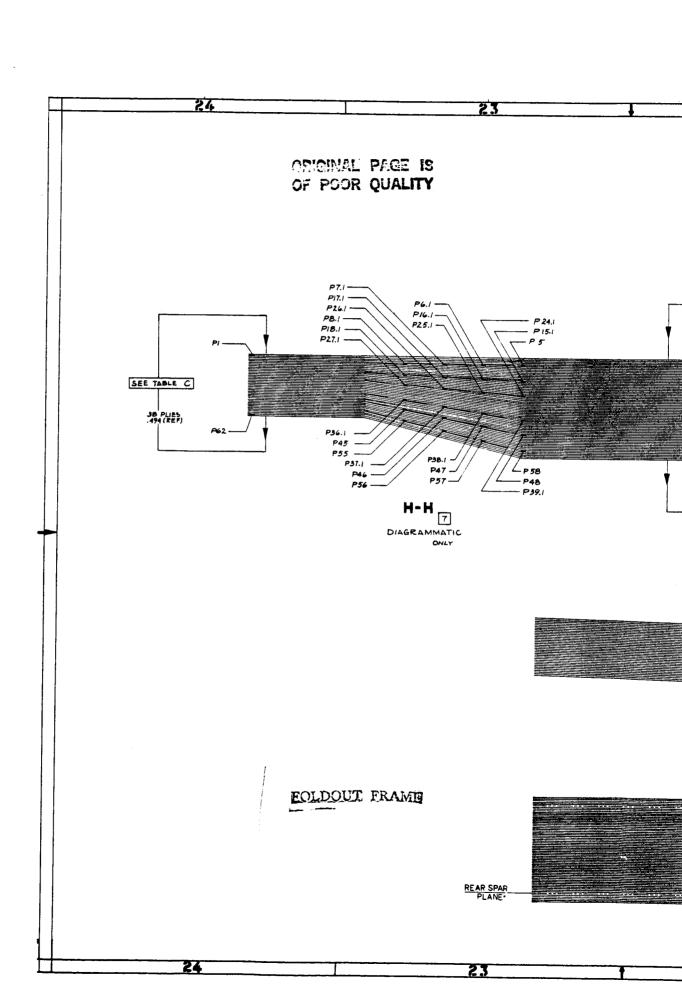
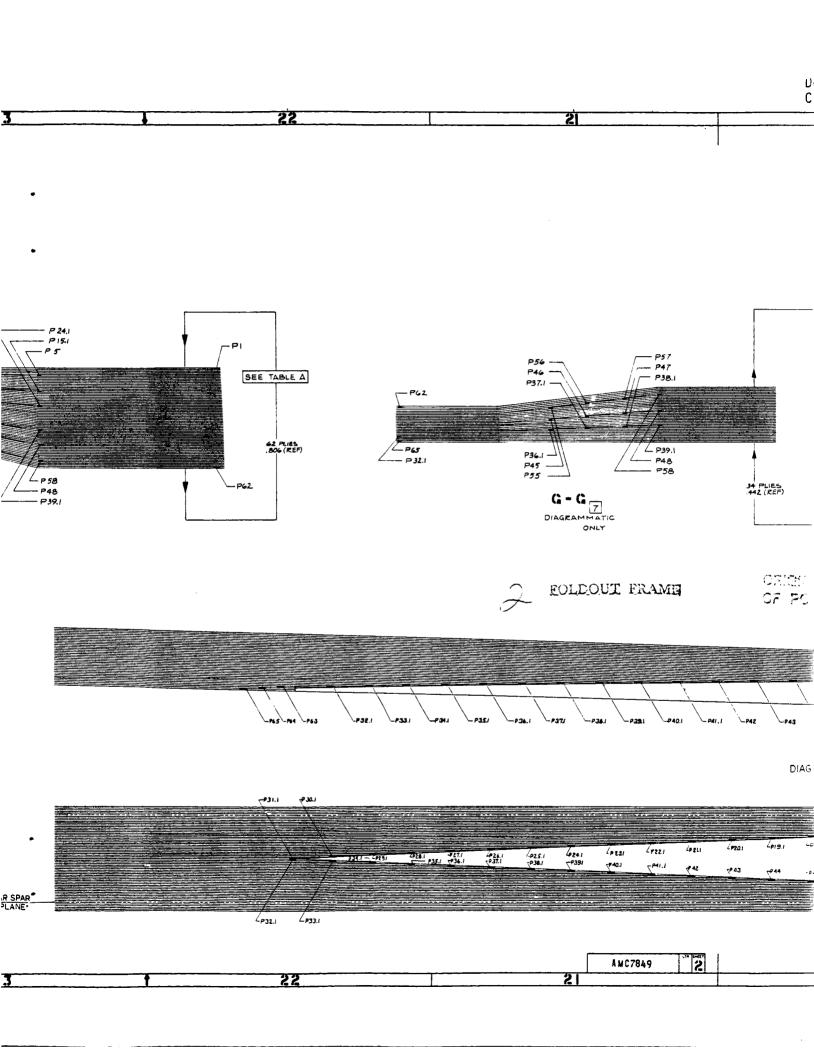


FIGURE A4. DRAWING AMC7849 - LOWER REAR SPAR ASSEMBLY (SHEET 3 OF 4

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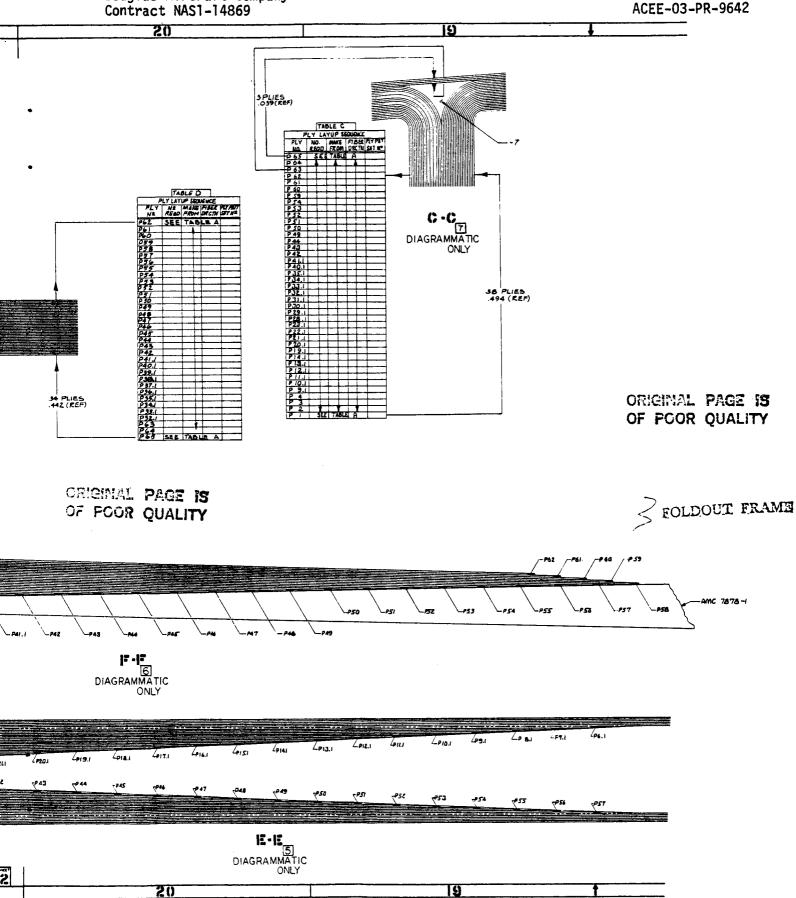
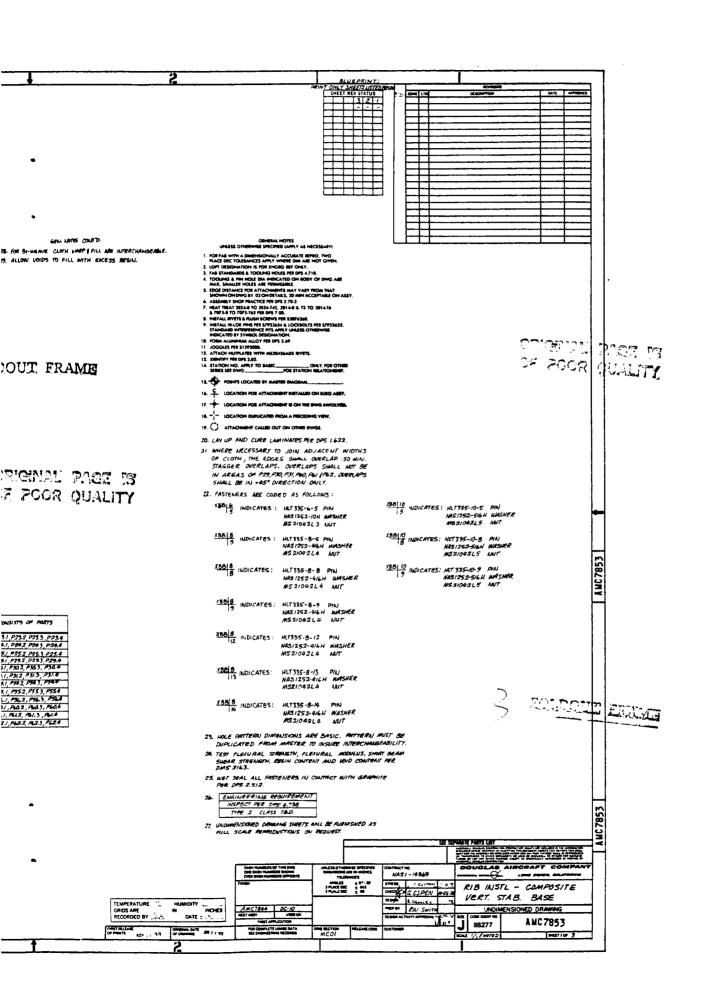
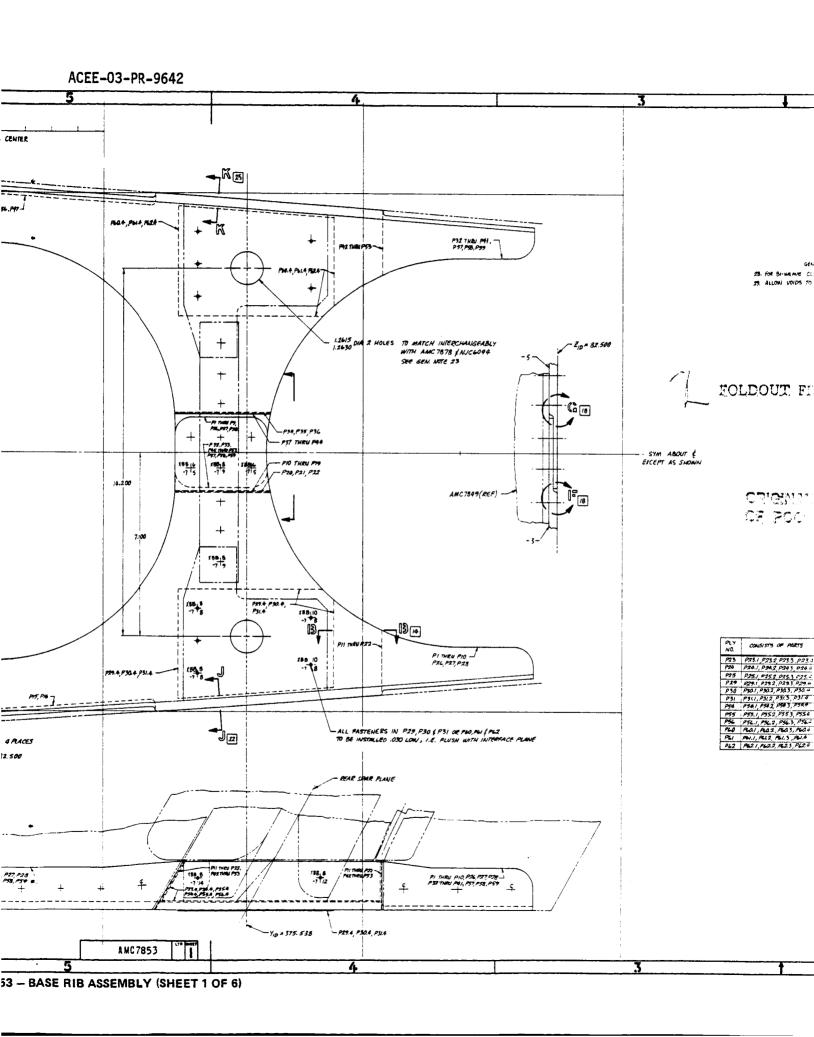
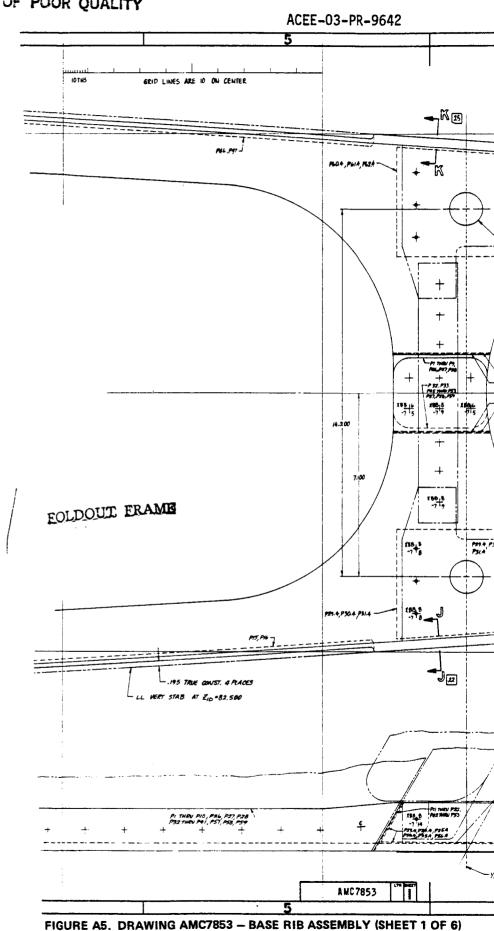


FIGURE A4. DRAWING AMC7849 - LOWER REAR SPAR ASSEMBLY (SHEET 4 OF 4)





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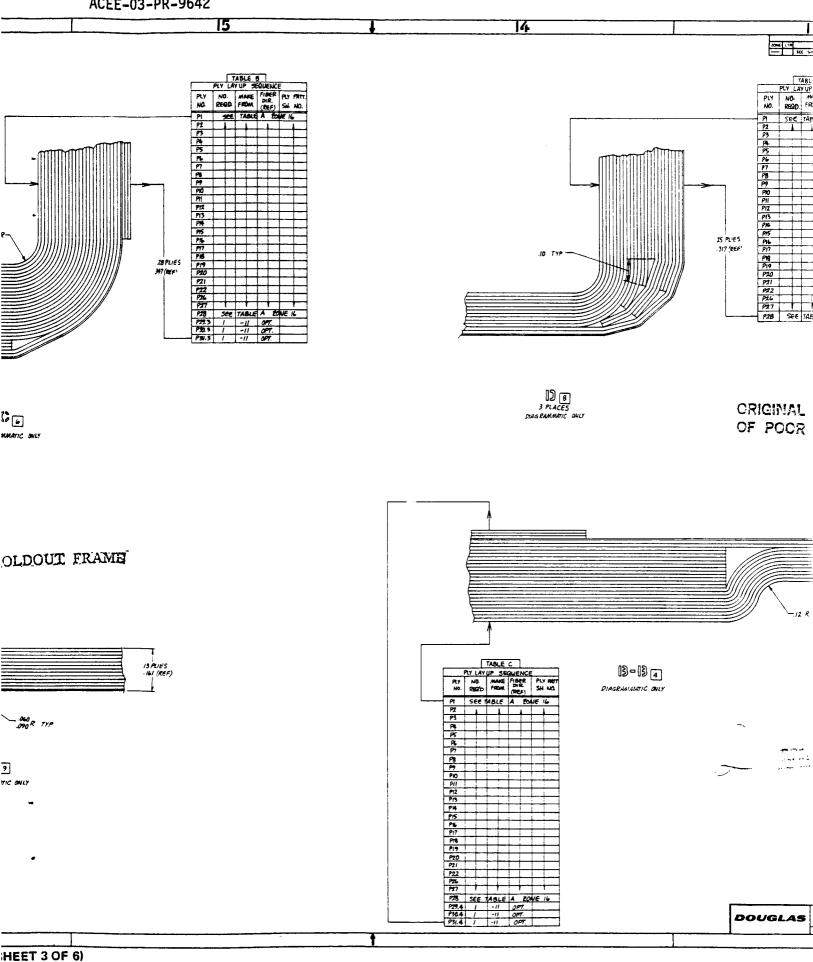
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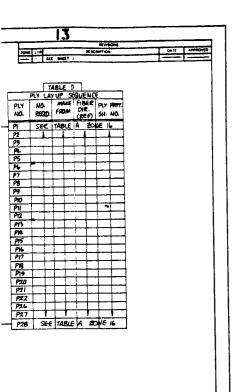
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FIGURE A5. DRAWING AMC7853 - BASE RIB ASSEMBLY (SHEET 3 OF 6)

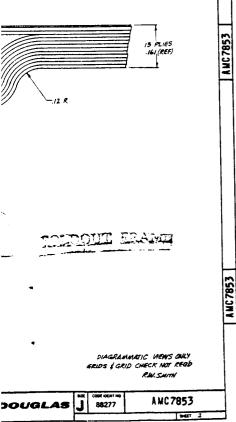
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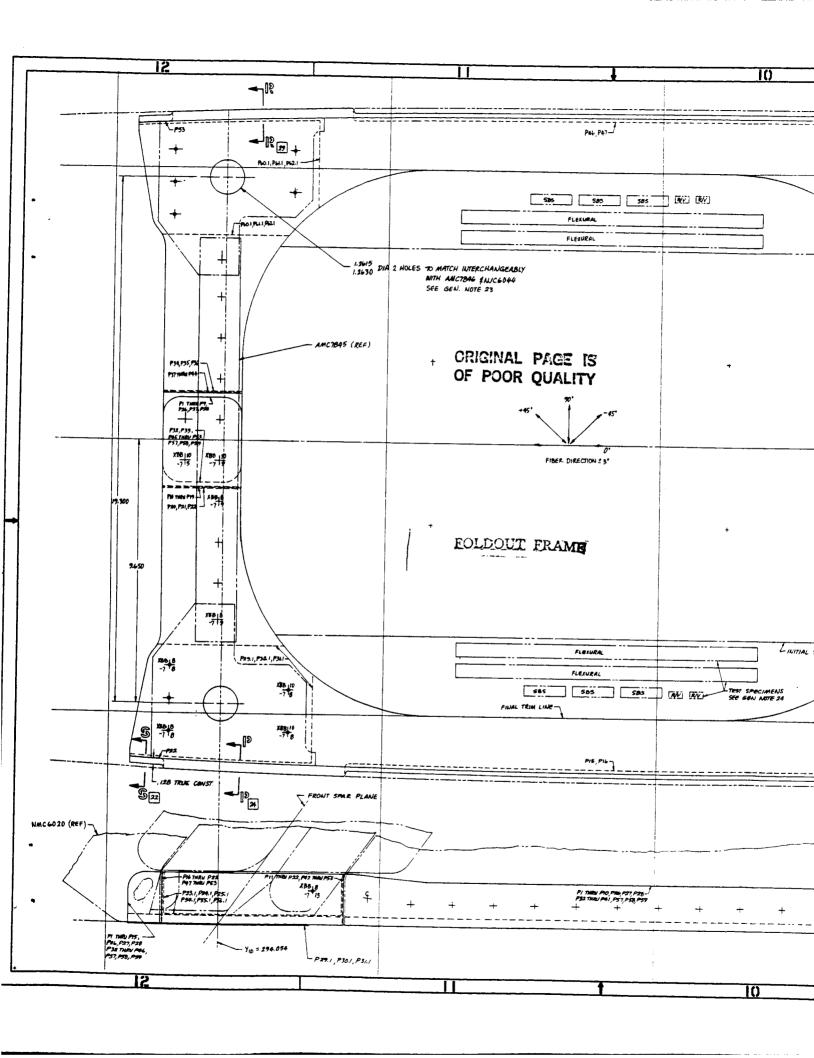
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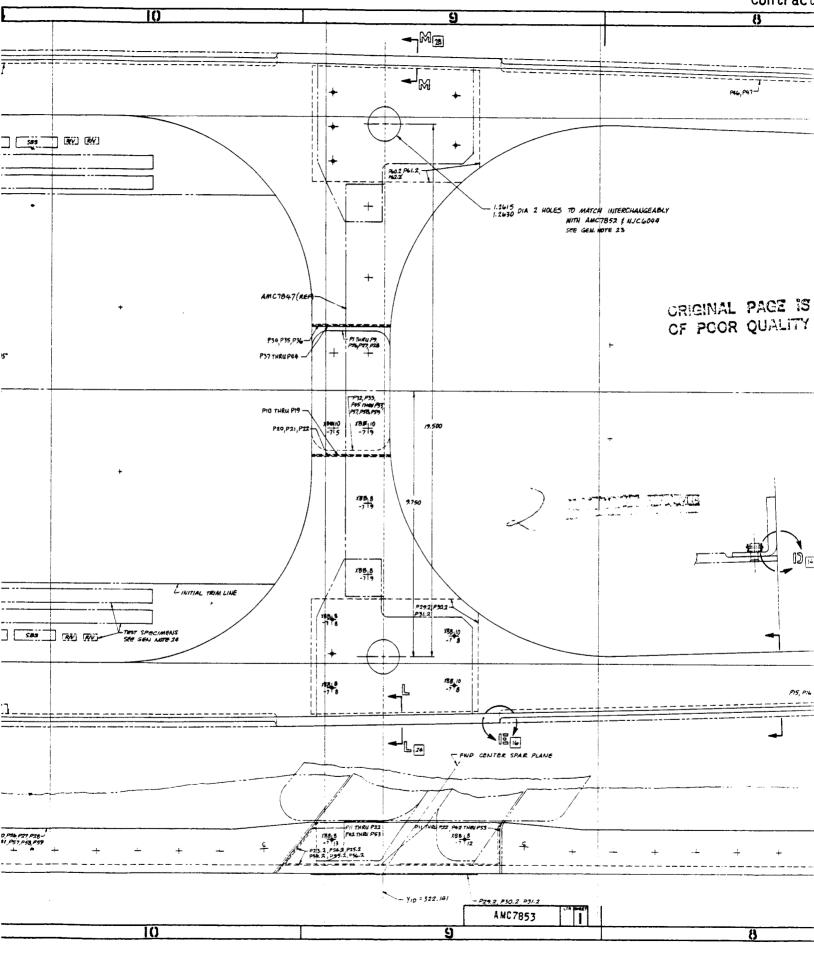


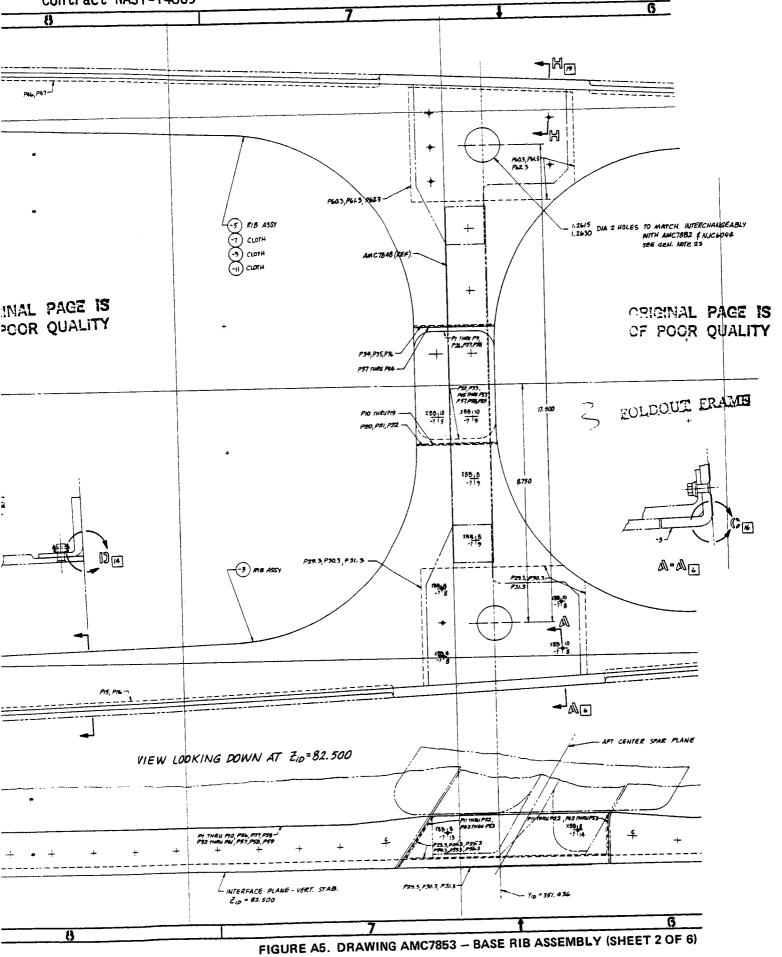


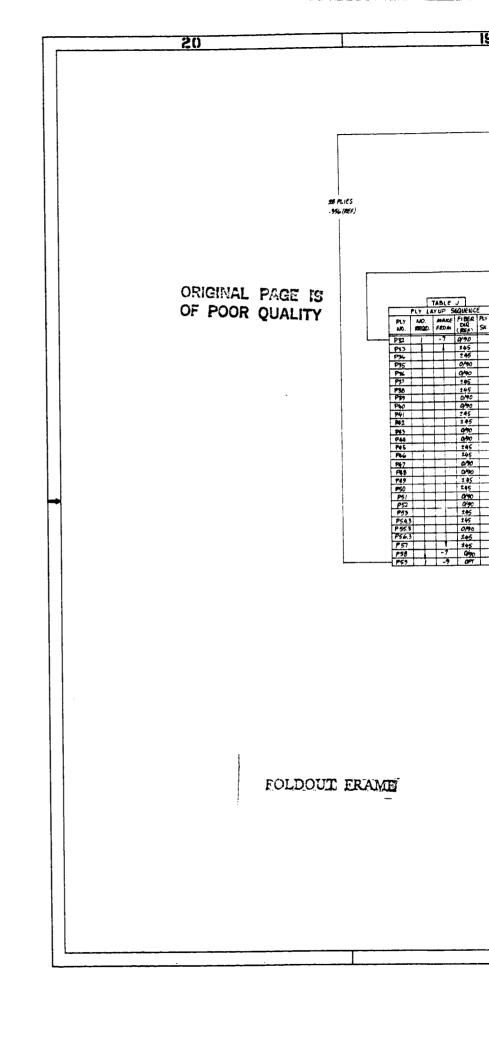
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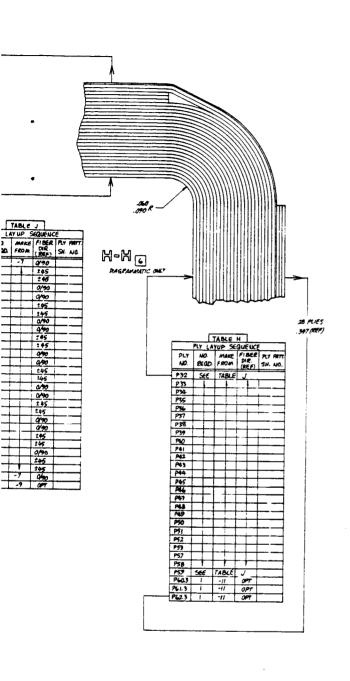


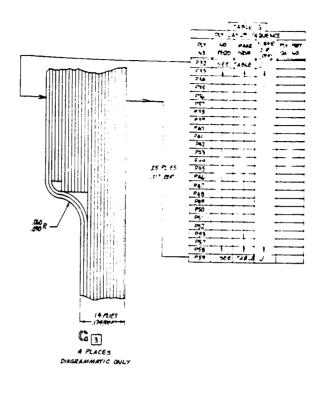


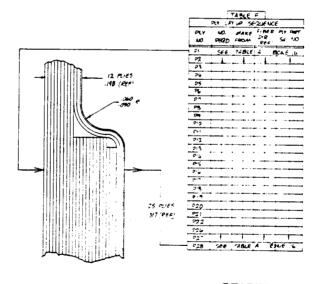




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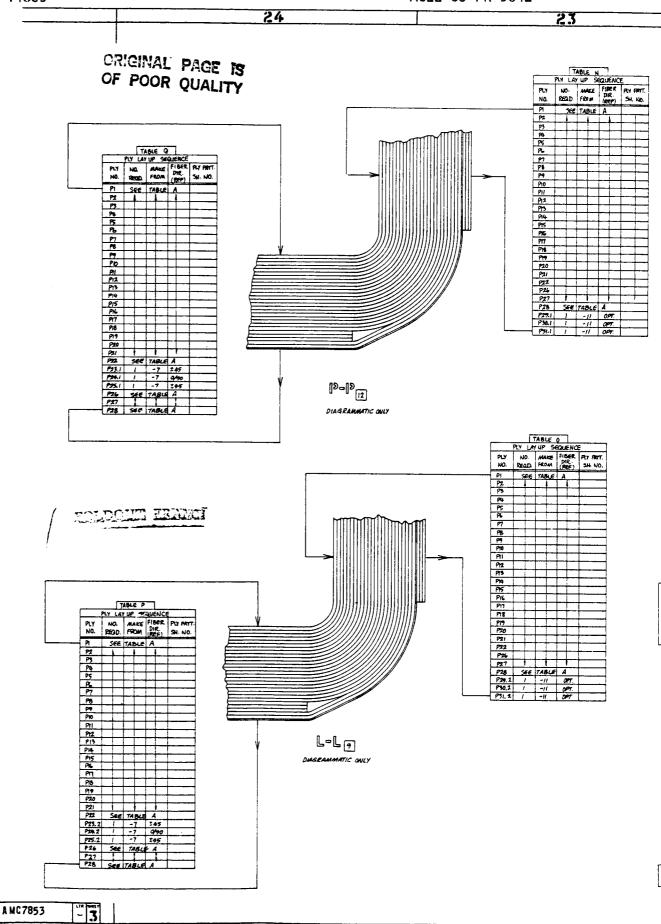


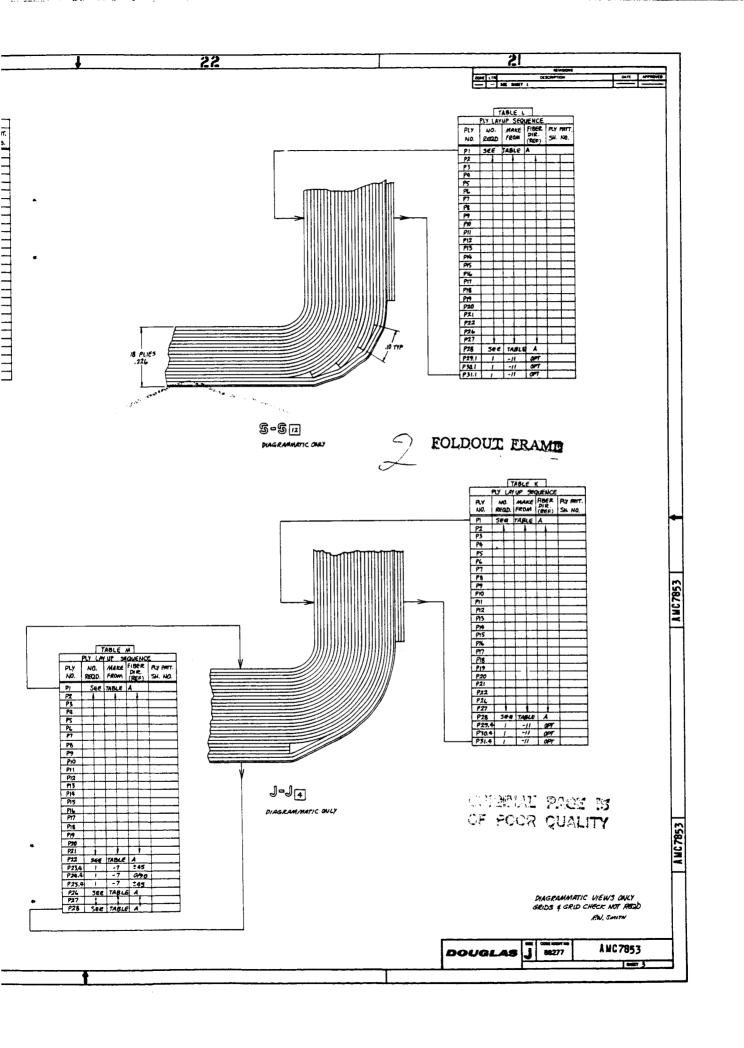


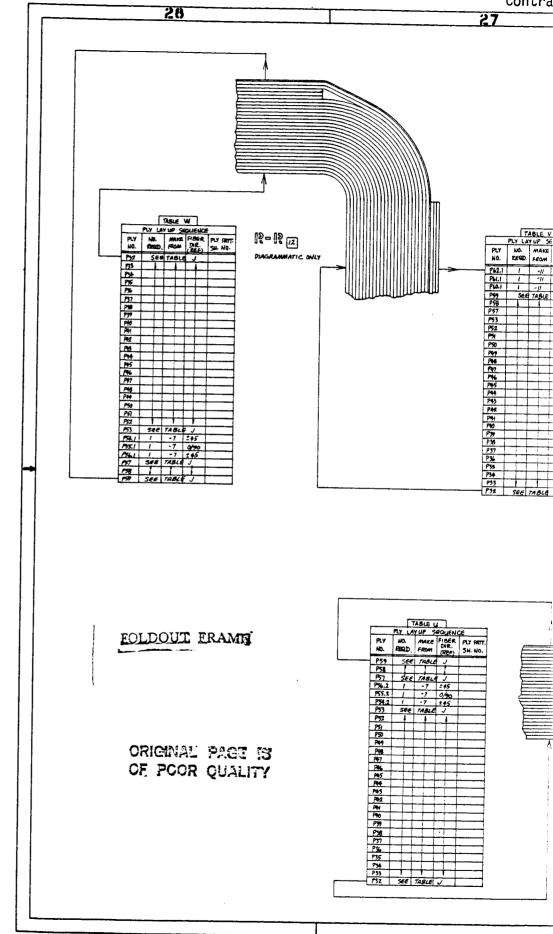


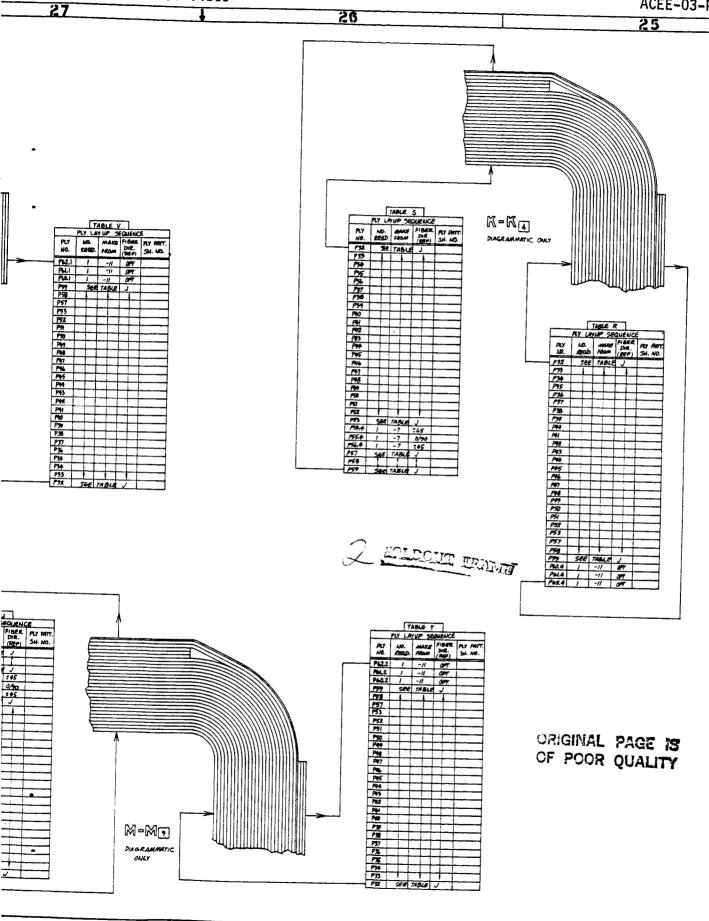
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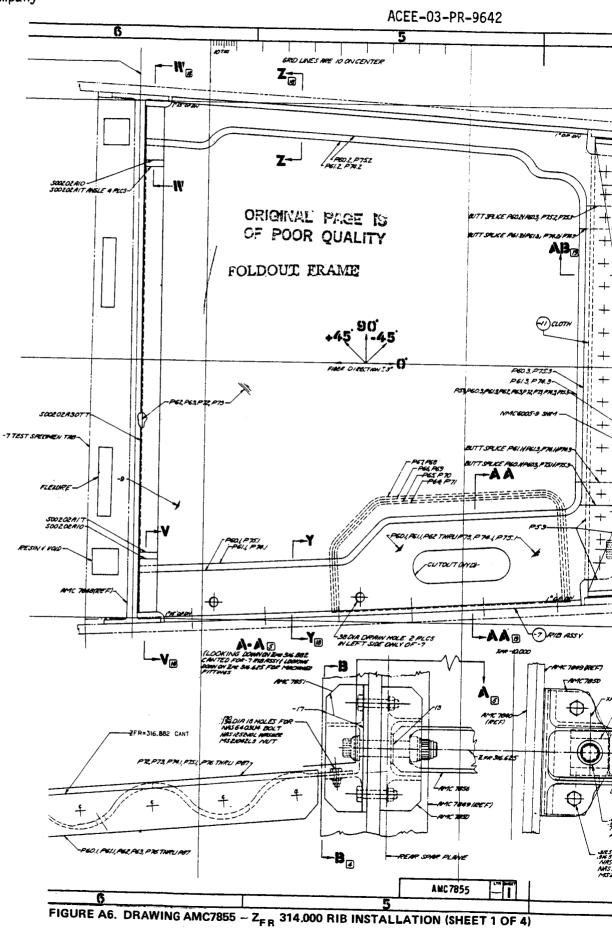
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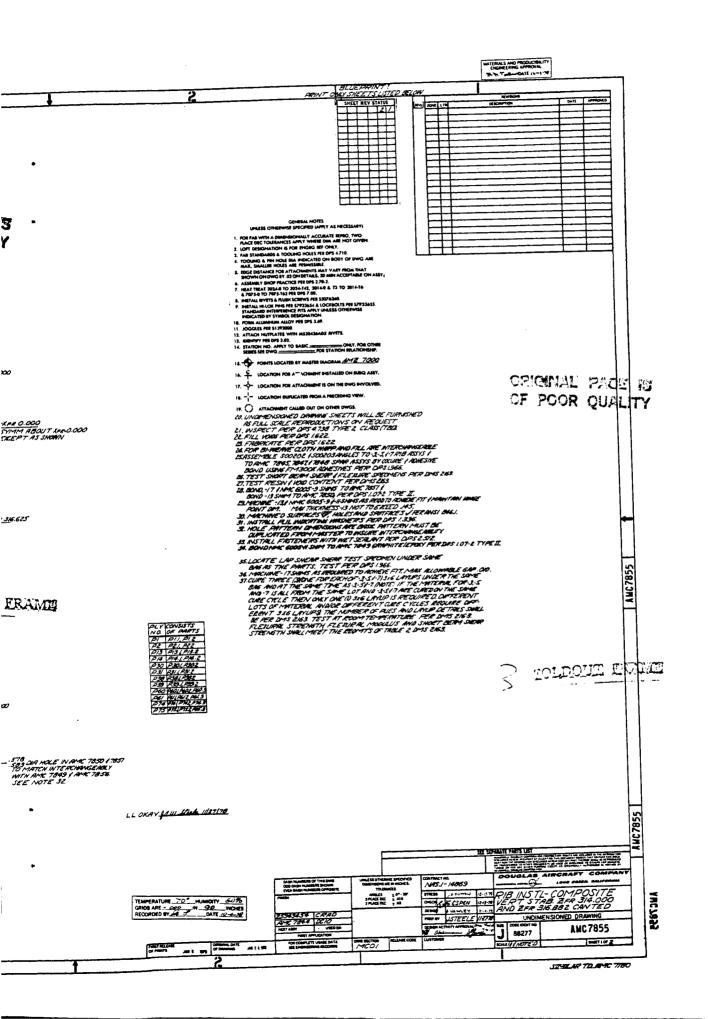












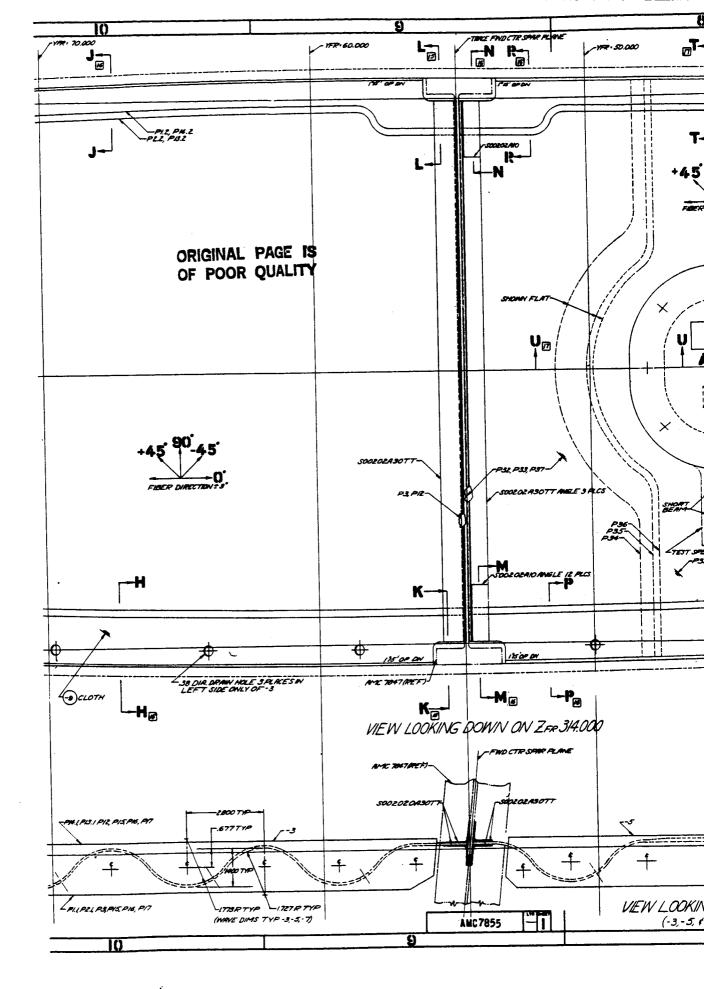


FIGURE A6. DRAWING AMC7855 $-Z_{FR}$ 314.000 RIB INSTALLATION (SHEET 2 OF 4)

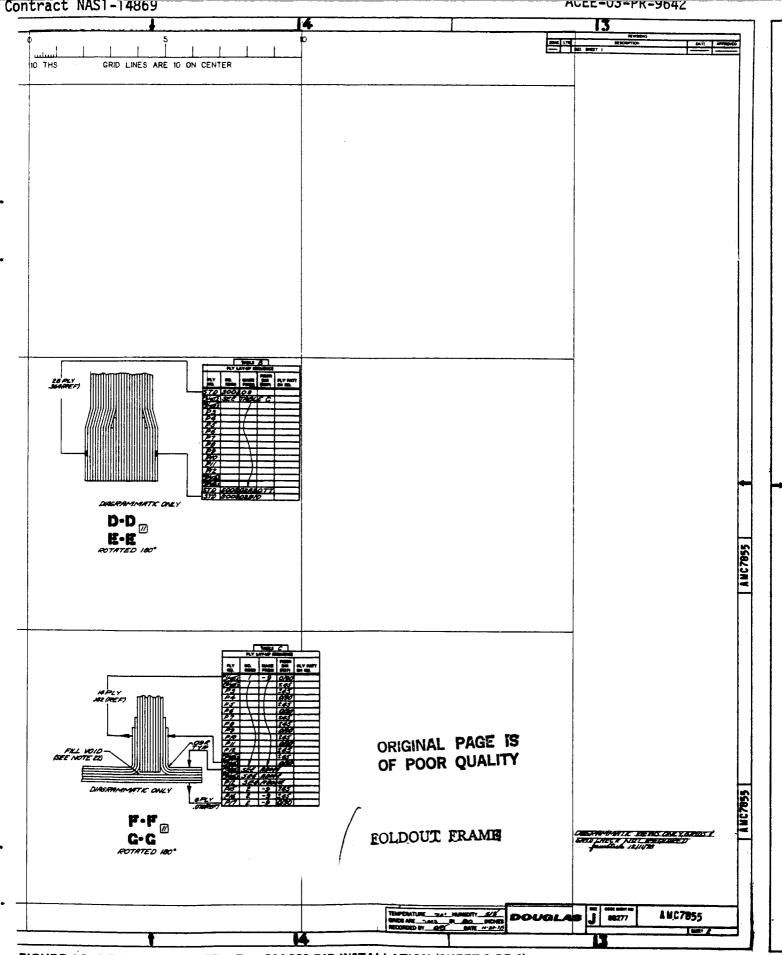
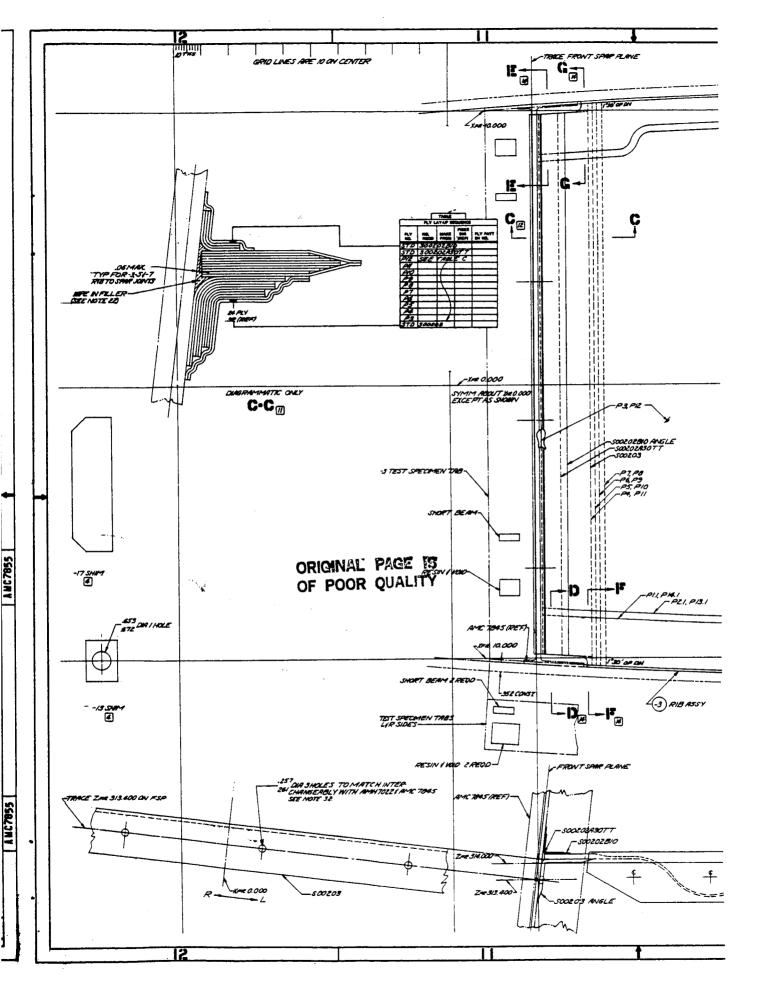
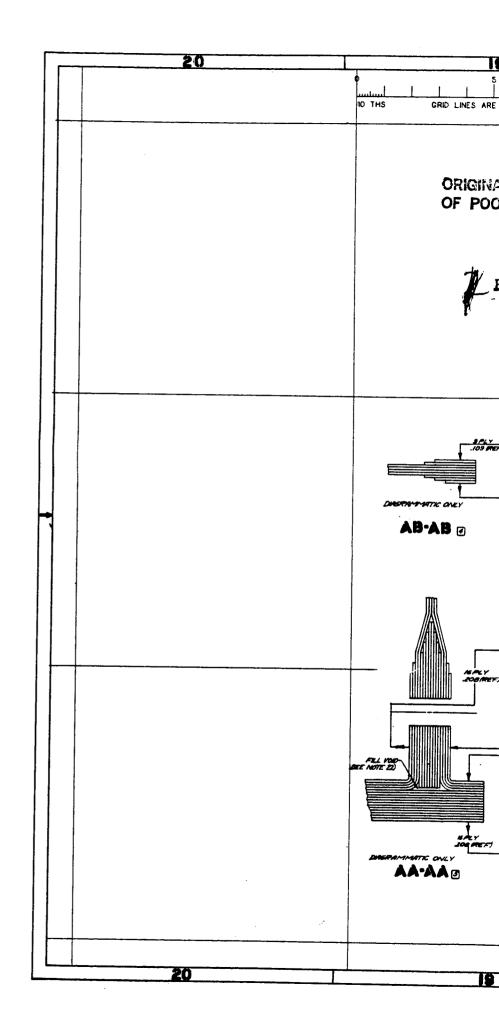


FIGURE A6. DRAWING AMC7855 - Z_{FR} 314.000 RIB INSTALLATION (SHEET 3 OF 4)







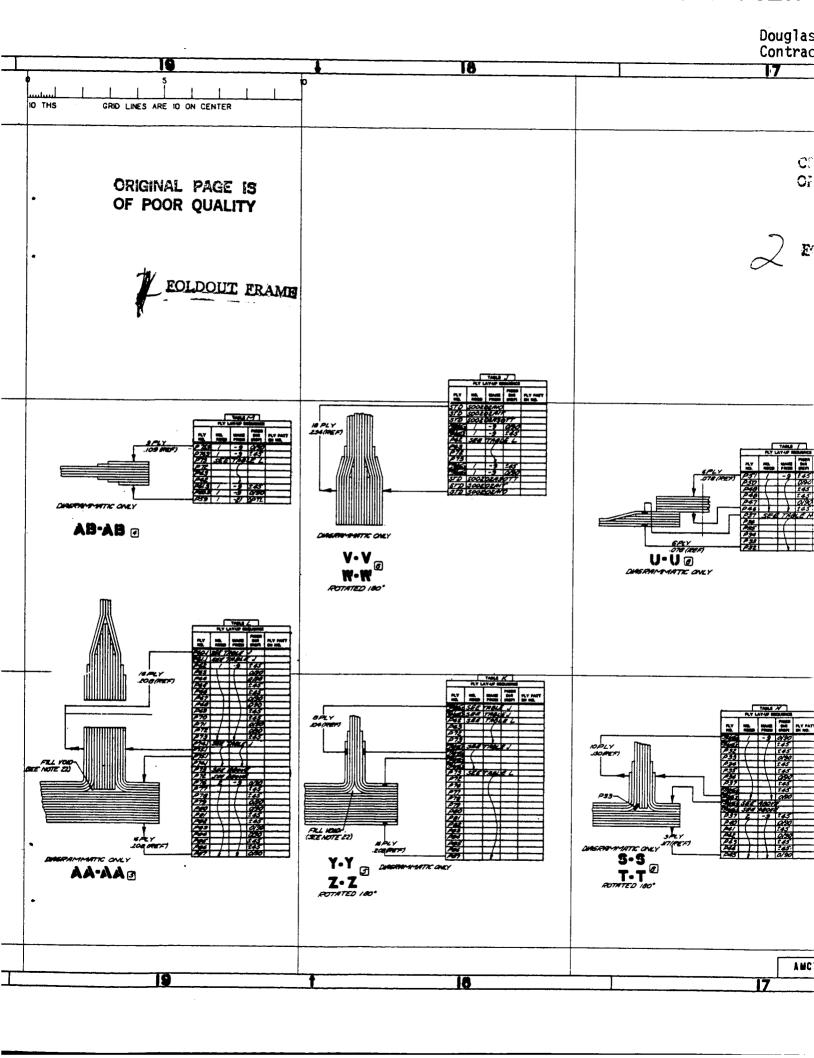
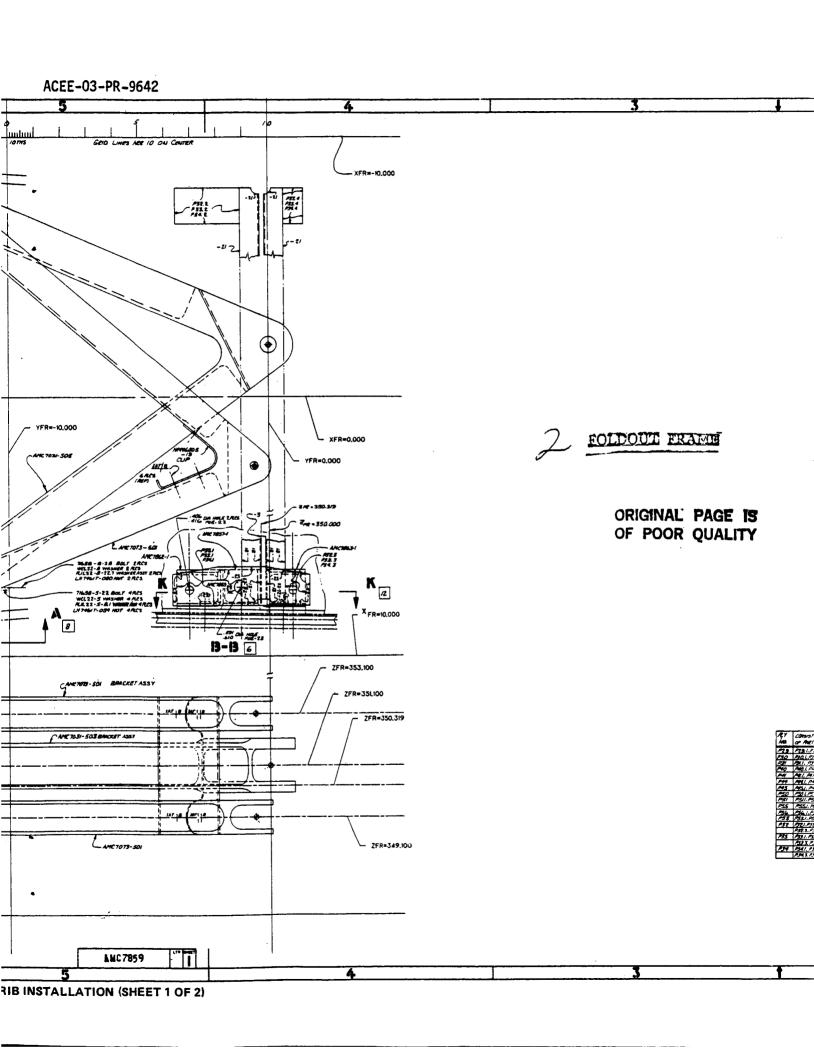
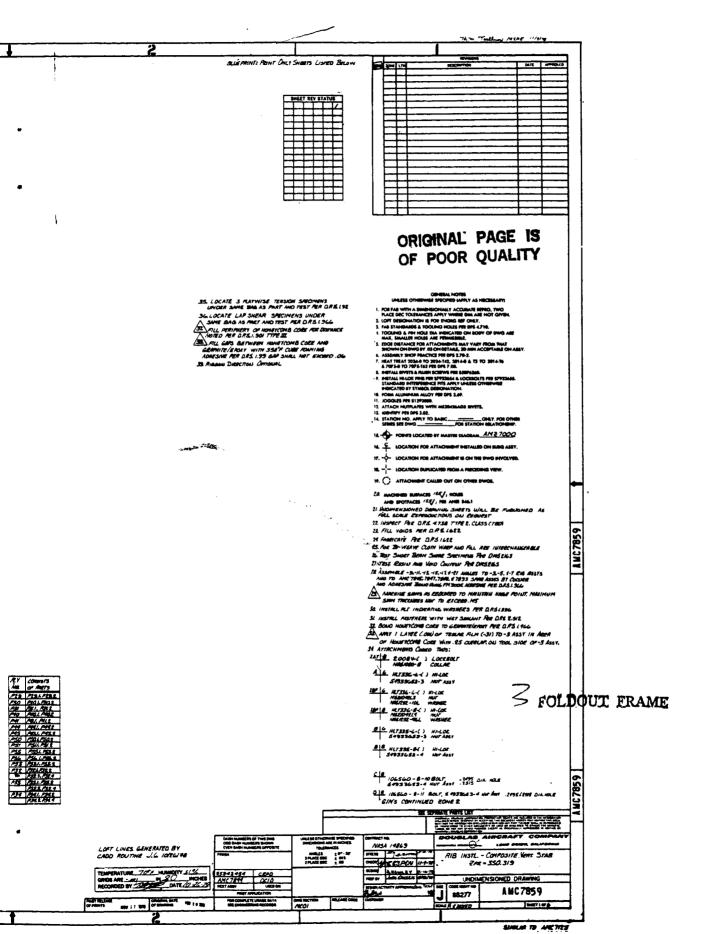


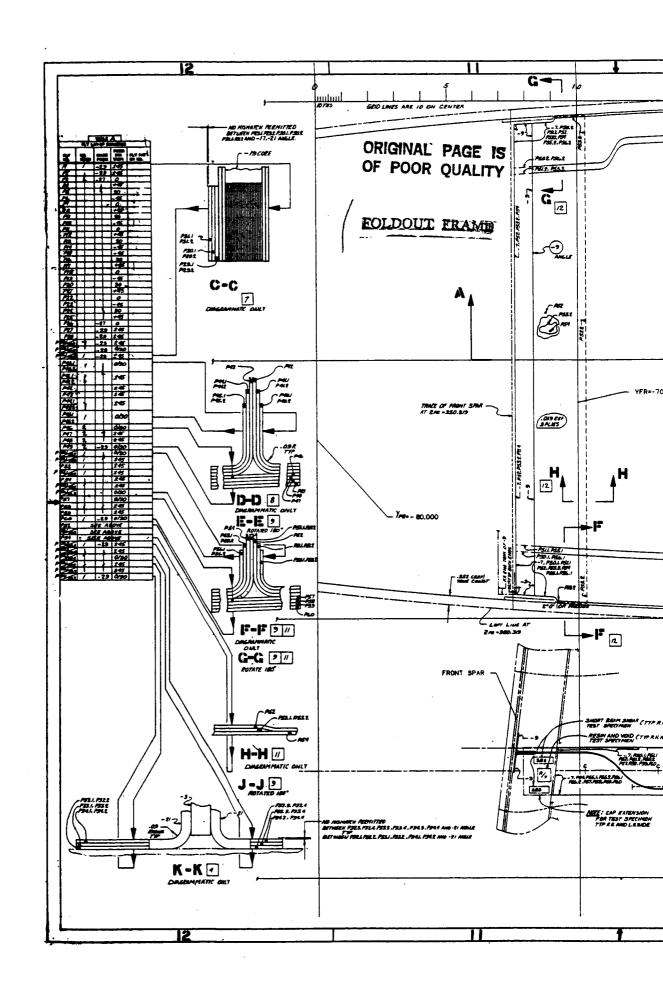
FIGURE A7. DRAWING AMC7859 - ZFR 350.319 RIB INSTALLATION (SHEET 1 OF 2)

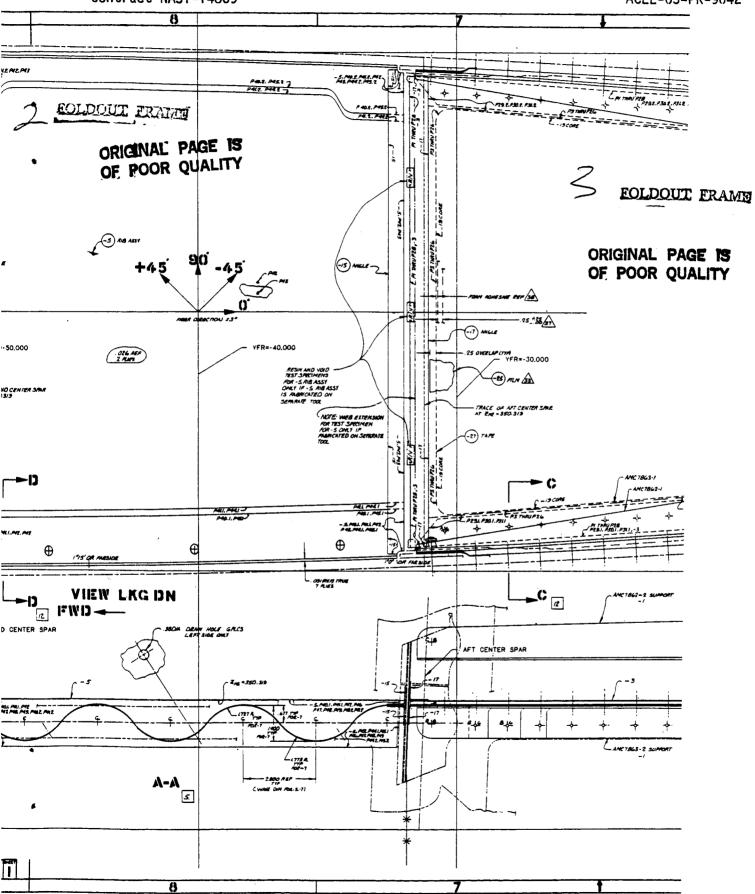




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APPENDIX B

MECHANICAL PROPERTIES TEST DATA

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Douglas Aircraft Company Contract NAS1-14869

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TABLE B-1 SANDWICH BEAM FATIGUE TEST RESULTS

Z3943432-505

LAYUP: (25/50/25) = PERCENT PLIES AT (0°, ±45°, 90°)

STRESS RATIO R = -1.0

	TECT		MAX	LAI	MINATE	SANDWICH	MAX NET		
SPECIMEN NUMBER	TEST TEMP OK (OF)	MOISTURE CONTENT (PERCENT)	LOAD NEWTONS (POUNDS)	WIDTH cm (IN.)	THICKNESS em (IN.)	DEPTH cm (IN.)	STRESS MPa (PSI)	CYCLES TO FAILURE × 10 ⁶	COMMENTS
BET 394	AMBIENT	1,12	3203	3.815	0.1501	4.394	268.69	-	②
			(720)	(1.502)	(0.0591)	(1.730)	(389.70)		
BET 395	AMBIENT	1.21	2135	3.815	0.1491	4.389	180.54	0.021	①
5. •••		1	(480)	(1.502)	(0.0587)	(1.728)	(26.186)	[
BET 396	AMBIENT	0.75	2135	3.818	0.1435	4.387	187.41	0.045	1 1
54. 555			(480)	(1.503)	(0.0565)	(1.727)	(27.182)		
BET 397	AMBIENT	0.76	2135	3.820	0.1455	4.404	183.90	0.127	1
52			(480)	(1.504)	(0.0573)	(1.734)	(26.672)		
BET 398	AMBIENT	0.75	2447	3.823	0.1453	4.381	212.09	0.044	1
52. 555			(550)	(1.505)	(0.0572)	(1.725)	(30,761)		
BET 399	AMBIENT	1.24	2446	3.820	0.1430	4.392	215.06	0.014	①
52. 555		1	(550)	(1.504)	(0.0563)	(1.729)	(311.91)		
BET 400	AMBIENT	0.99	2446	3.820	0.1440	4.387	213.83	0.027	10
DC1 400	/		(550)	(1.504)	(0.0567)	(1.727)	(31.013)		
BET 401	AMBIENT	0.71	2002	3.820	0.1458	4.384	172.96	0.433	10
DE1 40.	,		(450)	(1.504)	(0.0574)	(1.726)	(25.086)	1	
BET 402	AMBIENT	0.70	2002	3.817	0.1427	4.392	176.40	0.267	1
521 402	1		(450)	(1.503)	(0.0562)	(1.729)	(25.585)		_
BET 403	AMBIENT	0.65	1913	3.807	0.1415	4.382	171.02	>1.580	2
DE: 400	1	1	(430)	(1.499)	(0.0557)	(1.725)	(24.804)		
BET 404	AMBIENT	0.64	1933	3.787	0.1377	4.389	176.47	>1.853	2
UL1 404	7		(430)	(1.491)	(0.0542)	(1.728)	(25.595)	1	
BET 391	219	1.24	2002	3.810	0.1384	4.389	182.36	3.000	NO FAILURE
52 1 2 0.	(-65)		(450)	(1.500)	(0.0545)	(1.728)	(26.449)		
BET 392	219	1.12	2135	3.815	0.1405	4.386	191.56	3.000	♦ ♦
DE1 001	(-65)		(480)	(1.502)	(0.0553)	(1.727)	(27.783)		
	1 , 55,		2447				219.50	0.224	
		1	(550)				(31.836)		
BET 393	219	0.91	2447	3.813	0.1433	4.392	215.20	0.291	1 1
DE1 393	(-65)	0.5 ((550)	(1.501)	(0.0564)	(1.729)	(31.212)		
BET 405	219	0.65	2269	3.815	0.1415	4.394	201.72	0.968	10
521 -03	(-65)	0.55	(510)	(1.502)	(0.0557)	(1.730)	(29.258)	1	
BET 427	350	_	1975	3.805	0.1425	4.394	174.95	-	12
BE1 42/	(170)	1 -	(444)	(1.498)	(0.0561)	(1.730)	(25.374)		
BET 407	350	0.58	2358	3.797	0.1457	4.379	205.44	0.023	1
051 40/	1	0.58	(530)	(1.495)	(0.0574)	(1.724)	(29.796)		
DET 400	(170)	0.54	2358	3.797	0.1455	4.386	205.41	0.085	1
BET 408	(170)	0.54	(530)	(1,495)	(0.0573)	(1.727)	(29.791)		1
1	\ \1707		(330)	100/					

LAMINATE FAILURE THROUGH CENTER OF 0.630 cm (0.248 IN.) DIAMETER HOLE

DELAMINATION OF LAMINATE ADJACENT TO LOADING CLAMP

TEST EQUIPMENT MALFUNCTION ALLOWED 480 LB LOAD LEVEL TO DROP OFF TO UNKNOWN MAGNITUDE. RESET LOAD LEVEL TO 550 LB AND CYCLED UNTIL FAILURE.

TABLE B-1 SANDWICH BEAM FATIGUE TEST RESULTS (CONTINUED)

Z3943432-505

LAYUP: (25/50/25) = PERCENT PLIES AT (0°, ±45°, 90°)

STRESS RATIO R = 0.05

	TEST		MAX	LA	MINATE	SANDWICH	MAX NET		
SPECIMEN NUMBER	TEMP K (°F)	MOISTURE CONTENT (PERCENT)	LOAD NEWTONS (POUNDS)	WIDTH cm (IN.)	THICKNESS cm (IN.)	DEPTH cm (IN.)	STRESS MPa (PSI)	CYCLES TO FAILURE × 10 ⁶	COMMENTS
BET 412	AMBIENT	1.45	3692	3.820	0.1372	4.384	338.75	0.101	①
	ļ		(830)	(1.504)	(0.0540)	(1.726)	(49.132)		
BET 413	AMBIENT	1.45	3692	3.817	0.1448	4.392	320.88	-	
			(830)	(1.503)	(0.0570)	(1.729)	(46.540)	0.001	⟨1⟩
BET 414	AMBIENT	1.47	3692	3.815	0.1473	4.394	315.50	0.001	
ļ			(830)	(1.502)	(0.0580)	(1.730)	(45.760)	9.000	NO FAILURE
BET 415	AMBIENT	1.41	3470	3.817	0,1478	4.412	(42.640)	9.000	NO PAILONE
			(780)	(1.503)	(0.0582)	(1.737)	301.81	4.000	NO FAILURE
BET 416	AMBIENT	1.46	3514	3.822	0.1463	4.392		4.000	NO FAILURE
1		1	(790)	(1.505)	(0.0576)	(1.729)	(43.744)	5:000	NO FAILURE
BET 417	AMBIENT	1.37	3670	3.830	0.1534	4.394	(43.500)	5.000	NO FAILURE
l			(825)	(1.508)	(0.0604)	(1.730)		1.833	①
BET 418	AMBIENT	1.50	3692	3.823	0.1468	4.392	316.01 (45.833)	1.533	
			(830)	(1.505)	(0.0578)	(1.729)	1	1.629	\odot
BET 419	AMBIENT	1.64	3959	3.822	0.1499	4.399	331.47	1.029	
			(890)	(1.505)	(0.0590)	(1.732) 4.387	(48.076) 348.82		(4) (6)
BET 420	AMBIENT	1.41	3959	3.828	0.1425	1		_	❖ ❖
i			(890)	(1.507)	(0.0561)	(1.727)	(50.592)	0.506	
BET 421	AMBIENT	1.40	3914	3.840	0.1514	4.384	323.91	0.526	
	1		(880)	(1.512)	(0.0596)	(1.726)	(46.979)	2514	\Diamond
BET 422	AMBIENT	1.29	3870	3.830	0.1527	4.394	317.83	3.514	
			(870)	(1.508)	(0.0601)	(1.730)	(46.098)	0.004	1
BET 423	219	1.32	4137	3.828	0.1476	4.407	350.43	0.021	•
	(-65)		(930)	(1.507)	(0.0581)	(1.735)	(50.826)	4.407	NO FAILURE
BET 409	219	1.02	3603	3.813	0.1450	4.387	313.50	1.107	NOFAILURE
	(-65)		(810)	(1.501)	(0.0571)	(1.727)	(45.468)	4.750	NO FAILURE
BET 410	219	0.85	3692	3.813	0.1438	4.394	323.42	1.750	NOFAILORE
	(-65)		(830)	(1.501)	(0.0566)	(1.730)	(46.910)	1.900	NO FAILURE
BET 411	219	1.37	3825	3.815	0.1430	4.394	336.60	1.900	NO PAILURE
	(-65)		(860)	(1.502)	(0.0563)	(1.730)	(48.820) 242.86		⑤
BET 424	350	_	2785	3.833	0.1440	4.379	(35.224)	-	🔻
	(170)		(626)	(1.509)	(0.0567)	(1.724)	231.95		⑤
BET 425	350	_	2785	3.804	0.1519	4.389	1	_	💝
	(170)		(626)	(1.498)	(0.0598)	(1.728)	(33.641)	8.000	NO FAILURE
BET 426	350	0.80	2785	3.815	0.1438	4.392	(35,373)	0.000	NO AILONE
	(170)		(626)	(1.502)	(0.0566)	(1.729)	(35.3/3)		

LAMINATE FAILURE THROUGH CENTER OF 0.630 cm (0.248 IN.) DIAMETER HOLE

IMMEDIATE FAILURE AFTER START OF TEST. UNABLE TO OBTAIN NUMBER OF CYCLES SINCE COUNTER IS CALIBRATED IN 1000 CYCLES.



JIG FAILURE

LAMINATE TENSILE FAILURE ADJACENT TO LOAD PAD

Douglas Aircraft Company Contract NAS1-14869

LAYUP: (25/50/25) = PERCENT PLIES AT (0°, ±45°, 90°)

TABLE B-2
FATIGUE TEST RESULTS FOR DEBONDED LAMINATE TENSION SPECIMENS

Z394342-1 STRESS RATIO R = -1.0

																										А
атн2)	PSI	ı	ı	ı	1	ı	39,304	ı		ı	I	1	36,369		l	1	ı	1		39,973	ı	ı	I	1	l	ı
STREN	MPa	ı	ı	ļ	ı	ı	270.99	1	ı	ı	ì	1	250.76		1	1	1	1	1	275.61	ı	ı	ı	ı	ı	_
FAILURE	O	130,000	12,750	130,000	22,090	130,000	130,000	130,000	23,280	130,000	87,500	130,000	130,000		130,000	1,650	130,000	23,900	130,000	130,000	130,000	460	130,000	14,170	130,000	109,620
ESS	PSI	14,911	29,823	14,776	26,865	14,775	21,492	14,971	29,942	14,906	27,103	14,652	125,771		16,180	28,800	16,323	21,803	14,916	18,276	14,901	28,271	14,785	21,039	14,916	18,276
STR	MPa	102.81	205.62	101.87	185.23	101.87	148.18	103.22	206.44	102.78	186.87	101.02	146.94		104.66	198.57	105.65	150.33	102.84	126.01	102.74	194.92	101.94	145.06	102.84	126.01
	POUNDS	2630	20909	2530	4600	2630	3680	2530	2060	2530	4600	2530	4450		2530	4800	2630	3600	2630	3100	2530	4800	2530	3600	2430	3100
	NEWTONS	11,254	22,508	11,264	20,462	11,254	16,369	11,264	22,508	11,254	20,462	11,254	16,369		11,254	21,351	11,254	16,014	11,254	13,789	11,264	21,361	11,254	16,014	11,254	13,789
NESS	S.	0.0566		0.0570		0.0570		0.0562		0.0565		0.0576			0.0555		0.0650		0.0566		0.0666		0.0670		0.0565	
THICK	CM	0.1435		0.1448		0.1448		0.1428		0.1435		0.1461			0.1410		0.1397		0.1435		0.1435		0.1448		0.1435	
тн	Z.	<u> </u>		3.004		3.004		3.007		3004		3,003			3.003		3.002		3.002		3006		3.002		3.002	
MI	3	7.628		7.630		7.630		7.638		7.630		7.628			7.628		7.625		7.625		7.633		7.625		7.625	
MOISTURE	%	AMB		AMB		AMB		1.55		1.58		1.45			AMB		AMB		AMB		1.18		1.31		96'0	
ST MP	3	-65		9		8		-65		8		59			170		170		170		170		170		170	
¥ Ē	A A	219		219		219		219		219		219			320		320		320		320		320		380	
	SPECIMEN	A065-1		A065-2		A065-3		P065-1		P065 2		P065-3			A170-1		A170-2		A170-3		P170-1		P170-2		P170-3	
	WIDTH THICKNESS STRENGTH(2)	TEST MOISTURE SECTION TO THICKNESS FAILURE TEMP LEVEL WIDTH THICKNESS FAILURE OK OF % CM IN. CM IN. NEWTONS POUNDS MPa PSI ① M	TEST MOISTURE WIDTH THICKNESS FOUNDS FALLURE STRENGTH (2) TEMP LEVEL WIDTH THICKNESS FALLURE STRENGTH (2) OK OF % CM IN. NEWTONS POUNDS MPa PSI (1) MPa PSI 219 -65 AMB 7.628 3.003 0.1436 0.0666 11,264 2630 102.81 14,911 130,000	TEST MOISTURE LEVEL WIDTH THICKNESS FOUNDS MPa PSI TEMP FAILURE STRENGTH (2) A MPa PSI TEMP P	TEST MOISTURE TEMP LEVEL WIDTH THICKNESS TEMP SECTION STRESS FAILURE STRENGTH (2) OK OF % CM IN. NEWTONS POUNDS MPa PSI (1) MPa PSI 219 -65 AMB 7.630 3.004 0.1448 0.0570 11.264 2530 101.87 14,776 130,000	TEST MOISTURE LEVEL WIDTH THICKNESS TEMP LEVEL WIDTH THICKNESS TEMP LEVEL WIDTH THICKNESS TEMP SECTION STRESS FAILURE STRENGTH (2) STRESS FAILURE STRENGTH (2) NPa PSI (1) MPa PSI (1) MPa PSI (219 —65 AMB 7.630 3.004 0.1448 0.0570 11,264 26,300 101.87 14,776 130,000	TEST MOISTURE LEVEL WIDTH THICKNESS TEMP LEVEL LEVEL WIDTH THICKNESS TEMP THORNESS TEMP THORNESS TEMP THORNESS THESS FALLURE STRENGTH (2) STRENGTH (2) STRENGTH (2) STRENGTH (2) THORNESS THORNESS	TEST MOISTURE LEVEL WIDTH THICKNESS TEMP LEVEL WIDTH THICKNESS TEMP TEMP LEVEL WIDTH THICKNESS TEMP THORNESS THORN	TEST MOISTURE LEVEL WIDTH THICKNESS TEMP LEVEL WIDTH THICKNESS TEMP TEMP LEVEL WIDTH THICKNESS TEMP THICKNESS THENGTH (2) SECTION STRESS FAILURE STRENGTH (2) STRESS FAILURE STRENGTH (2) MPa PSI THICKNESS THICKNESS STRESS FAILURE STRENGTH (2) MPa PSI THICKNESS TH	TEST MOISTURE LEVEL WIDTH THICKNESS FOUNDS MPa PSI (1) MPa PSI PSI (1) MPa PSI PSI PSI (1) MPa PSI PSI PSI (1) MPa PSI PSI (1) MPa PSI PSI (1) MPa PSI PSI PSI (1) MPa PSI PSI PSI (1) MPa PSI	TEST MOISTURE LEVEL MIDTH THICKNESS SECTION THICKNESS THICKNESS FAILURE STRENGTH STRENGTH	TEST MOISTURE LEVEL WIDTH THICKNESS FALLUNE STRENGTH (2) OK OF % CM IN. CM IN. NEWTONS POUNDS MPa PSI (1,911 130,000	TEST MOISTURE THICKNESS THICKNESS	TEST MOISTURE LEVEL MIDTH THICKNESS OK OK OK OK OK OK OK OK OK	TEST MOISTURE LEVEL WIDTH THICKNESS OK OK OK OK OK OK OK OK OK	TEST MOISTURE MOISTURE LEVEL WIDTH THICKNESS SECTION TEST SECTION TO KNOW THE LEVEL WIDTH THICKNESS SECTION TO KNOW THE LEVEL WHITH THICKNESS SECTION TO KNOW THE LEVEL SECTION THICKNESS SECTION TO KNOW THE LEVEL SECTION THICKNESS SECTION THICKNES	TEST MOISTURE MO	TEST MOISTURE MIDTH THICKNESS SECTION THEST MIDTH THICKNESS THESS THESS THESS THESS THESS THESS	TEMP MOISTURE MIDTH THICKNESS SECTION TEMP STREESS FAILURE FSI STREESS FAILURE FSI STREESS TAGGO STATE STA	TEST MOISTURE LEVEL WIDTH THICKNESS STREES FALLING THE STREES STREES FALLING THE STREES THE STREES	THENT MOISTURE MOISTURE MOISTURE MOISTURE MOISTURE STREESS FILLING STREESS FILLON FILL	TEMP	TEMPT MOSTURE THICKNESS STRICTON THICKNESS TEMPTON THEORY TEMPTON THEORY THICKNESS STRICTON THEORY THEORY THEORY TEMPTON THEORY THE	TEST MOSTURE WIDTH THICKNESS STRICT STRENGTH & STRENGT	TEMP LEVEL MIDTH THICKNESS STRINGS STRENGTH STRENGTH	Test

TABLE B-2 (Cont'd) FATIGUE TEST RESULTS FOR DEBONDED LAMINATE TENSION SPECIMENS Z39432-1

STRESS RATIO R = -1.0

LAYUP: (25/50/25) = PERCENT PLIES AT (0°, ±45°, 90°)

					LAMIN,	MINATE		LOAD LEVEL	EVEL	GROSS	SS	CYCLES	RESIDUAL	JAL JIC
	TEST	<u>;</u> €	MOISTURE	WIDTH	Ŧ	THICKNESS	IESS			STRESS	ESS	FAILURE	STRENGTH(2)	стн ②
SPECIMEN	°×	٠ ۳	*	CM	ż	CM	ż	NEWTONS	POUNDS	MPa	PSI	Θ	MPa	PSI
	A MAB	AMB	AMA	7 628	3 003	0.1397	0.0550	11,387	2660	106.87	15,500	130,000	ı	ı
- away				3				22,775	5120	213.73	31,000	2,600	١	I
0 0000	0 8 40	AMB	AMB	7 628	3 003	0.1442	0.0560	11,387	2560	104.96	15,222	130,000	ľ	ı
AAMB 2		2		2	}	!		11,387	2560	104.96	15,222	130,000	319.79	46,382
6 0000	AAAB	AMB	AMB	7 625	3004	0.1447	0.0570	11,121	2500	100.67	14,600	130,000	1	1
2000		})					17,793	4000	161.07	23,361	28,800	ı	١
V WW	AMB	AMB	AMB	7.620	3,000	0.1461	0.0575	11,387	2560	102.32	14,840	130,000	ł	ı
1								21,351	4800	191.85	27,826	009'8	ı	I
2 0 8 8 0 6	AMA	AMB	AMB	7 630	3 004	0.1473	0.0580	11,254	2530	100.12	14,521	130,000	ł	1
C GMAA	2	}	1	}		:		15,569	3500	138.50	20,088	130,000	296.79	43,046
2 0440 6	A A A B	AMB	AMB	7 630	3.004	0.1485	0.0585	11,254	2530	99.26	14,397	130,000	ı	1
2								15,569	3500	137.32	19,916	130,000	275.42	39,947
DAMR 1	AMA	AMB	127	7.628	3.003	0.1397	0.0550	11,030	2480	103.53	15,015	130,000	l	1
								11,030	2480	103.53	15,015	130,000	357.75	51,888
DAMB.2	AMB	AMB	1.19	7.630	3.004	0.1422	0.0560	11,121	2500	102.46	14,861	130,000	ı	1
4				!				22,241	2000	204.93	29,722	008	ı	ł
DAMB.3	AMB	AMB	1.18	7.625	3.002	0.1461	0.0575	11,121	2600	99.86	14,483	130,000	l	i
2016				}				21,352	4800	191.73	27,808	3,700	1	!
DAMB 4	AMB	AMB	1.18	7,630	3.004	0.1448	0.0570	11,121	2500	100.67	14,600	130,000	ı	1
		!						15,568	3500	140.93	20,440	130,000	263.34	38,195
DAAAD G	AMA	AMB	1 20	7 630	3 004	0.1473	0.0580	11,121	2600	98.93	14,349	130,000		
O OME			}	}				15,568	3200	138.50	20,088	130,000	254.45	36,905
O A MAD G	AMA	AMB	1 18	7 625	3.002	0.1473	0.0580	11,121	2600	99.00	14,358	130,000	ł	l
								17,793	4000	158.39	22,973	55,600	ı	1

ALL SPECIMENS SUBJECTED TO 130,000 LOAD CYCLES (ONE LIFE) AT APPROXIMATELY 2000 MICROSTRAIN (DESIGN LIMIT STRAIN) WITHOUT FAILURE PRIOR TO UNDERGOING SECOND FATIGUE LOAD TEST. ALL FATIGUE FAILURES OCCURRED AWAY FROM THE DEBOND AREA NEAR THE NECK — DOWN. Θ

ALL RESIDUAL STATIC FAILURES OCCURRED AWAY FROM THE DEBOND AREA NEAR THE NECK — DOWN. (0)

FATIGUE TEST RESULTS FOR DAMAGED LAMINATE TENSION SPECIMENS

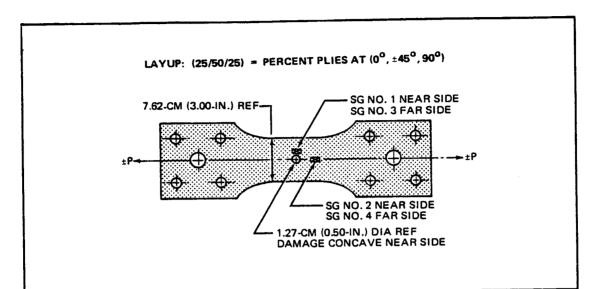
STRESS RATIO R = -1.0 Z3943442-505

LAYUP: (26/50/25) - PERCENT PLIES AT (0°, ±45°, 90°)

		!			LAMINATE	ATE		LOAD LEVEL	EVEL	NET	L NC	CYCLES	NET SECTION RESIDUAL STAT	NET SECTION RESIDUAL STATIC
	TEMP	MP .	LEVEL	WIDTH	H	THICKNESS	NESS			STRE	STRESS (1)	FAILURE	STRENGTH(2)	зтн@
SPECIMEN	o _K	96	*	₹	ž	CM	N	NEWTONS	POUNDS	MPa	PSI	0	MPa	PSI
A DBE 1	210	, es	AM8	7 087	2.790	0.145	0.067	11,565	2600	137.34	19,919	172,380	ı	i
A065.2	219	3 49	AMB	7.087	2.790	0.144	0.067	11,121	2600	133.22	19,322	260,000	185.98	26,974
A065-3	219	3 18	AMB	7.084	2.789	0.142	990.0	12,900	2900	155.99	22,624	8,630	1	1
P065-1	219	8	1.1	7.079	2.787	0.149	0.069	11,566	2600	133.99	19,433	260,000	190.67	27,655
P065-2	219	. s	1.34	7.084	2.789	0.146	0.058	12,900	2900	151.91	22,033	438,220	I	l
P065-3	219	-65	1.26	7.082	2.788	0.145	0.067	14,234	3200	169.17	24,536	6,890	i	1
AAMB-1	AMB	AMB	AMB	7.074	2.786	0.141	990'0	11,810	2655	144.33	20,934	20,600	1	, i
AAMB-2	AMB	AMB	AMB	7.082	2.788	0.144	0.057	98,786	2200	117.34	17,018	149,400	1 (۱(
AAMB-3	AMB	AMB	AMB	7.079	2.787	0.145	0.067	968′8	2000	105.79	15,343	260,000	ම	ි ම
AAMB4	AMB	AMB	AMB	7.082	2.788	0.147	0.058	10,231	2300	119.49	17,331	80,500	ļ	1
AAMB-5	AMB	AMB	AMB	7.074	2.785	0.142	0.056	9,341	2100	113.15	16,411	260,000	237.62	34,464
AAMB 6	AMB	AMB	AMB	7.082	2.788	0.143	0.056	968′8	2000	106.86	15,499	260,000	176.32	25,573
PAMB-1	AMB	AMB	1.03	7.074	2.785	0.142	0.056	9,341	2100	113.15	16,411	260,000	206.90	9000e
PAMB 2	AMB	AMB	1 05	670.7	2.787	0.144	0.057	10,231	2300	122.72	17,799	83,800	l	
PAMB-3	AMB	AMB	1.07	7.082	2.788	0.145	0.057	10,231	2300	121.60	17,636	260,000	181.86	26,377
PAMB4	AMB	AMB	1.12	7.074	2.785	0.145	0.057	11,743	2640	139.75	20,269	15,600	ı	l
PAMB-5	AMB	AMB	1.15	7.081	2.788	0.144	0.057	10,675	2400	128.01	18,566	62,400	ı	1
PAMB 6	AMB	AMB	1.13	7.079	2.787	0.145	0.057	9,341	2100	111.07	16,109	260,000	174.54	25,315
A170.1	350	170	AMB	7.076	2.786	0.145	0.057	11,566	2600	137.58	19,954	5,850	l	-
6170.2	35.0	2 2	AMB	7.079	2.787	0.147	0.058	10,231	2300	120.06	17,413	11,310	ı	ı
A1703	320	170	AMB	6/0/	2.787	0.147	0.058	8,896	2000	103.95	15,077	260,000	203.24	29,477
P170-1	320	170	66 0	7.079	2.787	0.147	0.058	11,565	2600	135.14	19,600	3,120	ı	!
P1 70-2	350	170	0.91	7.079	2.787	0.150	0.059	10,231	2300	117.52	17,045	44,760	l	1
P170-3	320	170	0.92	7.076	2.786	0.144	0.057	968'8	2000	106.77	15,485	0/0/1	ı	1
?	3	?	,											

(1) based on a damaged area having a diameter of approximately 1.27 cm (0.500 inch)
(2) all specimens failed through damaged area
(3) residual strength not obtained because of test equipment malfunction

TABLE B-4
STRAIN GAGE MEASUREMENTS FOR Z3943442 DAMAGE AND DEBOND SPECIMENS



	APPLIED GROSS AREA STRESS			EASUREI μ CM/CM	STRAIN , IN./IN.		GROSS AREA STRESS CONCENTRATION		
SPECIMEN IDENTIFICATION	^σ G MPa	σ _G PSI	SG NO. 1	SG NO.2	SG NO. 3	SG NO. 4	FACTOR K ₁ (1)		
DEBOND									
-1 AAMB-1	106:87	15,500	2050	2168	1957	2131	_		
-1 AAMB-3	100.67 100.67	14,600 -14,600	1989 -1645	2091 -2504	2014 -2410	2075 -1790			
-1 PAMB-1	103.53 -103.53	15,015 -15,015	2010 -2191	2079 2130	2020 -1951	2161 2252	-		
-1 PAMB-2	102.46 -102.46	14,861 -14,861	1950 1550	2032 2545	2092 -2550	2130 1790	_		
DAMAGE									
-505 AAMB-1	118.42	17,176	2085	3617	1925	3259	1.71		
-505 AAMB-2	96.29	13,966	1660	3030	1650	2712	1.73		
-505 PAMB-1	116.71 -116.71	16,927 -16,927	1975 1130	3427 -4744	2022 -2976	3614 -2535	1.76 1.77		
-505 PAMB-2	100.71 100.71	14,606 -14,606	1818 -2034	3011 -2520	1774 -1635	3236 4001	1.74 1.78		

⁽¹⁾ gross area stress concentration factor κ_{t_g} , shown above, is the sum of the microstrains for SG no. 2 and 4 divided by the sum of the microstrains for SG no. 1 and 3.